



Initial Design of a Ship shaped Self-Propelled Drillship

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ABSTRACT

Drillships are special purpose vessels, which are equipped with goal-oriented state of the art technology products. The main objective of this drillship design is to focus on functionality while maintaining flexibility in operations. The initial/concept design will be concentrated on two major aspects: general design and hydrodynamic performance investigation. The general design will include ship forms, preliminary freeboard calculation, preliminary general arrangement, tanks and compartmentation, lightship weight (distribution) initial estimation, intact stability calculations, still water shear force and bending moment evaluation. The general design shall not include damage stability calculations, scantling evaluation, and midship section design. The hydrodynamic performance study will include ship speed/power prediction, the study of moonpool effect on resistance and dynamic positioning initial prediction. Hydrodynamic studies shall not include seakeeping calculations.

In addition, the behavior of the flow in open moonpool has been observed by Computational Fluid Dynamic (CFD) software NUMECA/Fine Marine V5.1. Confirmation of the CFD studies was carried out on similar moonpool dimensions as used in a model test constructed by Delft University of Technology, Ship Hydrodynamics Laboratory. The approaches available in this study can be used in the preliminary design phase of a drillship.

Keywords: Drillship, Initial design, Hydrodynamic performance, Moonpool effect on resistance, Dynamic positioning

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1. INTRODUCTION

1.1. Ship design considerations in general

A ship is a complex vehicle that holds many engineering disciplines. According to reference [Mollard, 2008], ship design is not always a science, but it is a combination of theoretical analysis and numerical data with an excellent experience. For the naval architects and designers, a really well-understood design in the past can be guided with their data. This information can easily be used during the preliminary design phase to make quick estimates of principal characteristics and costs of alternative concepts for another ship. On the other hand, for the unique design, the naval architect(s) must determine priorities. For example, during the development of a high-speed craft, to estimate the required propulsion power and its cost is the most critical steps during the early design stage. This process has unchangeable essential principles such as, to understand the requirements perfectly, the time and cost constraints, co-operating work between art, science, creativity, and teamwork. [Lamb, 2003]

1.1.1. *Preliminary design criteria of service ships and offshore working vessels*

Ship design process consists of many interwoven topics, and each type of ship has its own requirements. On the other hand, a ship has to satisfy imposed production objective by owner's requirements. The following are some examples of distinctive requirements for common service ships and offshore working vessels, [Watson, 1988]

- **Tugs** – The Pollard pulls skill, fuel capacity, the turning capability, high maneuvering capability particularly harbor tugs.
- **Dredgers** – The hopper, the range of the water depth, the type of dredging gear and the quality of the spoil have to verify as far as possible compatible.
- **Icebreakers** – The thickness of the ice on the road that ship will clean, the Pollard pull skill during the cleaning period, fuel and stores capacity, high standard accommodation for a large crew.
- **Research vessels** – Good seakeeping and reliability for safety case in stormy regions, noise, and vibration for some scientific researches, attainability to scientific equipment.
- **Fishing vessels** – Storage ability according to the types of fish caught, the time that has to spend the fishing and distance.

1.1.2. Design spiral

For many years, designers and engineers used the design spiral to make clear and understandable decisions regarding the ship design process. Figure 1.1 illustrates that the ship design process is subdivided into phases based on nature of the work done, the design skills required, the number of contributors, the level of detail of the design and other features. The spiral is repeating the iterative procedure until spiral core that represents detail design for the determination of ship characteristics and other properties. [Lamb, 2003]

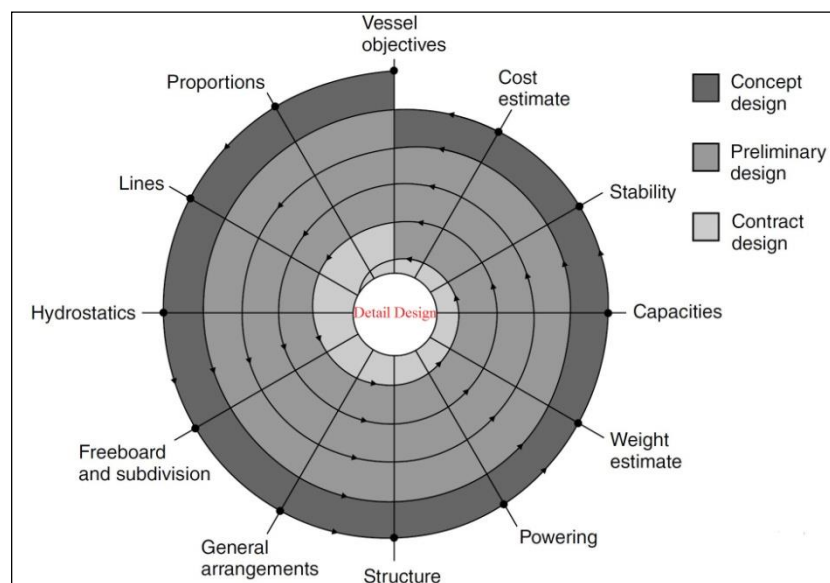


Figure 1.1 Design Spiral [Eyres & Bruce, 2012]

According to design spiral method, all process can be broken down broadly into four stages. The idea is the results are used as feedback in the next steps. Main parameters of those stages are following, [19]

- **Concept design** – Indication of the feasibility of the proposed ship as described by design requirements. Preliminary estimation of ship dimensions, layout, major items, powering & propulsion system.
- **Preliminary design** – The accurate determination of ship's main characteristics, progress in the hull form design, calculation of the stability, resistance and strength.
- **Contract design** – Technical specification document, GA plan, shaft & engine arrangement, structural plan, Interior plans, electric load analysis, bridge layout.
- **Detail design** – Each detail is investigated to initiate construction. Trim & stability calculations as per weight, noise & vibration calculations, structural details.

1.2. Offshore Drilling Vessels

Offshore vessels are the most important structures for the oil industry to explore and uncover resources located below the seabed. The increase in technological developments and the competition between companies were naturally the factors that brought the drilling industry a step further [Lamb, 2003]. These developments have increased day by day, allowing drilling vessels to work on deeper ocean bases, as shown in Figure 1.2. Also, engineers and designers working on the design of such vessels have endeavored to produce innovative solutions to meet the growing requirements. The following sections have summarized those most common types of offshore drilling vessels.

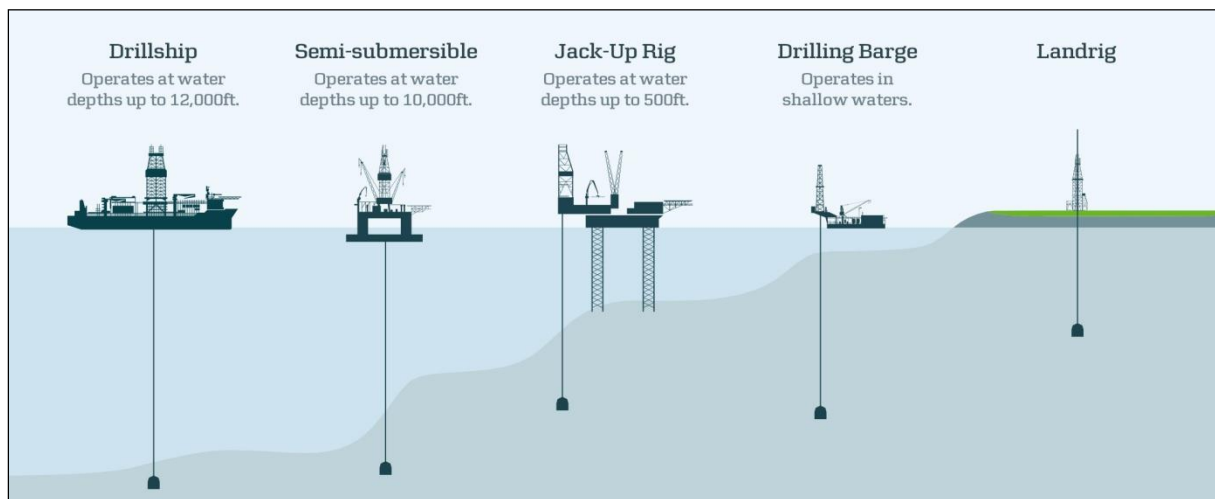


Figure 1.2 Major types of offshore drilling vessels. Available from <http://www.maersk.com/en/industries/energy>

1.2.1. Jack-ups

These mobile offshore drilling units are generally used in water depths up to 400 feet (120 m). Jack-ups are used for development drilling and well servicing operations also, these units are not designed to move from place to place [Lamb, 2003].

1.2.2. Semi-submersibles

Semi-submersibles may qualify as the group in which technological developments show the greatest impact. It is possible to encounter five different generations in literature. Especially, fifth semi-submersible generations can be used in water depths up to 10,000 feet (3,000 m) with their own dynamic positioning ability [Lamb, 2003].

1.2.3. Drillships

Drillships are special purpose vessels, which are equipped with goal-oriented state of the art technology products. Essentially, these ships designed for use in the exploratory offshore drilling of new oil and gas wells or scientific drilling purposes on the ocean beds at deep seas. Drillships are completely independent with their mobility characteristic, contrary with deepwater semi-submersibles. New generation ultra-deep-water drillships provide large variable load capacity and internal storage capacity. In addition, another attractive feature of the ships is their advanced positioning control system. Compared with earlier generation design, using their dynamic positioned system (DP), these ultra-deepwater drillships are capable of operating at drilling depths of up to 40,000 feet and at water depths of up to 12,000 feet.

The first drillships were developed by American maritime engineers in the late 1950s [1]. At the very beginning, tankers and cargo ships were converted as drillship, having equipment such as drilling derrick, moonpool, and positioning systems. In the upcoming years, the reliability of drillships was revealed by progressively developing technology. Meanwhile, increasing operational and performance needs had played a fundamental role in designing drillships. To take into consideration the ever-increasing needs of the energy sector, drillship technologies have developed rapidly. Especially, developments on the positioning control system have improved operation capability of drillships at any water depth.

As a result of this section, like every design process, the vessels are designed with their design requirements and special considerations. Certainly, drillships are not the only offshore units only capable of drilling in deep waters. Semi-submersibles, platform rigs, and jackups can also perform these operations. However, drillships are more effective for long period operations.

1.2.3.1. Drillship Concept and literature view

As mentioned before, at the end of the 1950s, drillship studies were being started. Though, studies related to full drillship' design exists in special literature; and there are many studies which relate to design purposes that will provide the most important two design criteria, such as dynamic positioning and moonpool effect on the hull resistance. As an opinion, while subjects of dynamic positioning concern about ship stability and speed power prediction. In addition, subjects related to the impact of the moonpool, which is concerning ship hull resistance and stability, can be said have an direct effect on ship design. Particularly, as per the study reference [Hammargren & Tornblom, 2012] concerning the effects of moonpools on ship resistance will be used for comparing results of resistance done in this study. Types of moonpools that are used in the reference [Veer & Tholen, 2008]. of ships are mentioned. Then, the most effective type of the moonpool, which is explained in the part of this study, is used for the designed drillship.

The design criteria differences for drillships compared to other ship and drilling unit designs can be summarized, for preliminary studies, as follows [Lamb, 2003],

- The load carrying capability
- Deck space for drilling equipment
- Propulsion system by retractable thrusters
- Motion characteristics, stability, and seakeeping
- Dynamic positioning system
- Crew carrying capability (Accommodation)

1.2.3.2. Drilling Process Steps

Step 1 – The conductor pipes are installed as the drill pipes are connected the conductor pipe and guide run down to the sea floor after the conductor pipe penetrates the seabed the drill pipe is released and pulled back to the vessel. [15].

Step 2 – To begin an easier drilling, firstly, A large drill bit connected to the bottom of the drill run down to the sea floor, the drill bit is led down to the bottom of the hole through the conductor pipe the drill bit rotates and drills the sediment and rock below the seabed. Meanwhile, seawater is sprayed from nozzles on the drill bit to raise the cuttings to the sea floor. After drilling several hundred meters, the drill bit is pulled back to the vessel. [15].

Step 3 – A casing pipe about 50 centimeters in diameter is set into the drilled hole to keep it from collapsing. The casing pipe is run down through the conductor pipe and is inserted into the hole using the drill pipe. [15].

Step 4 – Cement is pumped into the space between the hole and the casing pipe to fix the pipe in place. The drill pipe is withdrawn to the ship, after cementing operation,. The drillship is equipped with a riser system in order to drill into the earth even deeper. As the riser pipes are added one after the other, the Blow out Preventer (BOP) is run down to the sea floor. The Blow out Preventer is connected to a well head which is located on top of the casing pipe. After this stage, the marine vessel is connected to the seabed with the riser pipe. [15].

Step 5 – A drill bit, smaller than the one first used is run down the through riser pipe and casing pipe. The drilling begins. Once the riser pipe has been connected, drilling mud is used instead of sea water. When the target depth is reached the drill bit is pulled back to the vessel. To drill the hole even deeper, a narrower casing pipe is sent to set in the drilled hole. After the casing pipe has been installed, cement is pumped into the space between the hole and the casing pipe to fix the pipe in place. Again, an even smaller drill bit is run down through the riser pipe and casing pipe and the drilling continues. The drill ship will drill the ocean floor by repeating this operation. [15].

1.3. Aims and Objectives

The academic aims of the study are to contribute to existing initial ship design steps for the drillship that present research with the following objectives,

- To design a modern drillship vessel with some required specifications like marine operations at deep waters all around the world.
- To investigate the existing literature about drillship design considerations through a database search.
- Generation of ship forms and hydrostatic particulars.
- Preliminary Freeboard Calculation.
- Preliminary design of General Arrangement and Tank Plan.
- To clearly define and investigate the importance of dynamic position (DP) that has spotlighted in the offshore drilling industry.
- To reveal the hydrodynamic performance of the drillship hull by using the computational fluid dynamic software and investigate the moonpool effects on resistance.
- To compare the results from the numerical method and computational fluid dynamics software for moonpool effect on the resistance of the ship hull.
- Generate a set of minimal loading conditions as required by chosen Classification Society (ABS).
- To investigate ship Intact Stability compliance (by IS CODE 2008 standards) and Longitudinal Strength.

1.4. Methods

1.4.1. Software

Aveva Marine Initial Design Module was used for all static calculations and check for compliance rules and regulations. For hydrodynamic calculation, NUMECA/Fine Marine and ship flow were used. Where deemed necessary, hand calculations were made to supplement or replace software computations.

1.4.2. Rules and Regulations

The concept design was based on following rules, standards, and regulations,

- ABS Rules for Building and Classing Mobile Offshore Drilling Units, 2012.
- IMO SOLAS – International Convention for the Safety of Life at Sea
- IMO MARPOL – International Convention for the Prevention of Pollution from Ships
- IMO ICLL – International Convention on Load Lines,
- IMO COLREG – International Convention for the Prevention of Collisions at Sea
- IMO MSC/Circ.645 Guidelines for Vessels with Dynamic Positioning Systems

1.4.3. Calculations summary

At the beginning of the thesis, AVEVA Lines Module is used with the aim of forming hull body plan. By using available ships' main dimensions, a database (available in Appendix 1) is created to perform the regression analysis. Then, AVEVA's modules which are needed to design of drillship are used such as AVEVA Surface and Compartment Module and AVEVA Hydrostatics and Hydrodynamic Module. Also, During the hull design; it is taken advantage of numeric approaches for main dimensions of the drillship. The relationship between main dimensions was checked by the references (Watson, 1988), (K. Von Dokkum, 2003) and (Barras, 2004). Essentially, classification rules and regulations are followed. During the hull resistance estimation and moonpool effect on the resistance of drillship, the computational fluid dynamic software NUMECA/Fine Marine was used. In addition, Holtrop-Mennen method was performed for resistance and powering estimation with Microsoft Excel.

2. LINES PLAN

2.1. Regression analysis

In order to define the dimensions of the ship, a database was generated from existing drillship designs. This method helped to determine principle dimensions and performance characteristics based on statistical data of the drillship database. For this study, the existing ships listed in appendices part will be referred as similar drillships. There are important parameters to choose from the analysis of the similar ships, such as the length overall (LOA), the length of waterline (LWL), Displacement (Δ). In addition, operation parameters were considered in the data analysis, such as drilling depth, water depth, and moonpool dimensions. A statistical measure was needed to filter the information from the database to find the strength between dependent variables and changing variables. The following Figure 2.1 shows that a linear regression was used for principal dimensions. The linear regression results in a line equation, which minimizes the distance between the fitted line and all of the data points using the least squares method. It means that an R^2 coefficient of determination is found that defines the precision of the regression.

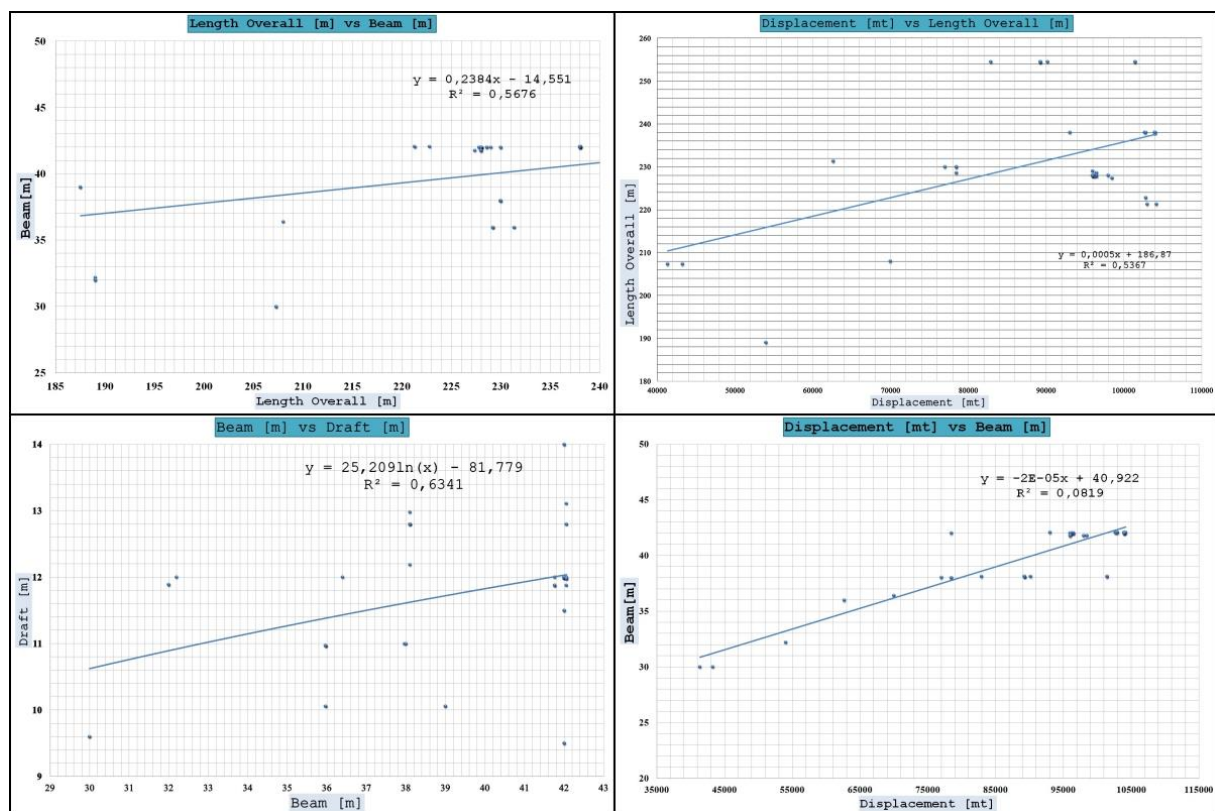


Figure 2.1 Regression Analysis by created database

From the linear regression, the main dimensions were estimated for the initial design of the drillship. Also, dimensional relationships have been used based on the reference (Watson, 1988) which has described six-dimensional relationships linking the four main ship dimensions of Length (L), Beam (B), Depth (D) and Draft (T). According to reference (Watson, 1988), it is possible to solve that the weight or volume equations with these three main dimensions. The reference (Von Dokkum, 2003) defines that B/D ratio effect on stability and the strength. On the other hand, L/D ratio plays a role in the determination of freeboard and longitudinal strength. Also, the relationships are following:

$$B = f(L) \quad , \quad D = f(L) \quad , \quad T = f(L)$$

$$D = f(B) \quad , \quad T = f(D) \quad , \quad T = f(B)$$

After all this, the values obtained in **Table 2.1** were not the final results obtained for the designed ship. They were used just as an initial step to have the main dimensions of the drillship for the subsequent analyses.

Table 2.1 Estimated Main Dimensions of the Drillship

Parameters	Values
Length overall (LOA)	231 m
Beam (B)	36 m
Depth (D)	19 m
Draft (T)	12 m

2.2. Surface Generation

For the design of the hull form, the 3D hull modeling software AVEVA Initial Design Module v.12.1.SP4 was used. Although it was possible to start the design after obtaining the main dimensions of the drillship without an initial hull form, however, first body plan generated in AVEVA LINES Module based on existing drillship lines plan. The main properties of existing drillship are presented in **Table 2.2**.

Table 2.2 Main Properties of the Original Hull Form

Parameters	Values
Length overall (LOA)	210,00 [m]
Length Between Perpendicular (LPP)	198,01 [m]
Length Waterline (LWL)	200,38 [m]
Beam (B)	34,00 [m]
Draft (T), summer	10,81 [m]
Depth (D), at Center Line	17,50 [m]
Depth (D), at Side	17,35 [m]
Displacement , at T=10,81 m	60566,80 [t]
Longitudinal Center of Buoyancy (LCB)	100,65 [m]
Vertical Center of Buoyancy (VCB)	5,64 [m]
C _b	0,8094
C _m	0,6260
C _w	0,8893
C _p	1,2937

Many modifications were made to the initial hull form after hull generation from its lines. Then, existing hull form scaled to match the characteristics determined at regression analysis. Some modifications were induced by design preferences, such as the moonpool dimensions, main deck transition, and dimension of the forecastle. For sure, after every modification, the hull lines had to fair in detail so that the hull would provide smooth sections, waterlines, and buttocks. Respectively, frame and longitudinal spacing defined as 700 mm for both transverse and longitudinal.

Firstly, according to ICLL Regulation 43(1), Regulation 33 and Regulation 3(g), forecastle height (3300 mm) and length ($\leq 0,07 \cdot L$) determined for the drillship. In addition, L is defined in the ICLL Regulation 3(1). Secondly, the gunwale (150 mm) and bulwark height also determined based on ICLL Regulation 43(3). Thirdly, sheer (taken= 0 mm) and camber height ($\approx B/50$, taken 700 mm) determined based on ICLL Regulation 38. Also, B is defined in ICLL Regulation 3(4). Then, deck at side curve generated based on camber and sheer. Lastly, for obtaining the exact main parameters of the hull, the model was exported to another module of AVEVA, which is Surface and Compartment Module. On this module respectively, the main deck generated including forecastle and transition.

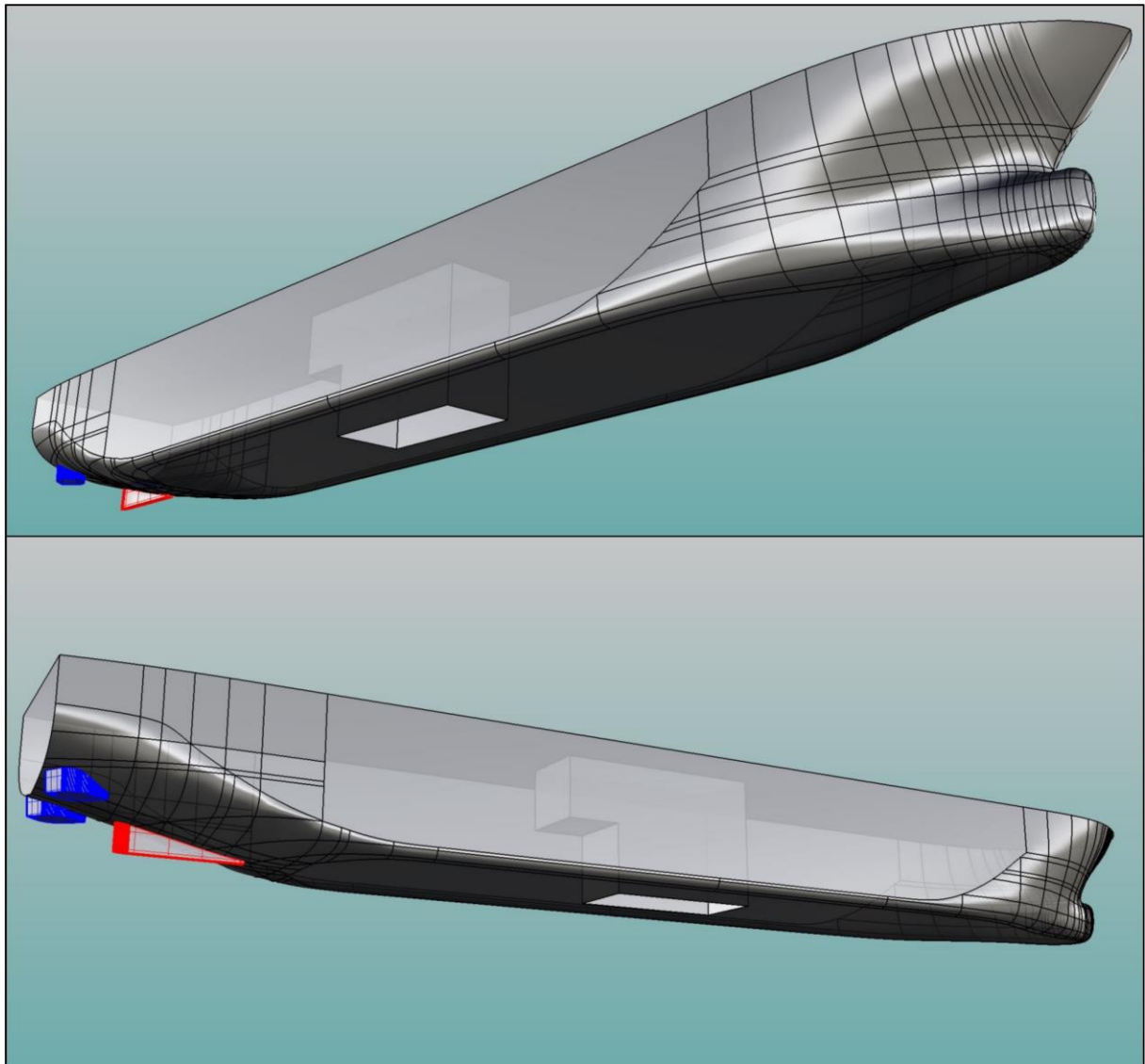


Figure 2.2 Drillship surface with appendices, Skeg,Thruster supports and Moonpool

2.3. Moonpool Definition

In simplest explanation, moonpool is an opening designed to facilitate drilling operations. The moonpool, which is one of the most important “parts” of the drillship, was generated based on drillship database, the resistance of moonpool studies [Veer & Tholen, 2008], [Hammargren & Tornblom, 2012] and frame spacing. To estimate the dimensions and characteristics of the moonpool, methods from reference [Hammargren & Tornblom, 2012] were used. In this reference, the result of the most effective moonpool type are available. For this reason, moonpool with aft recess was used on the designed ship. The dimensions of the moonpool are shown in **Figure 2.3** and its properties are shown in the following **Table 2.3**, respectively.

Table 2.3 Moonpool Dimensions

Parameters	Values
Length of Moonpool min (Lmin)	28 [m]
Length of Moonpool max (Lmax)	37,8 [m]
Breadth of Moonpool	12,6 [m]
Length of Recess (Lrecess)	9,8 [m]
Height of Recess	12,5 [m]

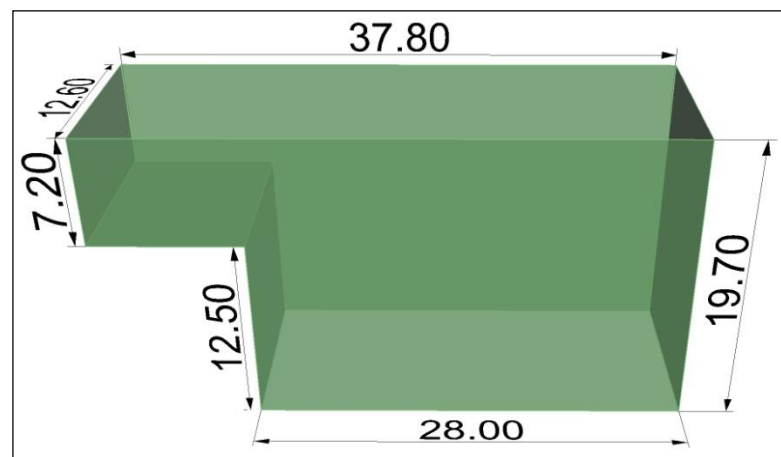


Figure 2.3 Moonpool Dimensions

After moonpool definition, other important appendices are included on the ship hull, such as a preliminary established a center skeg, preliminarily positioned sea chests, preliminarily generated thruster geometry and thruster supports. Finally, preliminary Lines plan was completed after adding to the model all appendices. The final lines plan of designed ship model is shown as Appendix 2 in the study, and its properties are shown in **Table 2.4**.

2.4. Axes Convention

The x-axis is the longitudinal axis directed from aft to fore, and positive y-axis is directed to the port side of the vessel while the positive z-axis is the normal axis to the horizontal plane pointing upwards.

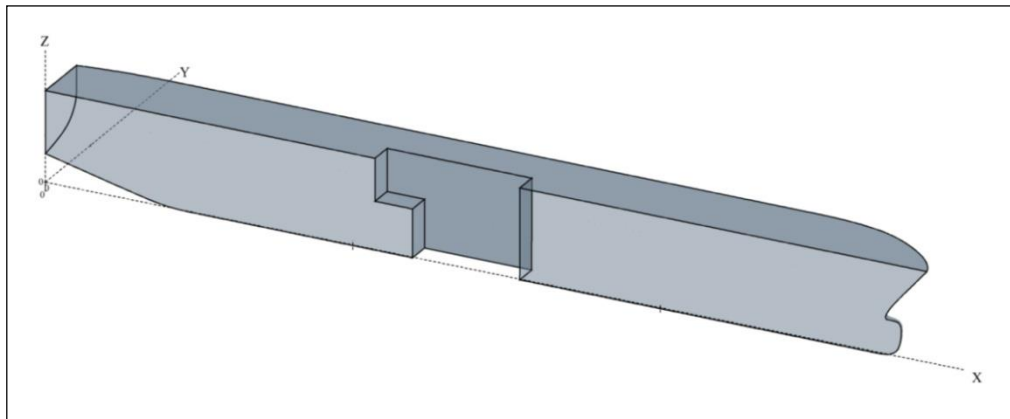


Figure 2.4 Drillship axis convention for all calculation (Half body view)

Table 2.4 Main Properties of the Final Hull Form

Parameters	Values	
Length overall (LOA)	230,89	[m]
Length Between Perpendicular (LPP)	220,61	[m]
Length Waterline (LWL)	220,61	[m]
Beam (B)	36,00	[m]
Draft (T), summer	12,89	[m]
Depth (D), at Center Line	19,70	[m]
Depth (D), at Side	19,00	[m]
Displacement , at T=12 m	79990,05	[t]
Longitudinal Center of Buoyancy (LCB)	110,711	[m]
Vertical Center of Buoyancy (VCB)	6,27	[m]
Cb	0,819	
Cm	0,996	
Cw	0,898	
Cp	1,267	

The surface of the drillship has exported to another module of AVEVA to continue with hydrostatics and hydrodynamics calculations.

3. HYDROSTATIC PARTICULARS

The scope of this part of the study is to define the hydrostatic particulars of the hull shape of the drillship. AVEVA Hydrostatics and Hydrodynamics module was used to perform the hydrostatic analysis and obtain the hydrostatic curves, which are shown below **Figure 3.1**. Using the generated surface, this module was able to provide all of the hydrostatic quantities such as buoyancy center (LCB & VCB), displacement and sectional areas of the drillship at multiple drafts. Also, the specific gravity used in this part of the report is 1,025 t/m³.

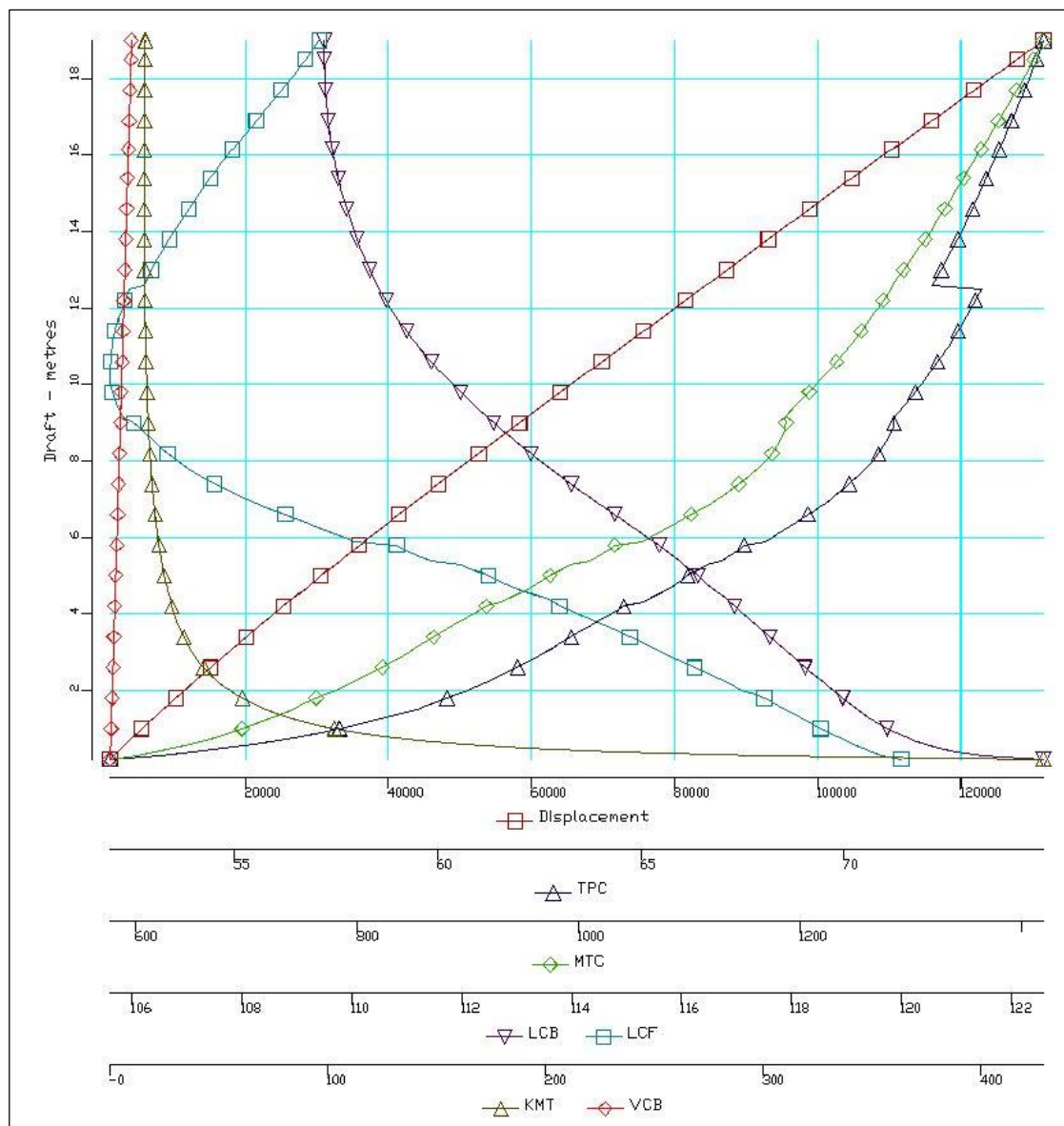


Figure 3.1 Hydrostatic curves of the Drillship

To estimate the hydrostatics and hydrodynamic parameters are almost always most significant step during the early stages of the design. Naval architect must estimate correctly main dimensions of ship and design coefficient such as C_b , C_m , C_p and C_w which are identified on the abbreviation part of the thesis. The erroneous estimations of hydrostatics can cause an unbalance for all design steps. For example; According to reference (Barras,2004) [Barras, 2004], Even to making a little change in the breadth of the ship may solve a problem of instability. The main particulars of the vessel are presented below in **Table 3.1**. In addition, another hydrostatics parameters for other drafts are included as Appendix 3. It's necessary to say that there is concurrence about C_m and C_p values that are rather small and the other is above 1. It is due to the moonpool presence.

Table 3.1 Hydrostatic Particulars of the Drillship

Parameters	Values				
Draft [m]	8,00	10,00	12,00	14,00	16,15
Displacement [t]	51410,22	65698,0	80235,04	94810,88	110612
LCB [m]	113,438	111,833	110,711	110,039	109,645
VCB [m]	6,89	7,01	7,13	7,11	7,20
WPA [m ²]	4,167	5,22	6,27	7,305	8,417
LCF [m]	106,826	105,599	105,821	106,771	107,826
KML [m]	507,732	414,647	356,563	314,777	280,071
KMT [m]	18,463	16,802	16,013	15,715	15,737
WSA [m ²]	11717,7	12907,65	14059,7	15368,95	16662,98
TPC [t/cm]	70,71	71,91	73,12	72,9	73,82
MTC [t-m/cm]	1168,84	1215,11	1270,12	1317,69	1363,48
C_b	0,786	0,804	0,819	0,83	0,839
C_m	0,644	0,645	0,646	0,647	0,647
C_p	1,221	1,246	1,267	1,283	1,297
C_w	0,869	0,883	0,898	0,895	0,907

The hydrostatic model of the drillship is shown in the following **Figure 3.2**. Moonpool and sea water inlet chests are excluded from the hull volume. Skeg and the buoyant parts of the thrusters are included on the hull form.

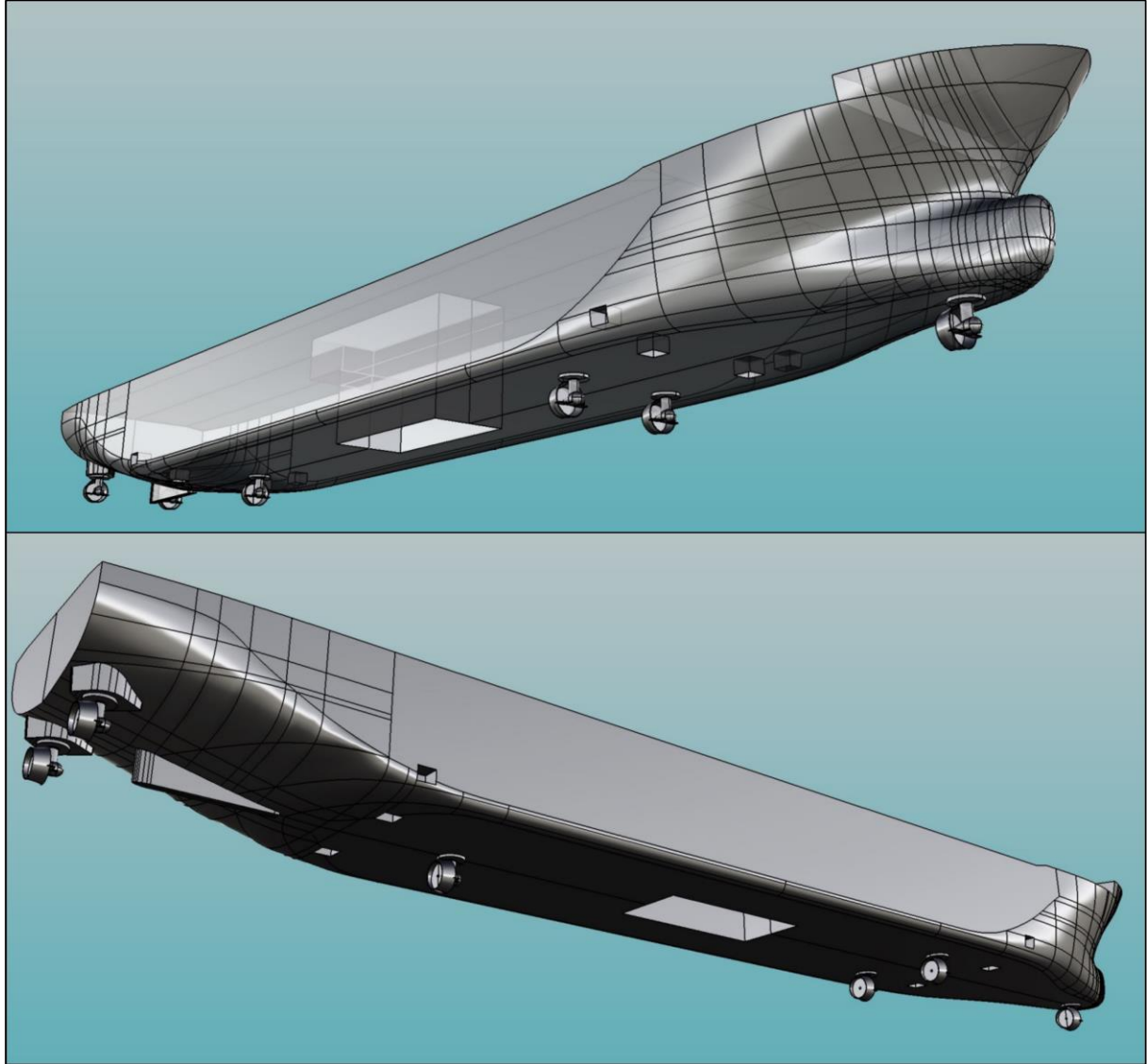


Figure 3.2 Designed Drillship bare Hull for Hydrostatic Calculations

4. FREEBOARD CALCULATIONS

The purpose of this part of the study is to obtain the results of the freeboard calculation. All ships are required to be surveyed and marked with permanent load line markings as per international conventions. By definition, the freeboard assigned to a ship is the distance measured vertically downwards amidships from the upper edge of the deck line at side to the upper edge of the related load line mark. This freeboard calculation was performed according to the provisions of the “International Convention on Load Lines, 1966 including Protocol of 1988, also Resolution MSC143 (77)”. The calculation was performed for Type of ship “B” drillship with following parameters, which are shown, on **Table 4.1**.

Table 4.1 Principal Dimensions for Freeboard Calculation

Parameters	Values
Length overall (LOA)	230,89 [m]
Length Between Perpendiculars (LBP)	220,61 [m]
Breadth (B)	36,00 [m]
Design Draught (T)	12,00 [m]
Scantling Draught	12,40 [m]
Depth (D), at Center Line	19,00 [m]
Depth (D), at Side	19,70 [m]
Stern Overhang	0 [m]
Stem Overhang	10,28 [m]
Bilge Radius	1,80 [m]
Main Deck Thickness	17,00 [mm]

4.1. Description of Terms

Various regulations have described the terms, which are used for freeboard calculation, respectively.

Length: Regulation 3(1)

$$d_1 = 0,85 * D = 0,85 * 19,0 = 16,15 \text{ m}$$

$$L_{0,85D} = 223,96 \text{ m} \rightarrow 0,96 * 223,96 = 214,999 \text{ m} \approx 215 \text{ m (taken)}$$

$$L_{Freeboard} = 215 \text{ m}$$

Where;

$L_{Freeboard}$ = Freeboard Length [m]

D = Moulded Depth [m]

d_1 = Draft at 85% of the side depth D [m]

Perpendiculars: Regulation 3(2)

The forward and after perpendiculars shall be taken at the forward and after ends of the length.

Amidships & Breadth : Regulation 3(4)

Amidships is at the middle of the length, L.

$$\frac{L_{Freeboard}}{2} = 107,5 \text{ m}$$

Moulded Depth: Regulation 3(5)

$$D = 19 \text{ m}$$

Depth for Freeboard: Regulation 3(6)(a)

$$D_F = D + t \rightarrow 19 + 0,017 = 19,017 \text{ m}$$

Where;

t = Thickness of freeboard deck [m]

Block Coefficient: Regulation 3(7)

$$C_B = 0,819$$

Freeboard Deck: Regulation 3(9)

The freeboard deck is the main deck.

4.2. Tabular Freeboard & Freeboard Corrections

The tabular freeboard for type 'B' ships shall be determined from 'Table B – Regulation 28(2)' in International Convention on Load Lines.

$$L_{Freeboard} = 215 \text{ m} \rightarrow F_{BO} = 3505 \text{ mm}$$

Where;

F_{BO} = Tabular freeboard from the regulation's table

➤ *Correction for Block Coefficient (C_b)*

It is necessary to correction for block coefficient ($C_B > 0,68$) so, multiply factor for C_b is determined by following equations that are described in regulation 30;

$$\begin{aligned} C_B &= 0,819 > 0,68 \\ f_{CB} &= \frac{C_B + 0,68}{1,36} = \frac{0,819 + 0,68}{1,36} = 1,1 \\ F_{BO} * f_{CB} &= 3505 \text{ mm} * 1,1 \rightarrow 3863,23 \text{ mm} \\ f_1 &= 3863.23 \text{ mm} - 3505 \text{ mm} = 358,23 \text{ mm} \end{aligned}$$

Where;

f_{CB} = Multiply factor for C_B

f_1 = First correction value

➤ *Correction for depth*

It is necessary correction for depth ($D_f > L_{freeboard}/15$) so, that is determined by following equations which are described in regulation 31;

$$\begin{aligned} \frac{L_{Freeboard}}{15} &= \frac{215 \text{ m}}{15} = 14,333 \text{ m} \\ D_F &= 19,17 \text{ m} > 14,333 \text{ m} \\ \text{for ship length above } 120 \text{ m} &\rightarrow f_2 = \left(D_F - \frac{L_{Freeboard}}{15} \right) * R \\ f_2 &= \left(19,17 \text{ m} - \frac{215 \text{ m}}{15} \right) * 250 = 1170,917 \text{ mm} \end{aligned}$$

Where;

f_2 = Second correction value

$R = 250$ (for length above 120m)

➤ *Superstructure Particulars and Standard height*

According to regulation 33, standard height of superstructure for $L_{\text{Freeboard}} = 215 \text{ m}$ is 2,30 m. In addition, Table 4.2 describes the length of the superstructure and its particulars based as defined in regulation 34.

Table 4.2 Superstructures Particulars for freeboard correction

Superstructure	Length [m]	Breadth [m]	Set-in [%]	Height [m]	Standard Height [m]	Effective Length [m]
Forecastle	39,158	24,00	0,00	3,30	2,30	39,158

➤ *Correction for Superstructure*

It is a necessary correction for the superstructure. According to length ratio, the deduction percentage is obtained by linear interpolation as shown in Table 4.3 (See table 37.1 in regulation 37 to determine the percentage of deduction for all types of superstructures). The deduction percentage is determined by following equations that are described in regulation 37;

$$\text{Length ratio} \rightarrow \frac{L_e}{L_{\text{Freeboard}}} = \frac{39,158 \text{ m}}{215 \text{ m}} = 0,1821$$

Table 4.3 Interpolation for Effective Length

	Deduction Percentage [%]
0,1 L	7,00
0,1821 L	12,749
0,2 L	14,00

The standard deduction is 1070 mm for superstructure correction for ship length above 122 m.

$$f_3 = -1070 * \frac{12,749}{100} = -136,415 \text{ mm}$$

Where;

f_3 = Third correction value

L_e = Effective Forecastle Length, (see Table 7)

➤ *Correction for Sheer*

The drillship has no actual sheer. However, it is a necessary correction for sheer that is determined by following equations that are described in regulation 38. Table 4.4 and Table 4.5 illustrate that aft half and forward part of sheer correction for the drillship, respectively. (See Regulation 38(8)).

Table 4.4 Sheer correction for Aft half

		Standard			Actual		
	Station	Ordinate [mm]	Factor	Product	Ordinate [mm]	Factor	Product
After half	AP	2041,67	1	2041,67	0	1	0
	1/6L from AP	906,50	3	2719,50	0	3	0
	1/3L from AP	228,67	3	686,00	0	3	0
	Amidships	0,00	1	0,00	0	1	0
	SUM	AS=		5447,17	AA=		0

$$\text{Deficiency of aft sheer} \rightarrow \frac{AS - AA}{8} = \frac{5447,17 \text{ mm}}{8} = 680,896 \text{ mm}$$

Table 4.5 Sheer correction for Forward half

		Standard			Actual		
	Station	Ordinate [mm]	Factor	Product	Ordinate [mm]	Factor	Product
Forward half	AP	0,0	1	0,0	0	1	0
	1/6L from FP	457,3	3	1372,0	0	3	0
	1/3L from FP	1813,0	3	5439,0	0	3	0
	Amidships	4083,3	1	4083,3	0	1	0
	SUM	FS=		10894,3	FA=		0

$$\text{Deficiency of forward sheer} \rightarrow \frac{FS - FA}{8} = \frac{10894,3 \text{ mm}}{8} = 1361,791 \text{ mm}$$

$$\begin{aligned} \text{Total Deficiency of sheer} &\rightarrow \frac{(\text{Aft sheer} + \text{Forward sheer})}{2} \\ &\rightarrow \frac{680,896 \text{ mm} + 1361,791 \text{ mm}}{2} = 1021,35 \text{ mm} \end{aligned}$$

According to regulation 38(11), sheer credit for forecastle should determine by ($FA < 0,5 \cdot FS$). For this case, no credit shall be given for excess sheer forward. It means that sheer credit is zero.

$$f_4 = (Total\ Deficiency\ of\ sheer + Sheer\ Credit) * (0,75 - \left(\frac{S}{2 * L_{Freeboard}}\right))$$

$$f_4 = (1021,35\ mm + 0) * (0,75 - \left(\frac{39,158\ m}{2 * 215\ m}\right))$$

$$f_4 = 672,999\ mm$$

Where;

f_4 = Forth correction value

S = Forecastle Length, (See Table 4.2)

➤ **Correction for Recess in freeboard deck**

According to regulation 32(1), the correction applied an addition to the freeboard obtained after all other corrections have been applied. Total freeboard corrections until recess correction is 5617,12 mm. The correction factor is determined by following equations;

$$R_r = \frac{Recess\ Volume}{Water\ plane\ area\ at\ 0,85D} = \frac{7839,216\ m^3}{7206,508\ m^2} = 1,0878$$

$$f_5 = (1,0878 * 5617,12\ mm) - 5617,12\ mm$$

$$f_5 = 493,17\ mm$$

Where;

R_r = Recess correction factor

f_5 = Fifth correction value

4.3. Freeboard Calculation

➤ **Minimum Summer Freeboard (F)**

$$F = F_{BO} + f_1 + f_2 + f_3 + f_4 + f_5$$

$$F = 3505 + 358.23 + 1170.917 + (-136.415) + 672.999 + 493.17$$

$$F = 6063.9 \text{ mm}$$

For the proposed draught $d=12.40\text{m}$ (scantling draught), the summer freeboard adopted is determined.

$$F = D_f - d = 19.017 \text{ m} - 12.40 \text{ m} = 6.617,0 \text{ m}$$

Determined F , is greater than calculated minimum F value. According to regulation 40 for this summer draught the following freeboards calculated;

➤ **Tropical Freeboard (F_T)**

$$F_T = F - \frac{d}{48} = 6.617 \text{ m} - \left(\frac{12.4 \text{ m}}{48}\right)$$

$$F_T = 6.359 \text{ m}$$

➤ **Winter Freeboard (F_W)**

$$F_W = F + \frac{d}{48} = 6.617 \text{ m} + \left(\frac{12.4 \text{ m}}{48}\right)$$

$$F_W = 6.875 \text{ m}$$

➤ **Winter North Atlantic Freeboard (F_{WNA})**

According to regulation 40(6), for the drillship, Winter North Atlantic Freeboard shall be Winter Freeboard.

$$F_{WNA} = F + \frac{d}{48} = 6.617 \text{ m} + \left(\frac{12.4 \text{ m}}{48}\right)$$

$$F_{WNA} = 6.875 \text{ m}$$

➤ **Fresh Water Freeboard (F_{FW})**

$$f_{FW} = \frac{\Delta}{40 * T} = \frac{80235,04 \text{ t}}{40 * 72,38 \text{ t/cm}}$$

$$f_{FW} = 0,277 \text{ m}$$

$$F_{FW} = F - f_{FW} = 6,617 \text{ m} - 0,277 \text{ m}$$

$$F_{FW} = 6,34 \text{ m}$$

Where;

T = tonnes per centimeter immersion in salt water at summer load waterline, T= 72,38 t/cm.

Δ = Displacement in salt water in tons at the summer load waterline, Δ = 80235,04 t

➤ **Tropical Fresh Water Freeboard (F_{TFW})**

$$F_{TFW} = F_{FW} - \frac{d}{48} = 6,34 \text{ m} - \left(\frac{12,4 \text{ m}}{48} \right)$$

$$F_{TFW} = 6,088 \text{ m}$$

The tropical, winter, Winter North Atlantic and Freshwater freeboards are then calculated as illustrated in Table 4.6.

Table 4.6 Freeboard Values with applied all Corrections

Parameters	Values
Tabular Freeboard Fbo	3,505 [m]
Block Coefficient Correction (f1)	0,3582 [m]
Depth Correction (f2)	1,1709 [m]
Superstructure Correction (f3)	-0,1364 [m]
Sheer Correction (f4)	0,6730 [m]
Recess Correction (f5)	0,4932 [m]
Summer Freeboard	6,617 [m]
Maximum Draught at Summer Draught	12.40 [m]
Proposed Summer Draught	12,40 [m]
Assigned Summer Freeboard	6,617 [m]
Tropical Freeboard	6,359 [m]
Winter Freeboard	6,875 [m]
Winter North Atlantic Freeboard	6,875 [m]
Fresh Water Freeboard	6,34 [m]
Tropical Fresh Water Freeboard	6,088 [m]

➤ **Minimum bow height**

According to Regulation 39(1) required bow height calculated by the following equation,

$$F_b = \left(6075 * \left(\frac{L}{100} \right) - 1875 * \left(\frac{L}{100} \right)^2 + 200 * \left(\frac{L}{100} \right)^3 \right) * (2,08 + 0,609 * C_B - 1,603 * C_{wf} - 0,0129 * \left(\frac{L}{d_1} \right))$$

$$C_{wf} = \frac{A_{wf}}{(L/2) * B} = \frac{3530,86 \text{ m}^2}{(215 \text{ m}/2) * 36 \text{ m}} = 0,912$$

$$F_b = \left(6075 * \left(\frac{215}{100} \right) - 1875 * \left(\frac{215}{100} \right)^2 + 200 * \left(\frac{215}{100} \right)^3 \right) * (2,08 + 0,609 * 0,837 - 1,603 * 0,912 - 0,0129 * \left(\frac{215}{16,15} \right))$$

$$F_b = 6293,236 \text{ mm}$$

Where,

F_B = Calculated minimum bow height [mm]

L = Freeboard Length

B = Moulded Breadth

d_1 = Draught at 85%D, $d_1 = 0,85 * 19 = 16,15 \text{ m}$

C_B = Block Coefficient

C_{wf} = Water plane Coefficient forward of $L/2$

A_{wf} = Water plane area forward of $L/2$, $3530,86 \text{ m}^2$

➤ **Actual bow height**

The actual bow height as defined by the rule, regulation 39(2).

$$H_a = D + \text{Sheer at FP} + \text{Height of Forecastle at FP} - \text{Proposed Summer Draught}$$

$$H_a = 19000 \text{ mm} + 0 + 3300 \text{ mm} - 12400 \text{ mm}$$

$$H_a = 9900 \text{ mm}$$

$$H_a > F_b = 6293,236 \text{ mm} \text{ (Rule compliance satisfied)}$$

5. SPEED AND POWER PREDICTION

In order to calculate the resistance of the drillship, two different approaches were used. Firstly, Holtrop-Mennen method [Holtrop & Mennen, 1982] was performed as a statistical regression method at the initial design of the ship, which focused on improving the power prediction of the high block, ships with low L/B ratios with main appendages and immersed transom sterns. Thus, the moonpool resistance of the vessel was also taken into account as an extra component of the total resistance based on the reference [Veer & Tholen, 2008]. Total ship resistance was estimated for design speed, which is 12 knots in following section 5.1.

Secondly, computational fluid dynamics (CFD) method was performed for both, drillship bare hull with moonpool and bare hull without moonpool. End of this part of the study, to have resistance comparison between bare hull with moonpool and without moonpool will be possible. It should be noted that, this is the same as the displacement CFD study used in the Holtrop-Mennen method to make this comparison more realistic.

5.1. Power prediction by Holtrop-Mennen Method 1982

In publication [Holtrop & Mennen, 1982], the total resistance of a ship has been subdivided as an equation is following.

$$R_{Total} = R_F * (1 + k_1) + R_{APP} + R_W + R_B + R_{TR} + R_A$$

Where:

R_F = Frictional Resistance according to ITTC 1957.

$(1+k_1)$ = Form factor describing the viscous resistance of the hull form about R_F .

R_{APP} = Resistance of appendages.

R_W = Wave-making and wave-breaking resistance.

R_B = Additional pressure resistance of bulbous bow near the water surface.

R_{TR} = Additional pressure resistance of immersed transom stern.

R_A = Allowance resistance.

➤ **Step 1 – Determination of R_F**

$$R_e = \frac{V * L_{WL}}{\nu} \rightarrow \frac{(12 * 0,51444) * 220,61}{1,19 * 10^{-6}} = 1144452008$$

$$C_f = \frac{0,075}{(\log_{10}(R_e - 2))^2} = 0,001505$$

$$S_{Hull} = 11982,12 \text{ m}^2 \quad \text{Determined by Aveva}$$

$$R_F = C_f * 0,5 * \rho_w * V^2 * S_{Hull}$$

$$R_F = 0,001505 * 0,5 * 1,025 * (12 * 0,51444)^2 * 11982,12 = 352,22 \text{ [kN]}$$

Where,

R_e = Reynold Number

C_f = The coefficient of frictional resistance, ITTC-1957

S_{Hull} = The wetted area of the bare hull [m^2]

ν = Kinematic viscosity of water [m^2/s]

V = Ship velocity [knots]

L_{WL} = Waterline length [m]

ρ_w = the water density [t/m^3]

➤ **Step 2 – Determination of $(1+k_1)$**

This step calculated by Holtrop 1984 method [Holtrop,1984] to have more accuracy for form factor, $(1+k_1)$.

$$c_{12} = \left(\frac{T}{L_{WL}} \right)^{0,2228446}, \text{ for } T/L_{WL} > 0,05$$

$$c_{12} = \left(\frac{12 \text{ m}}{220,61 \text{ m}} \right)^{0,2228446} = 0,5227$$

$$c_{14} = 1 + 0,011 * c_{stern} \quad \text{when, aft part U shape section } c_{stern} = 7$$

$$c_{14} = 1 + (0,011 * 7) = 1,077$$

$$L_{CB} = -0,42 \% L \text{ from } 1/2 L$$

$$C_P = 0,844$$

$$\frac{L_R}{L_{WL}} = 1 - C_P + (0,06 * C_P * L_{CB}) / (4 * C_P - 1)$$

$$L_R = 37,75 \text{ m}$$

$$(1 + k_1) = 0,93 + c_{14} * 0,487118 * \left(\frac{B}{L_{WL}}\right)^{1,06806} * \left(\frac{T}{L_{WL}}\right)^{0,46106} * \left(\frac{L_{WL}}{L_R}\right)^{0,121563} \\ * \left(\frac{L_{WL}^3}{\nabla}\right)^{0,121563} * (1 - C_p)^{-0,604247} \\ (1 + k_1) = 1,3796$$

Where,

T = Moulded draught [m]

L_R = Reflecting the length of the run

B = Breath of the ship [m]

C_p = Prismatic coefficient

As mentioned before in Hydrostatic calculations, C_p of the drillship is more than one, because of moonpool on the middle of the ship. However, the Holtrop-Mennen 1982 [Holtrop & Mennen, 1982] method is not able to calculate ship resistance with condition ($1 \leq C_p$). That's why, during the resistance calculations, the prismatic coefficient is found based on the numerical approximation. This approximation is shown as following,

$$C_p = \frac{\Delta}{(A_M * L_{WL})} = \frac{80136,2927 \text{ m}^3}{(430,364 \text{ m}^2 * 220,61 \text{ m})} = 0,844$$

Where,

A_M = Midship section area

Δ = Displacement Volume Moulded [m³] (was taken same as CFD value)

➤ Step 3 – Determination of R_{APP}

$$S_{APP} = 272,533 \text{ m}^2 \quad \text{Calculated by Aveva}$$

$$(1 + k_2)_{eq} = \frac{\Sigma(1 + k_2) * S_{APP}}{\Sigma S_{APP}}$$

$$(1 + k_2) = 2, \quad \text{Approximation with skeg}$$

$$R_{APP} = 0,5 * \rho_w * V^2 * S_{APP} * (1 + k_2)_{eq} * C_f$$

$$R_{APP} = 0,5 * 1,025 * (12 * 0,51444)^2 * 272,533 * 2 * 0,001505$$

$$R_{APP} = 16,025 \text{ [kN]}$$

Where,

(1+k₂) = Appendages Resistance factor

S_{APP} = Total wetted area of the appendages [m²]

➤ **Step 4 – Determination of R_W**

$$c_7 = \frac{B}{L_{WL}} = 0,1632, \quad \text{for } 0,11 < \frac{B}{L_{WL}} < 0,25$$

$$i_E = 24,80^\circ$$

$$c_1 = 2223105 * c_7^{3,78613} * \left(\frac{T}{B}\right)^{1,07961} * (90 - i_E)^{-1,37565} = 2,266$$

$$A_{BT} = 57,092 \text{ m}^2, \quad T_F = 12 \text{ m}, \quad h_B = 4,832 \text{ m}$$

$$c_3 = 0,56 * A_{BT}^{1,5} / ((B * T * (0,31 * \sqrt{A_{BT}} + T_F - h_B))) = 0,05880$$

$$c_2 = \exp(-1,89\sqrt{c_3}) = 0,6324$$

$$C_M = 0,9961$$

$$c_5 = 1 - 0,8 * \frac{A_T}{B * T * C_M} = 0,8406$$

$$c_{16} = 1,73014 - 0,7067 * C_p = 1,1336, \quad \text{for } C_p > 0,80$$

$$\nabla^{\frac{1}{3}} = \frac{L_{WL}^3}{\Delta} = 133,982$$

$$m_1 = \left(0,0140407 * \frac{L_{WL}}{T}\right) - \left(1,75254 * \frac{\nabla^{\frac{1}{3}}}{L_{WL}}\right) - (4,79323 * B * L_{WL}) - c_{16} = -2,0002$$

$$F_n = \frac{12 * 0,51444}{\sqrt{g * L_{WL}}} = 0,1327$$

$$\lambda = 1,446 * C_p - 0,03 * \frac{L_{WL}}{B} = 1,0367, \quad \text{for } L_{WL}/B < 12$$

$$c_{15} = -1,6939, \quad \text{for } \frac{L_{WL}^3}{\Delta} < 512$$

$$m_2 = c_{15} * C_p^2 \exp(-0,1 * F_n^{-2}) = -0,0041$$

$$R_W = c_1 * c_2 * c_5 * \nabla * \rho_w * g * \exp(m_1 * F_n^d + m_2 * \cos(\lambda * F_n^{-2}))$$

$$R_W = 4,345 [kN]$$

Where,

A_T = Immersed part of the transverse area of Transom [m²], at zero velocity

C_M = Midship section coefficient

F_n = Froude number based on the L_{WL}

A_{BT} = Transverse bulbous area at the position where the still water surface intersects the stern [m²]

T_F = Forward draught of the ship [m]

h_B = Position of the centre of the transverse area [m]

i_E = Angle of the waterline at bow [degrees]

g = Gravitational acceleration [m/s²]

➤ **Step 5 – Determination of R_B**

$$P_B = 0,56 * \frac{\sqrt{A_{BT}}}{T_F - 1,5 * h_B} = 0,8904$$

$$F_{ni} = \frac{V}{\sqrt{(g * (T_F - h_B - 0,25 * \sqrt{A_{BT}}) + 0,15 * V^2)}} = 0,8141$$

$$R_B = 0,11 \exp(-3 * P_B^{-2}) * F_{ni}^3 * A_{BT}^{1,5} * \rho_w * \frac{g}{1 + F_{ni}^2}$$

$$R_B = 3,520 [kN]$$

Where,

P_B = A measure for the emergence of the bow

F_{ni} = Froude number based on the immersion

➤ **Step 6 – Determination of R_{TR}**

$$F_{nT} = \frac{V}{\sqrt{2 * g * \left(\frac{A_T}{(B + (B * C_{WP}))} \right)}} = 1,244$$

$$c_6 = 0,2 * (1 - 0,2 * F_{nT}) = 0,150232, \quad \text{for } F_{nT} < 5$$

$$R_{TR} = 0,5 * \rho_w * V^2 * A_T * c_6 = 0,5 * 1,025 * (12 * 0,51444)^2 * 85,736 * 0,149641$$

$$R_{TR} = 251,57 \text{ [kN]}$$

Where,

F_{nT} = Froude number based on Transom immersion

A_T = Immersed Transom area [m²]

C_{WP} = Waterplane area coefficient

➤ **Step 7 – Determination of R_A**

The allowance resistance, R_A calculated by Holtrop method 1988 [Holtrop,1988]. In this method, the resistance separated two components, which are hull roughness and air resistance. The Hull roughness resistance calculated with following equations,

$$c_4 = 0,04, \quad \text{for } \frac{T_F}{L_{WL}} = 0,054395 > 0,04$$

$$C_A = 0,006 * (L_{WL} + 100)^{-0,16} - 0,00205 + 0,003 * \sqrt{\frac{L_{WL}}{7,5}} * C_B^4 * c_2 * (0,04 - c_4)$$

$$C_A = 0,000267$$

$$R_A = 0,5 * \rho_w * V_{Ship}^2 * (S_{Hull} + S_{App}) * C_A$$

$$R_A = 63,953 \text{ [kN]}$$

As mentioned the beginning of section 5.1, the total bare hull resistance of the ship is composed of components such as; R_F , $(1+k_1)$, R_{APP} , R_W , R_B , R_{TR} and R_A .

$$R_{Total} = R_F * (1 + k_1) + R_{APP} + R_W + R_B + R_{TR} + R_A$$

$$R_{Total} = 352,222 * 1,3796 + 16,025 + 4,345 + 3,52 + 251,57 + 63,953$$

$$R_{Total} = 825,355 \text{ [kN]}$$

Table 5.1 Resistance results for each speed by using Holtrop-Mennen method

Holtrop-Mennen Resistance Components							
V [knots]	Rf*(1+k1) [kN]	Rapp [kN]	Rw [kN]	Rb [kN]	Rtr [kN]	Ra[kN]	Tot.Res. [kN]
6	132,55	4,37	0,00	0,70	73,31	15,99	226,91
7	176,89	5,83	0,00	1,04	97,42	21,76	302,94
8	227,16	7,49	0,02	1,44	124,16	28,42	388,69
9	283,27	9,34	0,11	1,90	153,23	35,97	483,83
10	345,15	11,38	0,48	2,41	184,35	44,41	588,18
11	412,72	13,61	1,59	2,95	217,22	53,74	701,84
12	485,94	16,03	4,35	3,52	251,57	63,95	825,35
13	564,71	18,62	10,23	4,11	287,10	75,06	959,82
14	649,03	21,40	21,13	4,71	323,51	87,05	1106,83

The resistance due to the moonpool and air were also taken into account. The contribution of the moonpool to the total drillship resistance is derived from the results presented in the reference [Veer & Tholen, 2008]. According to the reference, (R. Van't Veer and H.-J. Tholen, 2008) the resistance increase due to the moonpool is expressed as a percentage of the bare hull resistance. The L/B ratio of the moonpool is 2,22 (see section 2.3). For the initial design of the drillship, the added resistance because of moonpool had taken 20% of the bare hull resistance based on the reference (R. Van't Veer and H.-J. Tholen, 2008). In addition, in the CFD tests, the resistance caused by moonpool was calculated as an average of twenty-one percent. This value is very close to the recommended percentage of the reference [Veer & Tholen, 2008] based on moonpool length and beam ratio.

The contribution of the air resistance, Holtrop 1988 method had suggested a simple rule incorporated in the ITTC-1978 method it is suggested to determine the air drag. However, this approach is for normal ships and superstructures. On the other hand, the drillships have special structures above the ship hull. To being more accuracy, another approach proposed by [Fujiwara et al. 2006] was used. The method describes the calculations for wind force parameters such as lateral force coefficient, yaw moment coefficient and roll moment coefficient in following section 5.2. For the resistance point of view, the only lateral force was calculated. In the end, the air resistance also was added to the total resistance.

5.2. Wind Resistance Prediction by Fujiwara et al. 2006

Fujiwara et al. [2006] calculate the wind-induced loads. Fujiwara identifies the components of the external forces and moments acting on the ship. He used subscripts as being the wind, A. The results for the wind resistance in the following equations;

$$F_{XA} = C_{AX}(\Psi_A) * q_A * A_F$$

Where,

F_{XA} = External force for wind

C_{AX} = Longitudinal-force coefficient

Ψ_A = Apparent wind angle

q_A = Aerodynamic pressure

A_F = Frontal projected area

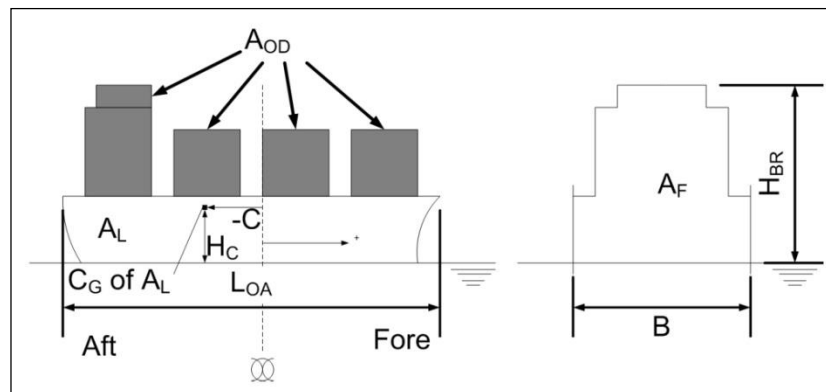


Figure 5.1 Parameters used by Fujiwara et al.[2006]

Table 5.2 Non-dimensional parameters for wind load estimation equations [Fujiwara,2006]

		$j \rightarrow$	0	1	2	3	4
	$i \downarrow$						
α_j			0,404	0,368	0,902		
β_{ij}	1		-0,922	0,507	1,162		
	2		0,018	-5,091	10,367	-3,011	-0,341
γ_{ij}	1		0,116	3,345			
	2		0,446	2,192			
δ_{ij}	1		0,458	3,245	-2,313		
	2		-1,901	12,727	24,407	-40,31	-5,481
ε_{ij}	1		-0,585	-0,906	3,239		
	2		-0,314	-1,117			

The longitudinal force coefficient is defined and solved for 12 knots with average wind speed, 20 knots at 180° head angle as in follows;

$$C_{AX}(\Psi_A) = F'_{LF} + F'_{XLI} + F'_{ALF}$$

$$C_{LF} = \beta_{20} + \beta_{21} * \frac{B}{L_{OA}} + \beta_{22} * \frac{H_C}{L_{OA}} + \beta_{23} * \frac{A_{OD}}{L_{OA}^2} + \beta_{24} * \frac{A_F}{B^2} \text{ for } \Psi_A = 180^\circ$$

$$C_{LF} = 0,018 + \left(-5,091 * \frac{36}{230,89}\right) + \left(10,367 * \frac{3,5}{230,89}\right) + \left(-3,011 * \frac{1000}{230,89^2}\right) + \left(-0,341 * \frac{500}{36^2}\right) = -0,8067$$

$$F'_{LF} = C_{LF} * \cos \Psi_A = -0,8067 * \cos 180 = 0,8067$$

$$C_{XLI} = \delta_{20} + \delta_{21} * \frac{A_L}{L_{OA} * H_{BR}} + \delta_{22} * \frac{A_F}{A_L} + \delta_{23} * \frac{B}{L_{OA}} + \delta_{24} * \frac{A_F}{B * H_{BR}} \text{ for } \Psi_A = 180^\circ$$

$$C_{XLI} = -1,901 + \left(12,727 * \frac{1600}{230,89 * 35}\right) + \left(24,407 * \frac{500}{1600}\right) + \left(-40,31 * \frac{36}{230,89}\right) + \left(-5,481 * \frac{500}{36 * 35}\right) = -0,21404$$

$$F'_{XLI} = C_{XLI}(\sin \Psi_A - 1/2 \sin \Psi_A * \cos^2 \Psi_A) * \sin \Psi_A * \cos \Psi_A$$

$$F'_{XLI} = -0,21404 * (\sin 180 - 1/2 \sin 180 * \cos^2 180) * \sin 180 * \cos 180$$

$$F'_{XLI} = 1,6064 * 10^{-13} \approx 0$$

$$C_{ALF} = \varepsilon_{20} + \varepsilon_{21} * \frac{A_{OD}}{L_{OA}} \quad for \quad \Psi_A = 180^\circ$$

$$C_{ALF} = -0,314 + \left(-1,17 * \frac{1000}{230,89} \right) = -1,01213$$

$$F'_{ALF} = C_{ALF} * \sin \Psi_A * \cos^3 \Psi_A$$

$$F'_{ALF} = -1,01213 * \sin 180 * \cos^3 180$$

$$F'_{ALF} = -1,24 * 10^{-16} \approx 0$$

$$C_{AX}(\Psi_A) = F'_{LF} + F'_{XLI} + F'_{ALF}$$

$$C_{AX}(\Psi_A) = 0,8067 + 0 + 0$$

$$C_{AX}(\Psi_A) = 0,8067$$

$$q_A = \frac{1}{2} * \rho_A * (U_A + V_{ship})^2$$

$$q_A = \frac{1}{2} * 0,001225 * ((20 + 12) * 0,51444)^2$$

$$q_A = 165,988 \left[\frac{kg}{s^2 * m} \right]$$

$$F_{XA} = C_{AX}(\Psi_A) * q_A * A_F$$

$$F_{XA} = 0,8067 * 165,988 * 500$$

$$F_{XA} = 66,949 [kN]$$

Where;

ρ_A = Air density [g/cm³]

U_A = Apparent wind velocity [m/s]

V_{ship} = Design ship speed [m/s]

A_L = Frontal cross-sectional area [m²]

A_{OD} = Over deck cross-sectional area [m²]

H_{BR} = Bridge height [m]

H_C = Vertical distance from waterline to aerodynamic pressure center [m]

F'_{LF} = Longitudinal flow drag

F'_{XLI} = Lift-induced drag

F'_{ALF} = Additional longitudinal drag

All parameters of total resistance were obtained for an each speed of the drillship as shown in Figure 5.2 and Table 5.3, respectively. The blue area represents the ship hull resistance calculated by Holtrop Mennen method. The Red zone represents the percentage of the moonpool's resistance, which is estimated as %20 percent based on moonpool dimension by the reference [Veer & Tholen, 2008], affecting the total resistance of the ship. Lastly, the brown zone represents wind resistance, which is calculated by another method [Fujiwara, 2006].

Table 5.3 Total Resistance components for each speed

V [knots]	Total Resistance by Holtrop [kN]	Moonpool Resistance [kN]	Wind Resistance by(Fujiwara) [kN]	Total Resistance [kN]
6	226,91	45,38	44,20	316,49
7	302,94	60,59	47,66	411,19
8	388,69	77,74	51,26	517,68
9	483,83	96,77	54,98	635,58
10	588,18	117,64	58,84	764,65
11	701,84	140,37	62,83	905,03
12	825,35	165,07	66,95	1057,37
13	959,82	191,96	71,20	1222,99
14	1106,83	221,37	75,58	1403,78

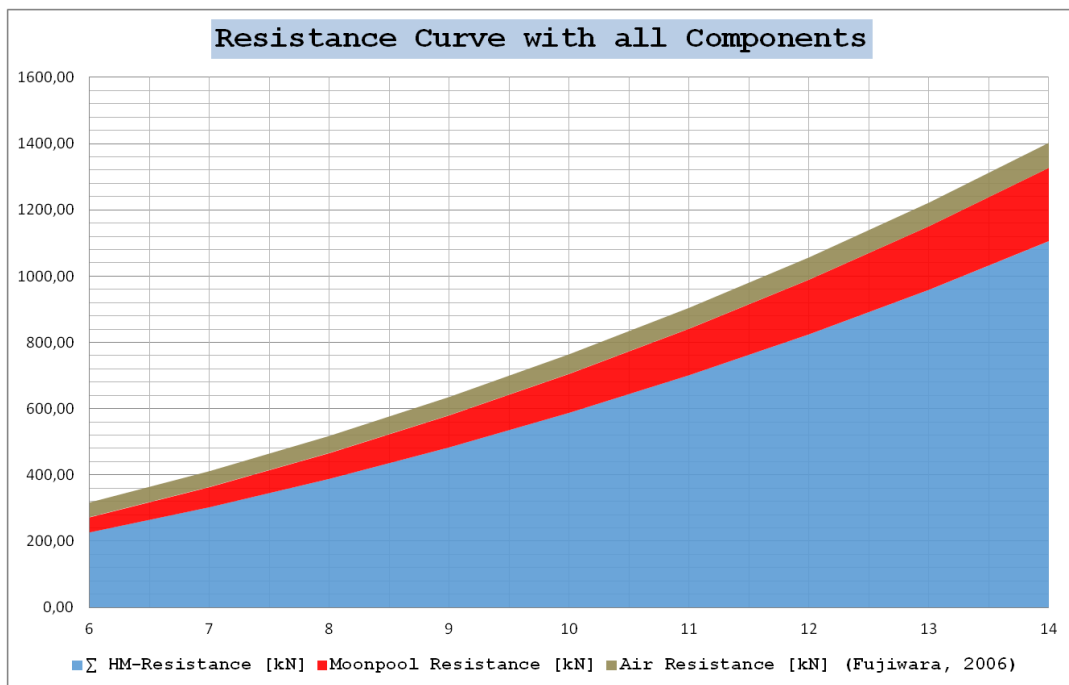


Figure 5.2 Total Resistance Curve with all components based on Holtrop-Mennen

5.3. Power prediction by Computational Fluid Dynamics (CFD)

As mentioned before, the focuses will be on resistance effect of moonpool and flow behavior in the moonpool to define the moonpool characteristics for preliminary design stage. The dimensions of the moonpool and its properties are described in the one of the previous section 2.3. Two identical ship models were used in resistance calculations during this study were designed as with moonpool and without moonpool. The models were exported to computational fluid dynamic software NUMECA/Fine Marine V5.1, which provides resistance calculations and observations of flow behavior around ship and moonpool. The ship hull and moonpool mesh for simulation are shown in Figure 5.3 and this mesh setup was used for all the different ship speeds, with a varying number of cells around moonpool walls and critical locations. Inside of the moonpool, there are many circulations of the flow and also the presence of the free surface that leads to high requirements on quality of the mesh.

The literature shows that generally research on moonpools has been carried out using experimental model tests, which are closest to reality. The resistance results and observed flow behavior in moonpool available in this study were compared with the results from such model tests [Veer & Tholen, 2008] & [Hammargren & Tornblom, 2012]. The accuracy of the results presented in the thesis study was checked by this method for the simple design phase.

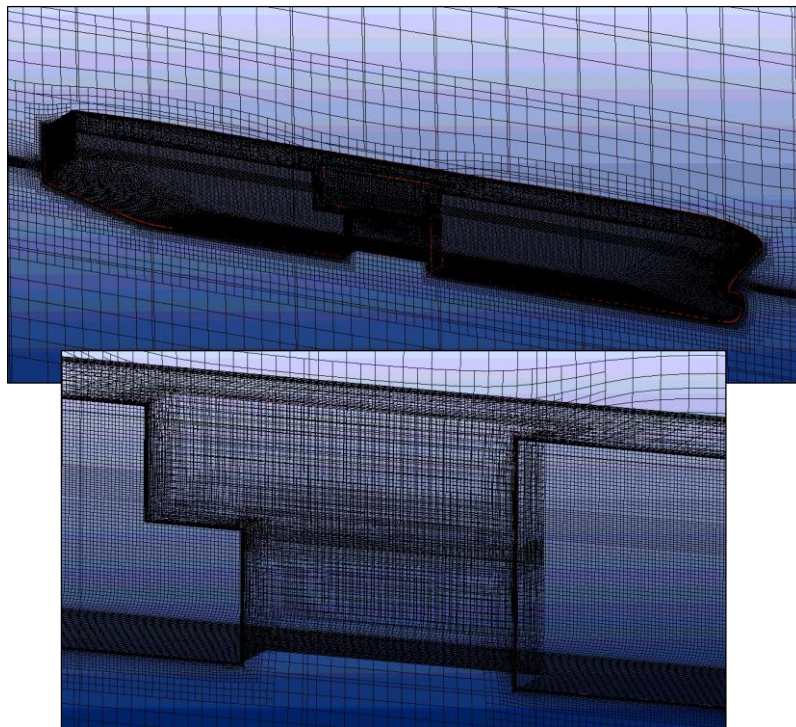


Figure 5.3 Mesh for Ship bare Hull and Moonpool area

5.3.1. Drillship resistance without Moonpool

Table 5.4 CFD Results without Moonpool

V [knots]	CFD - Resistance [kN]
6	132,20
7	177,60
8	245,10
9	288,86
10	372,78
11	432,62
12	514,06
13	610,88
14	740,21

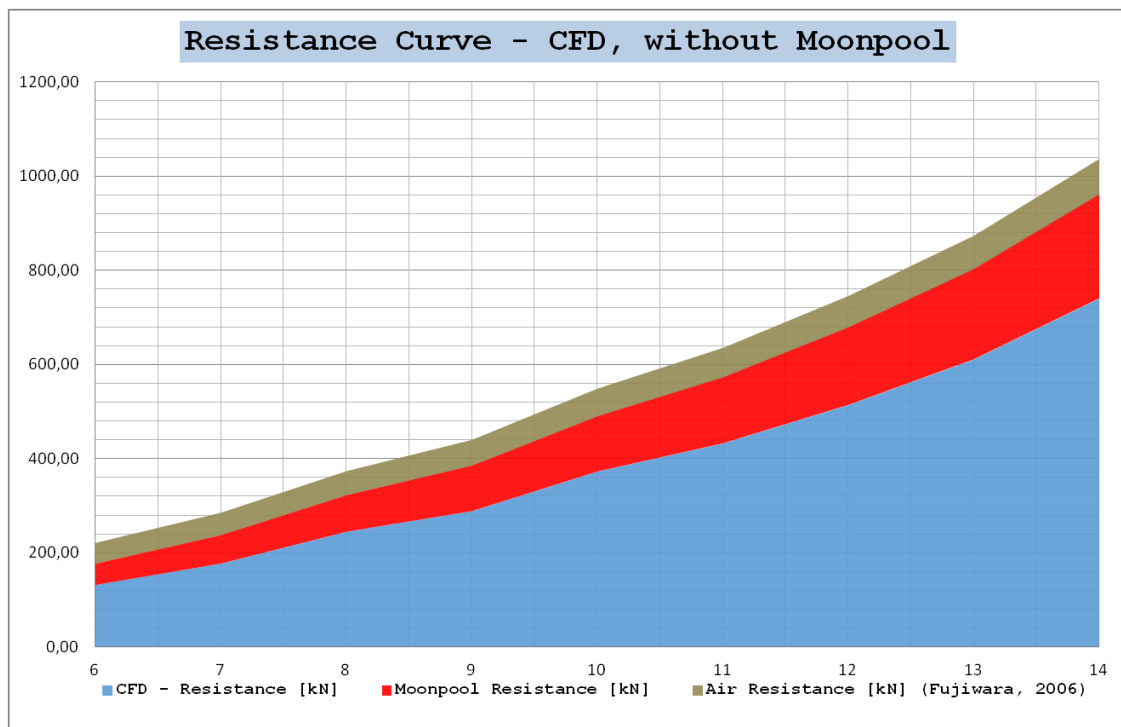


Figure 5.4 Drillship Hull Resistance results without Moonpool – CFD

Table 5.4 shows the results of the resistance of the ship designed without the moonpool. The blue area represents the ship hull resistance calculated by Holtrop Mennen method. The Red zone represents the percentage of the moonpool's resistance, which is estimated as %20 percent based on moonpool dimension by the reference [Veer & Tholen, 2008], affecting the total resistance of the ship. Lastly, the brown zone represents wind resistance, which is calculated by another method [Fujiwara, 2006].

5.3.2. Drillship resistance with Moonpool

Table 5.5 CFD Results with Moonpool

V [knots]	CFD - Resistance [kN]
6	172,98
7	230,64
8	290,60
9	365,02
10	513,06
11	549,80
12	658,76
13	791,10
14	967,08

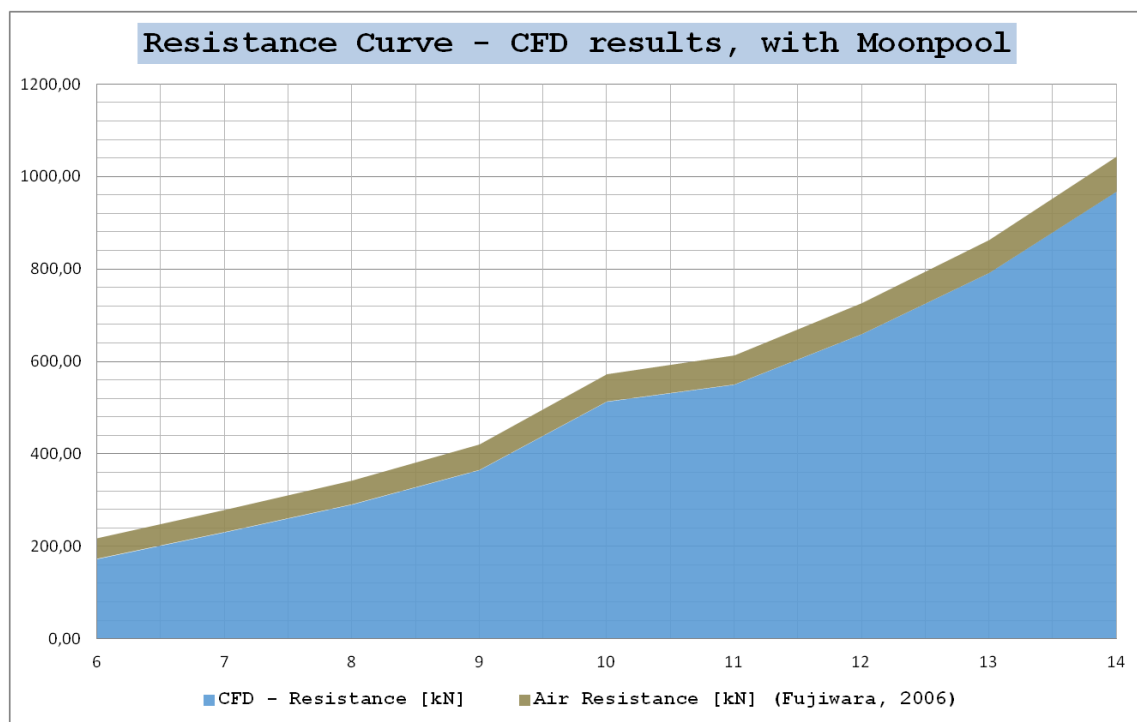


Figure 5.5 Drillship Hull Resistance results with Moonpool - CFD

Table 5.5 shows the results of the resistance of the ship designed with the moonpool. In addition, the blue area shown in Figure 5.4 represents the total ship resistance calculated by CFD method for drillship with moonpool. The brown zone represents wind resistance, which is calculated by another reference [Fujiwara, 2006].

5.3.2.1. Moonpool effect on ship resistance

One of the most important aims of this thesis study is to investigate and calculate the effect of moonpool on ship resistance by CFD method. The resistance caused by moonpool increases with the effect of increasing surface area. According to reference [Veer & Tholen, 2008], there are two fundamental water behaviors that can directly affect this resistance. The first one is the piston effect, and the other is the sloshing effect. It is seen that the force acting according to the moonpool dimensions of the designed drillship model is sloshing based on this reference. With this assumption, the cause of the incensement in ship resistance shown in Table 5.5 is the water oscillation in the moonpool. Also, the flow around the moonpool very complicated and included both strong vortices together with a free surface, which is shown in Figure 5.5 with different views.

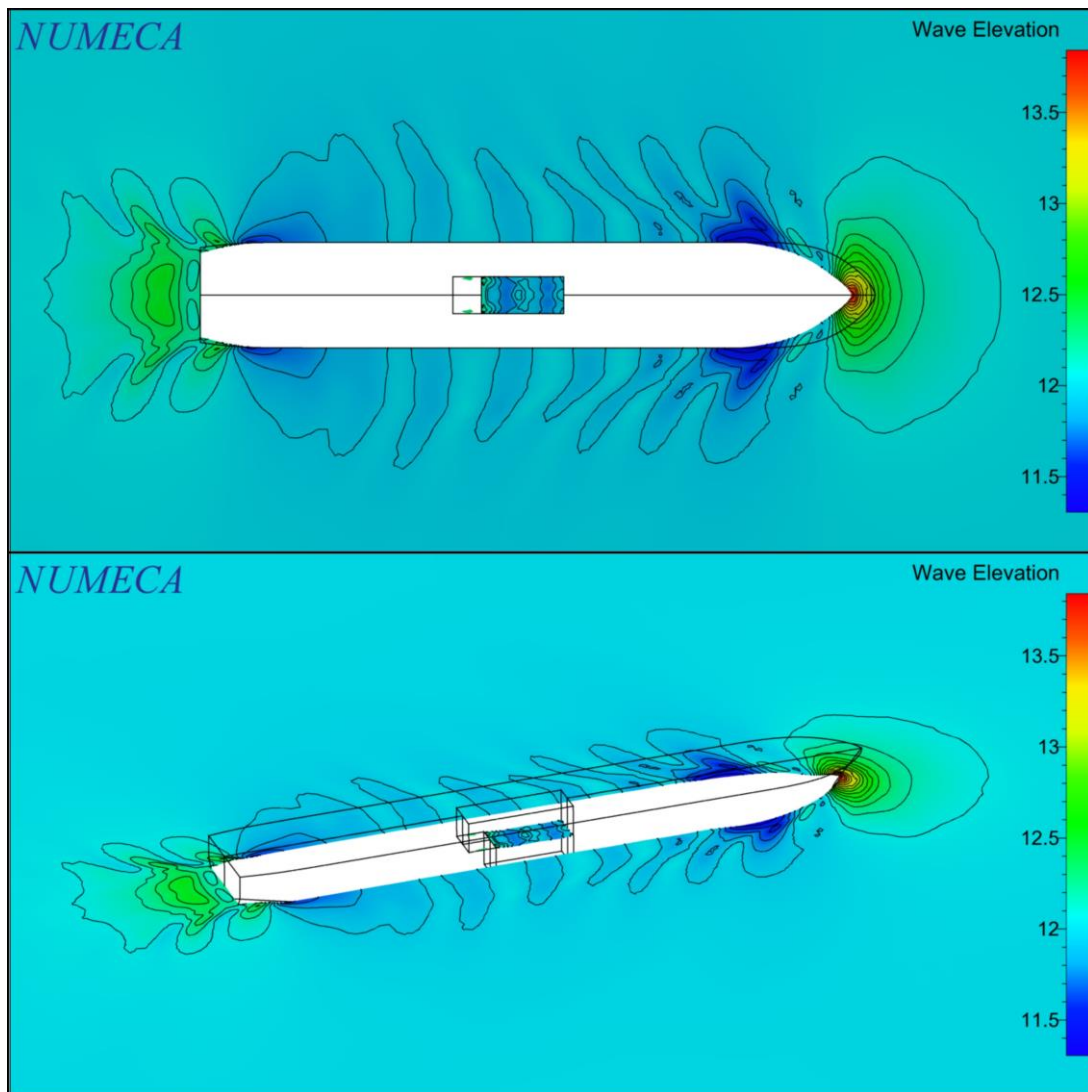


Figure 5.6 Free surface effect around the Drillship bare hull

5.3.2.2. Flow behavior inside of the Moonpool

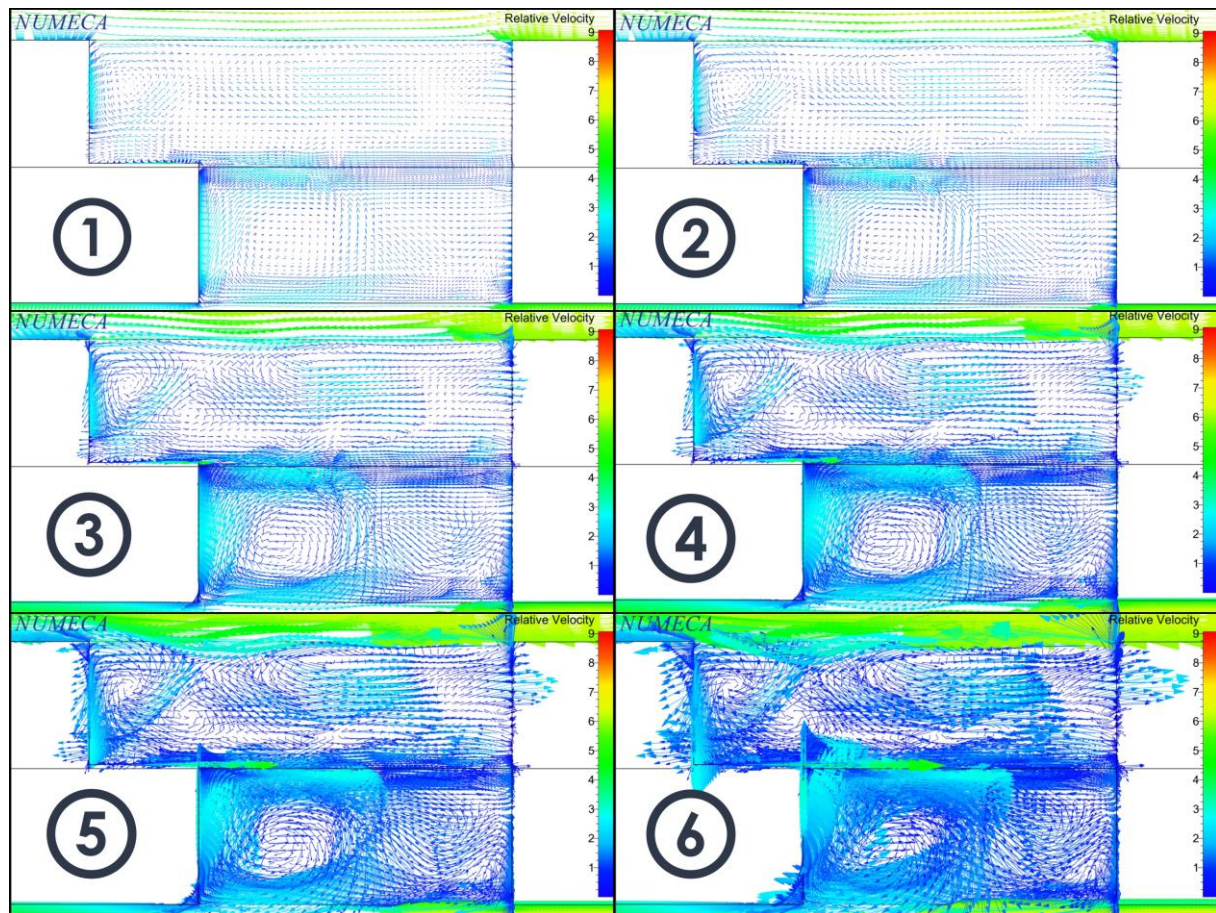


Figure 5.7 Initiation of vortices inside Moonpool at design speed

In this study, the flow behavior in the moonpool has been investigated by the same use method. The movement of the free surface is initiated by vortices created at the aft, and fore edge of moonpool are shown in Figure 5.7, respectively. As mentioned above, the sloshing effect is due to observed vortices. A study to reduce this oscillation might be another study topic; such work is not done for the preliminary design phase. Also, in this figure, the design water line can be seen as a black line which located below the recess. By varying the height of waterline or the recess, the amount of flow allowed to enter above the recess is varied. The comparison of the simulations for the flow in the moonpool, a master thesis [Hammargren & Tornblom, 2012] has been used which presents results and figures for another CFD simulations with different recess moonpool types.

5.4. Conclusions for Resistance results

As mentioned before, in the thesis the resistance of a drillship has been investigated by the statistical method is called Holtrop-Mennen and computational fluid dynamics (CFD) method was performed for both, drillship bare hull with and without moonpool.

Holtrop -Mennen Method – Taking into account the parameters used to calculate this method, the ship's resistance increases greatly depending on the displacement. It is impossible to formulate the resistance of the moonpool on a drillship, numerically. So, the approximate value observed in another study carried out and calculated according to the main dimensions of the moonpool was added to the total ship resistance. For these vessels in these main dimensions, the calculated resistance values are in a range that can be normal. But the results obtained from the Holtrop method will not be used for machine selection, assuming that the results calculated in the CFD method may also be closer to the truth.

CFD without moonpool – In the drill ship modeled for CFD operation, appendages resistance is not considered. Therefore, to obtain a more realistic approach, the R_{app} value calculated in the Holtrop method is also subtracted from the total ship resistance calculated by this method. However, there is a difference of 36 percent between the two approaches at design speed. The main reason for this difference may be the constant values on which the numerical approach is based such as, numerically estimated block and volumetric coefficients. Table 5.6 illustrated that all errors between two approaches for some other needed speeds. The error rate between the two approaches is visibly reduced compared to the increased speed values. For this, it may be appropriate to calculate both calculation methods at increasing speeds.

Table 5.6 Comparison between Holtrop Mennen and CFD Without Moonpool

V [knots]	HM-Resistance [kN]	CFD - without Moonpool [kN]	Error %
6	222,540	132,20	40,59%
7	297,106	177,60	40,22%
8	381,196	245,10	35,70%
9	474,485	288,86	39,12%
10	576,796	372,78	35,37%
11	688,224	432,62	37,14%
12	809,330	514,06	36,48%
13	941,200	610,88	35,10%
14	1085,426	740,21	31,80%

CFD with moonpool – The same ship model has been simulated with the same CFD parameters so that only the effect of the moonpool on the ship resistance can be observed. Table 5.7 shows clearly that the resistance trends at the calculated speeds are close to each other for the same speeds of ships with moonpool and those without moonpool. This value varies according to the calculated speed of the ship and is approximately 15 to 23 percent. These calculated values are carried out on similar moonpool dimensions as used in a model test constructed by reference [Veer & Tholen, 2008]. As mentioned in section 5.3.1.1, the flow in the moonpool also affects moonpool resistance based on dominate oscillation modes of water such as a piston and sloshing [Veer & Tholen, 2008].

Table 5.7 Comparison between CFD With Moonpool and CFD Without Moonpool

V [knots]	CFD - with Moonpool [kN]	CFD - without Moonpool [kN]	Error %
6	172,98	132,20	23,57%
7	230,64	177,60	23,00%
8	290,60	245,10	15,66%
9	365,02	288,86	20,86%
10	513,06	372,78	27,34%
11	549,80	432,62	21,31%
12	658,76	514,06	21,97%
13	791,10	610,88	22,78%
14	967,08	740,21	23,46%

It should be underlined that the ship resistance value determined by the CFD method also, the other important resistance parameter, which is the wind, has been added to total ship resistance to select the thrusters as shown in Table 5.8.

Table 5.8 Drillship resistance with added wind resistance to estimate thruster power

V [knots]	CFD - with Moonpool [kN]	Wind Resistance by(Fujiwara) [kN]	Total Resistance [kN]
6	172,98	44,20	217,18
7	230,64	47,66	278,30
8	290,60	51,26	341,86
9	365,02	54,98	420,00
10	513,06	58,84	571,90
11	549,80	62,83	612,63
12	658,76	66,95	725,71
13	791,10	71,20	862,30
14	967,08	75,58	1042,66

5.5. Propulsion system

5.5.1. Thrusters

As mentioned before, thrusters are one of the most important equipment for drilling rigs. For dynamically positioned drillship, the choice of thruster that can provide enough thrust is a crucial step. The major reasons for the selection of thrusters for this type of ship can be summarized as follows:

- The minimum impact in the general arrangement
- Capability of dynamic position operation
- Efficiency in transit operation
- Direct thrust at any angle
- Operation ability in all kinds of environmental conditions

The characteristic features of the propeller calculated by the AVEVA software are included in next the sections of the study. For the drillship, only two of the six thrusters were used for the transit operation.

5.5.1.1. Thruster design

By varying the characteristics of the propeller, an optimum thruster was developed by maximizing the open water efficiency based on AVEVA determination. The optimum propeller characteristics determined are outlined in Table 5.9 and Figure 5.5, respectively.

Table 5.9 Optimum propeller characteristics based on AVEVA

Parameters	Values
Diameter [m]	4,20
Pitch ratio (P/D)	0,796
Effective BAR	0,672 (0,667 min)
Number of Blades	4
Thrust load. coeff.	0,184 (0.185 max)
K_t/J^2	1,409
Adv. coeff. J	0,390
Thrust coeff. K_t	0,215
Torque coeff. K_q	0,0285
Open water efficiency	0,468

The thrusters developed in previous section 5.5.1.1 the required thrust to overcome the ship's resistance at 12 knots, which is design speed. Figure 5.5 illustrated those,

- During the operating at a low advance coefficient; thrusters develop higher thrust and
- Comparison K_t , K_q and their effect on open water efficiency the propeller curves at design speed for a Wageningen B-series propeller.

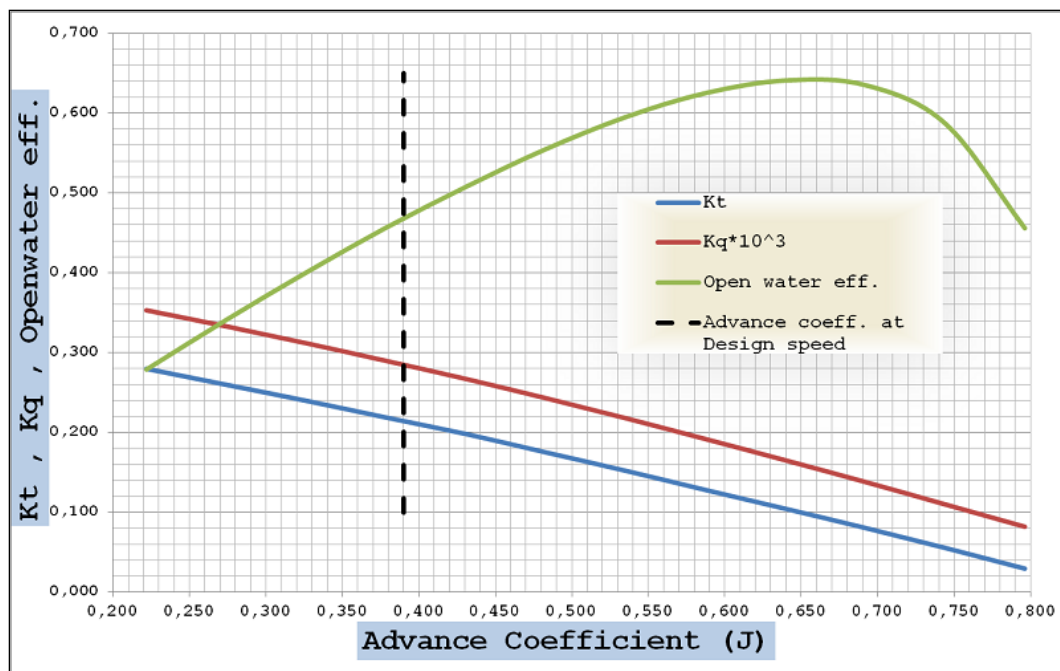


Figure 5.8 K_t & K_q curve by Wageningen B-series propeller [AVEVA]

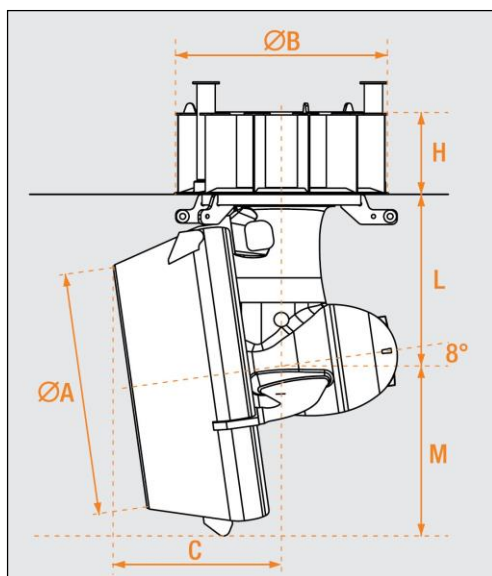


Table 5.10 Technical specification of selected Thruster

Thruster type	WST-65U
Thruster power range [kW]	5000-6500
A [mm]	4200
B [mm]	3820
C [mm]	3006
M [mm]	2716
L [mm]	3025
H [mm]	1390

Figure 5.9 Selected thruster Available from
<http://www.wartsila.com/products/marine-oil-gas/propulsors-gears/thrusters/>

5.5.2. Powering

5.5.2.1. Requirements

As previously determined, two thrusters will provide the propulsion. For an overview of the thruster, their configuration is shown Table 6.1. Also, Table 5.11 illustrated that needed parameters to estimate minimum brake power required. According to resistance calculation, the transit condition requires 725, 71 kN at transit speed.

Table 5.11 Determined parameters by AVEVA to estimate required Brake Power at design speed

Parameters	Values
Propeller Diameter [m]	4.200
Open water efficiency (η_o)	0.468
Wake fraction (w)	0,243
Thrust deduction (t)	0,243
Transit speed [knots]	12

Table 5.12 Efficiencies used in Brake Power Calculation [Parsons, 2002]

Parameters	Values
Mechanical efficiency of Thruster	0,95
Load correction factor	0,99
Gearing (one reduction & reversing)	0,985
Electrical Transmission and Conversion	0,985
Generator	0,96
Electric Motor	0,97

$$P_E = R_T * V$$

$$P_E = 725,71 \text{ kN} * (12 * 0,51444) \text{ m/s} = 4480,005 \text{ kW}$$

$$V_A = V * (1 - w)$$

$$V_A = (12 * 0,51444) \text{ m/s} * (1 - 0,243) = 4,673 \text{ m/s}$$

$$T = \frac{R_T}{(1 - t)}$$

$$T = \frac{R_T}{(1 - t)} = \frac{725,71 \text{ kN}}{(1 - 0,243)} = 958,664 \text{ kN}$$

$$P_T = T * V_A$$

$$P_T = 958,664 \text{ kN} * 4,673 \text{ m/s} = 4480,005 \text{ kW}$$

$$P_D = \frac{P_T}{\eta_o}$$

$$P_D = \frac{4480,005 \text{ kW}}{0,468} = 9572,66 \text{ kW}$$

$$P_B = \frac{P_D}{\eta_{Thruster} * \eta_{LC} * \eta_{Gear} * \eta_{Trans\&Conv} * \eta_{Generator} * \eta_{Motor}}$$

$$P_B = \frac{9572,66 \text{ kW}}{0,8242} = 11614,1296 \text{ kW}$$

Where,

R_T = Ship Resistance [kN]

P_E = Effective Power [kW]

P_T = Thrust Power [kW]

P_D = Delivered Power [kW]

P_B = Brake Power [kW]

V_A = Velocity of Advance [m/s]

T = Thrust [kN]

The Brake Power calculation was performed for the transit condition, and it will check later in the dynamic position analysis to ensure adequate brake power is installed. The above calculation shows that each thruster selected in Table 5.10 should provide 5807.06 kW, which is half of the estimated brake power (P_B). That means only two thrusters must be provided when the ship is transitting. It should be underlined that different efficiencies according to reference [Parsons, 2002] have been used for electrical equipment in the power transfer and these efficiencies are shown in Table 5.12.

5.5.2.2. Diesel generating sets and endurance

At this part of work, the choice of the diesel generating sets to provide the adequate power that has been investigated from the sources. Also, the generating set has to provide adequate power to remain on station in extreme weather conditions. The generating sets, which best meet the power requirements are six Wartsila 12V32, which can provide 6720 kW per engine. Detailed information about selected diesel generating set can be found in Appendix 6.

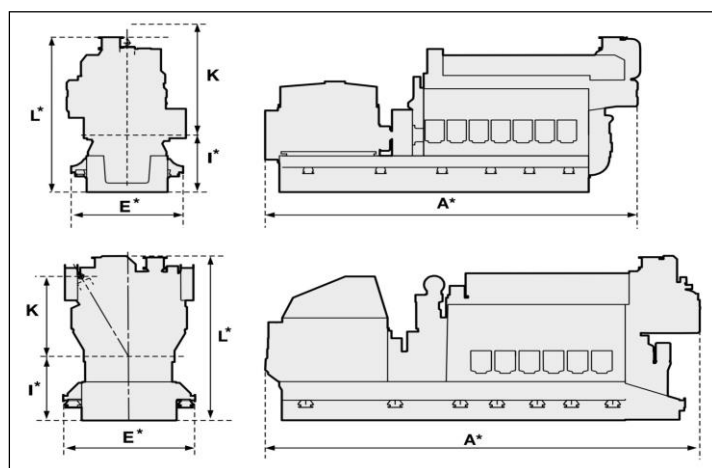


Table 5.13 Wartsila Genset 32 Main Dimensions and Weight

Parameters	Values
Engine type	12V32
A [mm]	10700
E [mm]	3060
I [mm]	1700
K [mm]	2120
L [mm]	4130
Weight [tonnes]	100

Figure 5.10 Wartsila Genset Main Dimensions Available from <http://www.wartsila.com/products/marine-oil-gas/engines-generating-sets/diesel-engines/wartsila-32>

Although the drillship's transit distance for the operation was specified in the company requirements, it is significant to ensure that these engines will not exceed the fuel consumption for the defined endurance. The operation area and prescribed endurance information can be found in Appendix 6. The endurance calculation to verify the ship's range of 11800 nautical miles is shown in following equations. According to diesel generating the producer company, using the fuel consumption rate of each genset is 174 g/kWh and knowing that only two engines are required to be running when transiting.

$$\text{Range} \rightarrow 11800 \text{ nm at 12 knots} \rightarrow 11800/12 = 983,33 \text{ hours}$$

$$\text{Fuel consumption} = 174 \text{ g/kWh} * (1E - 6) \text{ tonnes/s} * 6720 \text{ kW/engine}$$

$$\text{Fuel consumption} = 1,169 \text{ tonnes/hour}$$

$$\text{Total fuel consumption} = 1,169 \text{ tonnes/hr} * 2 \text{ engines} * 983,33 \text{ hours}$$

$$\text{Total fuel consumption} = 2299,58 \text{ tonnes}$$

6. DYNAMIC POSITION PREDICTION

Estimation of the dynamic position power requirement for drillships is one of the most important topics in this study. The environmental loads, which are the wind, waves, and current forces, should be analyzed together with the propulsion-located thrusters configuration of the drillships.

The dynamic positioning system is one of the complex computer controlled system, which can control automatically maintain a drillship position and heading by using its propellers and thrusters. Its need to be added that any vessel has six freedoms of movement, called yaw, surge, sway, heave, pitch and roll, respectively. The dynamic positioning systems are not able to control the heave, pitch, and roll of the vessel. These movements are concerned within the context of sea keeping the behavior of the ship [Steddum & Herrmann, 1997].

The results and approaches will present in following sections, which clarify the vessel can maintain position and keep heading with the riser connected and normal drilling operations when submitted to the worst annual environmental loads acting in the same direction for the following areas;

- Offshore Brazil
- The Gulf of Mexico
- Southeast Asia
- West and Central Africa
- South Africa
- Mediterranean Sea
- Australia

The main elements of a Dynamic positioning system are, Sensor system, Power system, controller, and Thruster system, to control the yaw, surge, and sway by using thrusters of the drillship automatically. Those three axes are concerned primarily for drillship in the horizontal plane [Steddum & Herrmann, 1997].

6.1. Axes Convention and Thrusters Configuration

The axis convention used for the dynamic positioning calculation shown in Figure 6.1 [de Wit, 2009]. The forces and moments are calculated in the fixed vessel reference system. The right-hand sign convention is used for the horizontal forces, F_x , F_y and the horizontal moment M_z . All forces and moments are given about the center of gravity of the vessel. As mentioned in section 5.5.1.1, the drillship is equipped with six 6000 kW azimuth thrusters. The thruster configuration and its particulars are illustrated in Figure 6.1 and Figure 6.2, respectively.

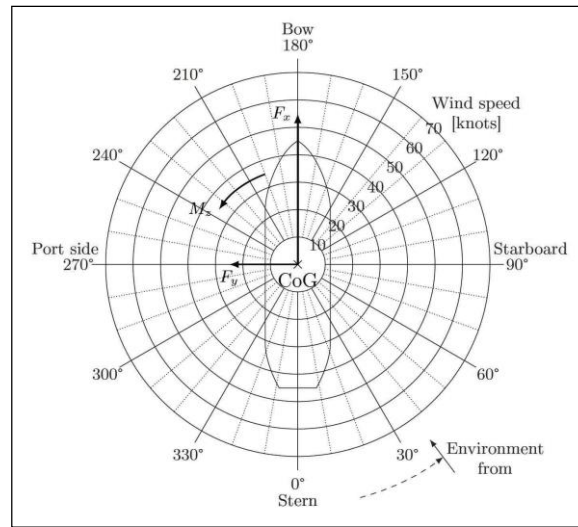


Figure 6.1 Axes convention for Dynamic Position Calculations [de Wit, 2009]

Table 6.1 Thrusters Particulars

Thruster Number	Position	X cord. From APP [m]	Y cord. From CL [m]	Power [kW]
1	PS aft	4,20	8,40	6000
2	SB aft	4,20	-8,40	6000
3	CL aft	56,70	0	6000
4	PS fwd	165,90	9,10	6000
5	SB fwd	165,90	-9,10	6000
6	CL fwd	210,70	0	6000

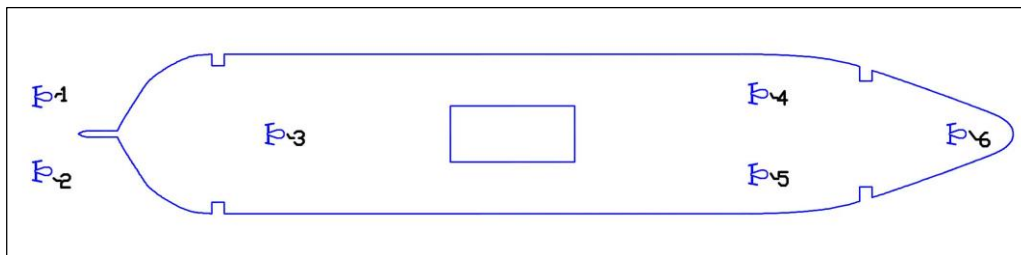


Figure 6.2 Thrusters Configurations

6.2. Thrust and Power

6.2.1. *Effective Thrust*

In dynamic position calculations, there are several thrust degradation effects which should be accounted, such as thruster-hull interaction, thruster-thruster interaction, thruster-wave interaction and thruster-current interaction [12]. An approach will be made in the following sections of this Study by using AVEVA Hydrostatic and Hydrodynamic Module for some of the forces that cause these interactions. The finding of hydrodynamic constants is a very important information and is very useful for the further design phase of the ship.

Based on a thrust – power ratio of 0,177 kN/kW which is typical for Dynamic Position (DP) calculations the selected thrusters are capable delivering 1062 kN of effective thrust at zero speed.

6.2.2. *Design requirements*

As mentioned before, the ability of a drillship to keep on the station is a critical design criterion. That's why the vessel will be designed for DP class 3 that can be defined by automatic and manual position and heading control under specified maximum environmental conditions. At the very begin of the chapter, most important parameters of DP systems was listed, respectively. To perform a design requirement of the drillship shall be able to maintain position and keep heading towards the lesser environment resulting forces within a sector of $45.0^{\circ} (\pm 22.5^{\circ})$. After worst DP3 failure, the vessel also should require the forces within a sector of $30.0^{\circ} (\pm 15.0^{\circ})$. The failures, which will be considered for the DP class 3, are designed such that a designated combination of failing thrusters is presented with all details in Appendix 6. Least & most effective thrusters and least & most effective pairs of thrusters will be available in the conclusion part of the study, based on AVEVA dynamic position estimation.

6.3. Environmental Conditions

6.3.1. IMCA and ITTC Conditions

The following table has illustrated that relationship between significant wave height, wave period and wind speed which are defined based on International Marine Contractors Association (IMCA) [12]. Also, a constant current velocity is defined. According to the sea conditions can be summarized as surface current speed [V_c] is 1 knots, Peak enhancement factor [γ] is 3,5 also the wind, waves and current are collinear.

Table 6.2 International Marine Contractors Association (IMCA) sea state conditions

Sig. Wave Height, H_s [m]	Crossing Period, T_z [s]	Peak Period T_p [s]	Wind Speed, V_w [m/s]
0	0	0	0
1,28	4,14	5,3	2,5
1,78	4,89	6,26	5
2,44	5,72	7,32	7,5
3,21	6,57	8,41	10
4,09	7,41	9,49	12,5
5,07	8,25	10,56	15
6,12	9,07	11,61	17,5
7,26	9,87	12,64	20
8,47	10,67	13,65	22,5
9,75	11,44	14,65	25

Table 6.3 ITTC Spectrum based on AVEVA module [Wind-Wave relationship]

Significant Wave Height [m]	Average Wave Period [sec]	Wind Speed [knots]	Wind Speed [m/s]
0,00	5,50	0	0,00
0,86	5,93	5	2,57
1,40	6,20	10	5,14
1,97	6,49	15	7,72
2,68	6,84	20	10,29
3,43	7,22	25	12,86
4,17	7,59	30	15,43
5,01	8,01	35	18,01
5,80	8,40	40	20,58
6,66	8,83	45	23,15
7,26	9,13	50	25,72

6.3.2. Weather Conditions

As mentioned before, operational areas have been specified at the beginning of the chapter. About this, the weathering requirements are illustrated for two conditions of Brazil in Table 6.4 and worldwide environmental conditions in Table 6.5 and Table 6.6, respectively.

Table 6.4 Offshore Brazil worst annual conditions [18] & [19]

Parameters	Offshore Brazil Condition A	Offshore Brazil Condition B
Significant Wave Height H_s [m]	6,0	4,5
Peak Period T_p [s]	9 -15	9 -15
Peak Enh. Factor, γ	2,2 – 1,7	2,2 -1,7
Wind Speed [m/s] ten-min	19,0	18,0
Wind Speed [m/s] one-min	20,9	19,8
Surf current speed V_c , [m/s]	1,2	1,5

Table 6.5 Worldwide worst annual conditions group 1 [18] & [19]

Parameters	The Gulf of Mexico	South-East Asia	West/ Central Africa
Significant Wave Height H_s [m]	4,4	3,9	3,4
Peak Period T_p [s]	8,8	8,5	12,9
Peak Enh. Factor, γ	2,0	2,6	4,0
Wind Speed [m/s] ten-min	22,7	20,0	22,7
Wind Speed [m/s] one-min	25	22,0	25,0
Surf current speed V_c , [m/s]	1,5	1,3	1,5

Table 6.6 Worldwide worst annual conditions group 2 [18] & [19]

Parameters	South Africa	Mediterranean Sea	Australia
Significant Wave Height H_s [m]	6,0	4,5	4,2
Peak Period T_p [s]	12,0	10,0	9,9
Peak Enh. Factor, γ	1,0	1,5	1,3
Wind Speed [m/s] ten-min	19,8	21,8	17,3
Wind Speed [m/s] one-min	21,8	24,0	19,0
Surf current speed V_c , [m/s]	0,6	0,5	0,7

The values are shown in Table 6.3, Table 6.4 and Table 6.5 were taken from websites serving as open source [18] and [19]. One-min value is usually used when calculating the dynamic position.

6.4. Environmental Loads

Three environmental loads are considered for the Dynamic position estimation. To determine the coefficients of the loads, AVEVA Hydrostatics and Hydrodynamic Calculation module was used.

6.4.1. Wave Drift Force

The wave drift force coefficients for the typical operating draught, an $H_s = 5,8$ meters is shown in following Figure 6.3 for the three degrees of freedom.

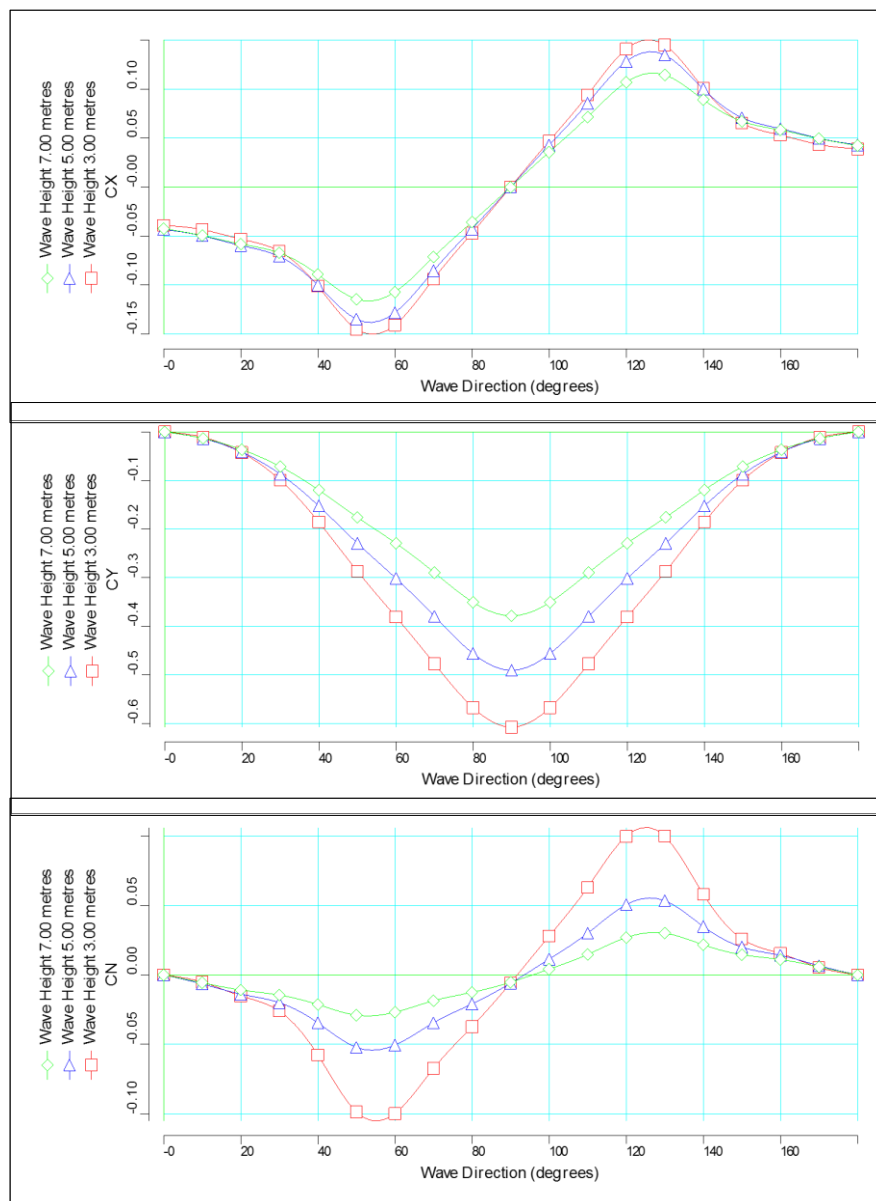


Figure 6.3 Estimated wave drift force coefficients - C_{fx} , C_{fy} , C_{fn}

6.4.2. Wind Forces

The wind profile was drawn based on general arrangement plan and the ship profile by using AutoCAD software. This drawing was also used to make wind resistance calculations with Fujiwara, 2006 method. The results of the wind forces calculation are presented in following Figure 6.4. The wind loads are calculated by using these force and moment coefficients with the actual wind speed which is updated to software by manually.

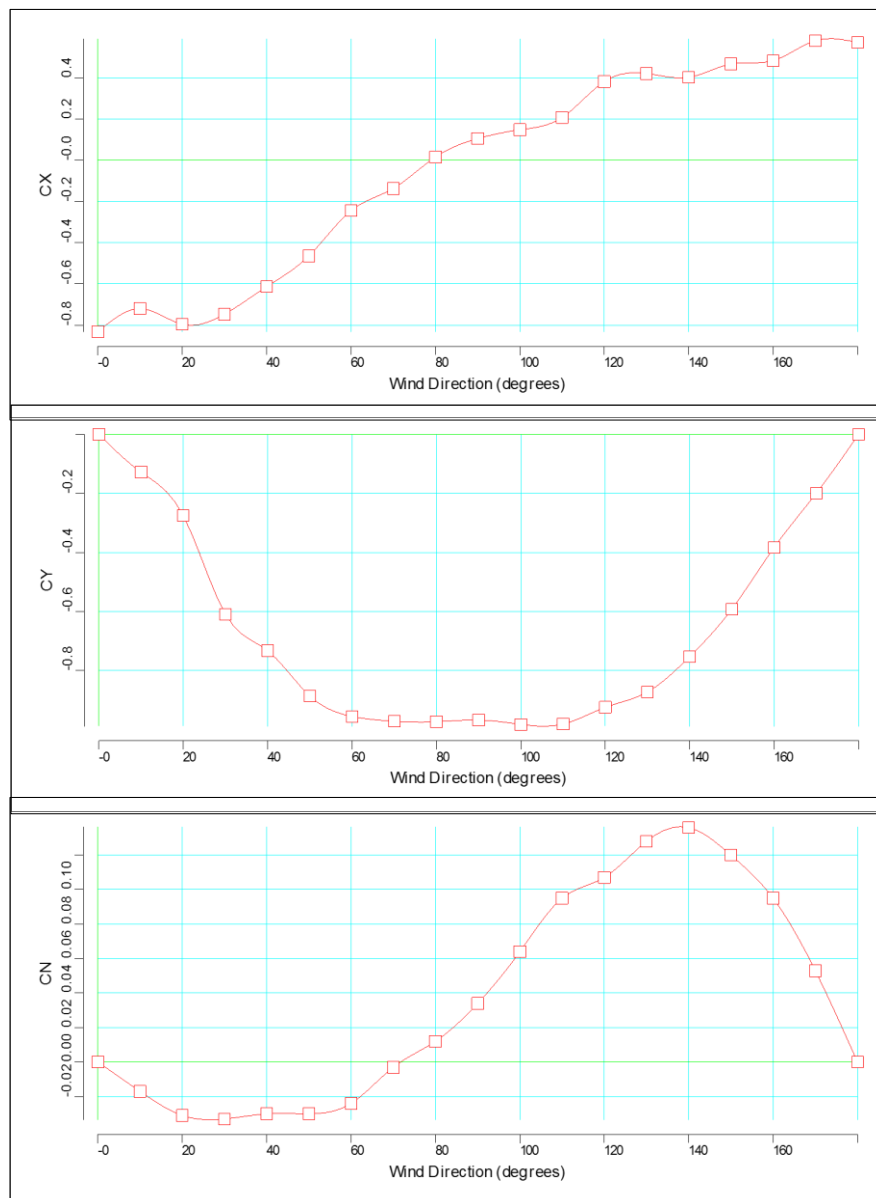


Figure 6.4 Estimated wind forces coefficients – C_{fX} , C_{fY} , C_{fN}

6.4.3. Current Forces

The results of the current forces calculation are presented in following Figure 6.5.

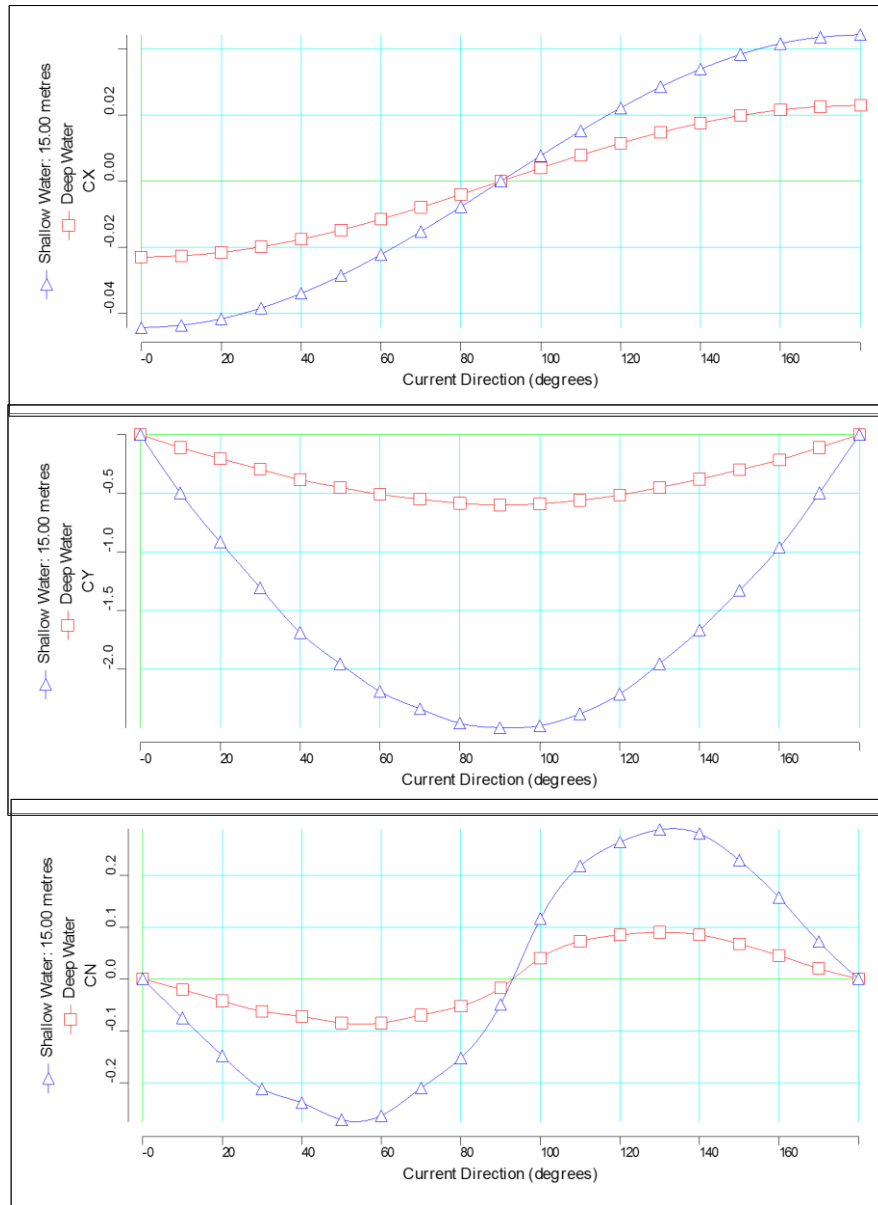


Figure 6.5 Estimated current force coefficients – C_{fX} , C_{fY} , C_{fN}

All environmental conditions can be summarized in Table 6.3 as, Significant Wave Height 5.80 meters coincident with wind direction, Current speed 1 knots coincident with the wind, Wind speed 40 knots between 0 to 360 degrees and Wave Period 8.40 seconds.

6.5. Conclusions for Dynamic Position Estimation

AVEVA Hydrostatics and Hydrodynamic module determine the thrust required in each unit to balance specified wind, wave, and current conditions. The following Table 6.7 illustrated that maximum thrust and power required by each unit for all environmental directions considered.

Table 6.7 Maximum thrust [tones] and max. Power required results by AVEVA

Thruster Number	Maximum Thrust [tones]	Maximum Power [kW]
1	77,04	4787,45
2	77,04	4787,45
3	81,12	5040,78
4	90,72	5637,49
5	90,72	5637,48
6	94,65	5881,53

Least & most effective thrusters and least & most effective pairs of thrusters can be listed as follows based on AVEVA dynamic position estimation,

- Least important thruster is thruster number 3
- Most important thruster is thruster number 6
- Least important pair are thruster numbers 1 and 4
- Most important pair are thruster numbers 4 and 6

As a result, the estimated power for the transit operation was then compared with the required thrust force to provide the dynamic position of the drillship. The selected 6000 kW thrusters are sufficient for this operation. The estimated values and all information about dynamic position estimation can be found in Appendix 7A, 7B and 7C.

7. GENERAL ARRANGEMENT

Overall the general arrangement for drillships is defined as a plan that shows in detail the living spaces of the crew and all equipment that will provide the operational requirements of a ship. The process followed when designing a general layout plan for a drill ship is like other ship types. However, this plan varies according to the own requirements that need to be fulfilled. For example, the accommodation is located in the fore part of the ship. Around the drill hole located in the middle of the ship, there is equipment to perform the drilling.

The design of the general arrangement plan for drillship in the preliminary design phase includes the following details [Papanikolaou, 2009].

- Identification of the main space of the ship according to their main purposes. (Hold spaces, engine room, superstructure, ballast and fuel tanks, watertight bulkheads.)
- The design of the plan according to complex operation flow.
- Easy and safe access to operation areas and equipment.

The ship hull was subdivided according to recommendations established by ABS rules for Building and Classing Steel Vessels Part 3, Chapter 2, Section 9. Also, the double bottom was dimensioned according to ABS rules Part 3, Chapter2, Section 3. The numbers of transverse watertight bulkhead provided were 7. In addition, frame spacing provided 700 mm based on existing ship that mentioned in **Table 2.2**. The general arrangement plan of the designed ship is explained with the following headings and AutoCAD drawing can be found in Appendix 3.

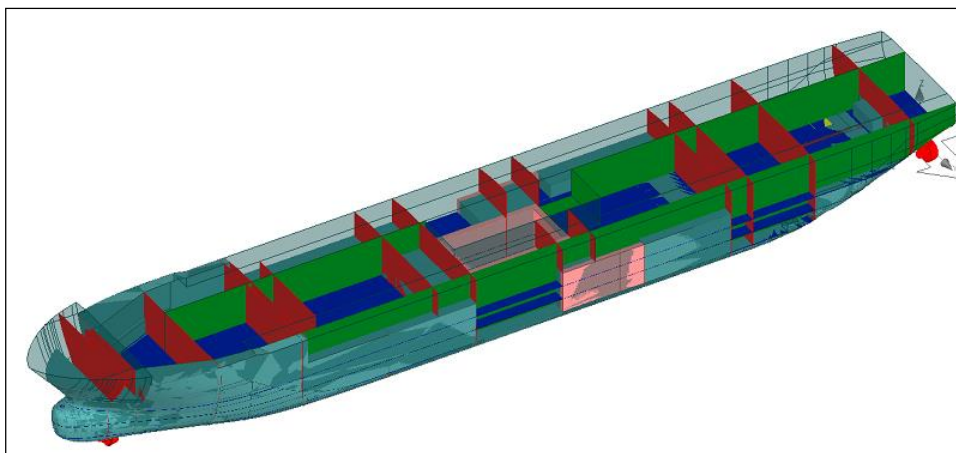


Figure 7.1 Initial locations of Transverse and Longitudinal bulkheads by AVEVA

The drillship general arrangement plan is divided into three independent areas, machinery spaces, accommodation and drilling as detailed in following Table 7.1.

Table 7.1 Subdivided independent areas

Machinery Spaces	Accommodation	Drilling
Engine rooms	Cabins	Cement unit
Propulsion rooms	Toilets	ROV modules
Thruster rooms	Galley	Subsea facilities
Thruster trunks	Mess room	Heavy tool store
Mud pump room	Recreation room	Drilling switchboard
Mud pits	Changing rooms	Drill fluid storage Tanks
Fluids laboratory	Pantry	Driller's cabin
Pump rooms	Hospital	Pipe Racks and Storage Racks
Workshops	Gymnasium	BOP Control Area
Stores	Laundry	X-mas Tree
Switchboard rooms	Offices	Driller control Room
Sack storage & mixing area	Wheelhouse and DP control room	Mud Lab
Passageways	Radio room	Derrick
Engine control room	Server room	Mud storage Tanks
Silo rooms	Battery room	Substructure
Emergency generator room		

7.1. Moonpool Area

As mentioned before, the moonpool used on designed ship is ‘‘Moonpool with aft recess’’. The moonpool (28,00 m x 12,60 m) was integrated into the hull between frames 123 and 177, extending from baseline level upwards to the Main deck. The Moonpool is shown in **Figure 2.3** and its characteristics are shown in **Table 2.3**. Around the moonpool area, two important equipment should be underlined, firstly, the Blow-out Preventer (BOP) equipment on the forward of moonpool and on the aft recess of moonpool, there is a Christmas tree to control the flow of oil out of the well. The moonpool has been surrounded by cofferdams (1400 mm in width) to reduce the effect of hydrodynamic forces and to protecting MDO tanks in case.

7.2. Drillfloor and Drilling Materials Storage

The designed drillship has the Drillfloor capability to perform drilling operations. The Drillfloor is located 32,8 m from baseline, above the main deck in order to give adequate clearance for the subsea and other equipment. The components and main equipment which are related with drill floor can be seen as a listed in Table 7.1, respectively.

7.3. Main Deck and Forecastle Deck

The main deck has camber, forward of frame 260, of 700 mm extending from the vessel's side and 6,0 m off CL (PS and SB). The forecastle height in general arrangement was designed reinforced in way of windlasses, mooring winches, mooring fittings, and other concentrated loads where necessary accordance to Load Line IMO Res. 143 (77) Forecastle and Freeboard. Forecastle deck has camber, forward of frame 312, of 700 mm extending from the vessel's side and approximately 6,0 m off CL (PS and SB). The Forecastle deck is located forward of frame 328 at a height of 23,0 m above base.

7.4. Accommodations

An accommodation block for a total complement of around two hundred persons shall be arranged forward, on Main deck. The accommodation shall comply with the rules and regulations of all applicable authorities including IMO MODU Code and SOLAS regulations with regard to fire resistance and sanitation and in accordance with the requirements of ABS. The overall vertical extent of the accommodation block above Main deck, is subdivided into the following decks:

➤ Main deck	19,70 m
➤ A-deck	22,90 m
➤ B-deck	26,20 m
➤ C-deck	29,50 m
➤ Drilling Floor D-deck	32,80 m
➤ Navigation Bridge deck	37,10 m
➤ Top deck	40,40 m

8. TANK ARRANGEMENT AND CAPACITIES

The primary factors that directly affect the tank plan are the locations of the moonpool and the drilling derrick. The location of the derrick is determined according to ships' longitudinal center of the gravity to limit the motions of the derrick during drilling operations. The tanks of drillship were positioned below the main deck and with big enough size so that the minimum requirements of fresh water capacities and fuel for the operating range were surpassed (See the calculations of fuel consumption in section 5.5.2.2). Exceptionally, fresh water tanks are limited to main deck level.

Three services and two settling marine diesel oil (MDO) tanks were positioned directly to the engine room, which located aft part of the ship. In addition, rest of MDO tanks for storage purpose were located around the moonpool and their positions can be seen in Figure 8.1 with magenta color. The MDO tanks were subdivided according to MARPOL regulations for the prevention of pollution by oil, Chapter 3, Regulation 12A. The calculation was made by following equations and Figure 8.1 shows that oil fuel tank boundary lines, respectively.

The amount of ballast capacity was presented in Table 8.1, which totaled more than 20,000 tons. There is a total of 31 water ballast tanks along the ship. All of the tanks associated with the drilling operations were situated around the moonpool. The Mud tanks, Brine tanks, Drilling water tanks and Base oil tanks were kept close to the drilling area. With this location, less piping and equipment are needed. This arrangement provides, therefore, many possibilities for equilibrium or stability corrections. Some of the watertight surfaces that separate the tanks are not part of the transversal bulkheads. This means that they do not provide structural resistance; they are present solely for isolating the tanks. Table 8.1 created below summarizes the capacity of the tanks, locations and their densities present in the ship. All details such as LCG, TCG, VCG and weight of each tank are located in ship shown in Appendix 6.

Table 8.1 The total capacities plan of Tanks

Idendity	Density [t/m ³]	Volume [m ³]	Weight [t]	LCG [m]	TCG [m]	VCG [m]
Ballast	1,025	21179,0	21179	118,313	0	3,64
Fresh water	1,00	1969,08	1969,08	199,988	0	12,44
Marine Diesel Oil	0,89	4317,38	3842,47	109,428	0	7,51
Lube Oil	0,90	235,30	211,77	10,150	0	10,85
Base Oil	0,80	524,38	419,50	105,700	12,25	7,35
Drill Water	1,00	3163,56	3163,56	108,500	0	7,35
Brine	1,92	1230,28	2362,14	130,200	0	7,35
Mud	2,16	3959,76	8553,08	75,839	0	5,60
Miscellaneous	2,00	607,21	1214,42	85,050	4,73	6,74
Dirty Oil	1,00	302,54	302,54	21,000	0	7,35
Total	-	37488,49	43217,56	110,501	0,252	5,432

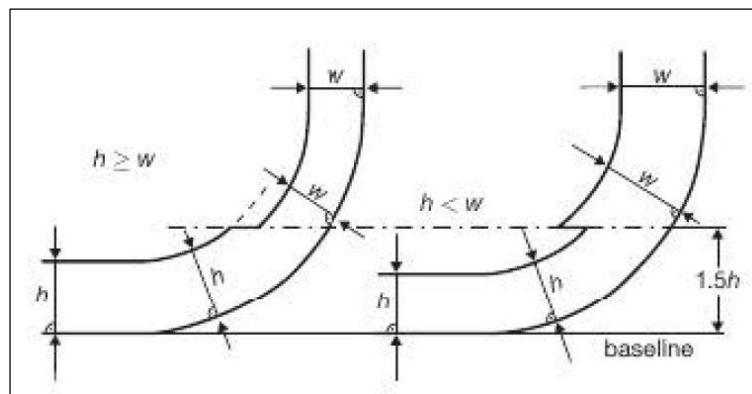


Figure 8.1 Oil fuel tank boundary lines Available from
http://www.marpoltraining.com/MMSKOREAN/MARPOL/Annex_I/

For ships having an aggregate oil capacity between 600 - 5000 m³, oil fuel tanks should be located according to the rule. The minimum value of w and h are 1 m and 2 m, respectively. Therefore, the taken value for drawing w is 1,3 m, and h is 2,1 m. Where C is the total oil capacity.

$$h = \frac{B}{20} = \frac{36 \text{ m}}{20} = 1,8 \text{ m}$$

$$w = 0,5 + \frac{C}{20000} (\text{m}) = 0,5 + \frac{4317,38 \text{ m}^3}{20000 \text{ m}^3} = 0,7159 \text{ or } 2,0 \text{ m (which is the lesser)}$$

A list of content types of the tanks that were used can be seen in Table 8.1 below. In addition, the table illustrated that all the tank types and their attributes, on the ship, are represented by different colors. Figure 8.1 shown below shows the layout of the tanks in the designed drillship. During the detailing of the tank layout, the information of the previous ship was used because of time constraints.

Table 8.2 Content types of Tanks

Identity	Type	Density [t/m ³]	Colour	
Ballast (WB)	Liquid	1,025		Green
Fresh water (FW)	Liquid	1,00		Dark Blue
Marine Diesel Oil (MDO)	Liquid	0,89		Magenta
Lube Oil (LO)	Liquid	0,90		Olive
Base Oil (BO)	Liquid	0,80		Cyan
Drill Water (DW)	Liquid	1,00		Blue
Brine (BO)	Liquid	1,92		Gray
Mud (MUD)	Liquid	2,16		Yellow
Miscellaneous (MIS)	Liquid	2,00		Dark Green
Void	-	-		Turquoise
Dirty Oil (DO)	Liquid	1,00		White

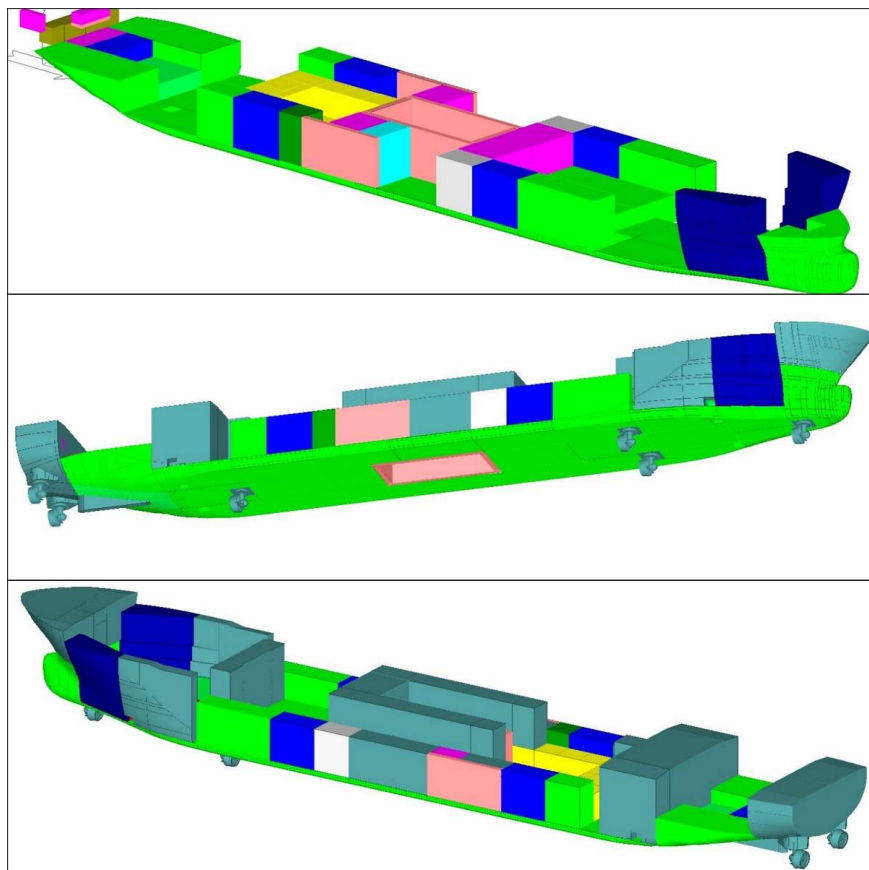


Figure 8.2 Colored Tank Arrangements for designed ship by AVEVA

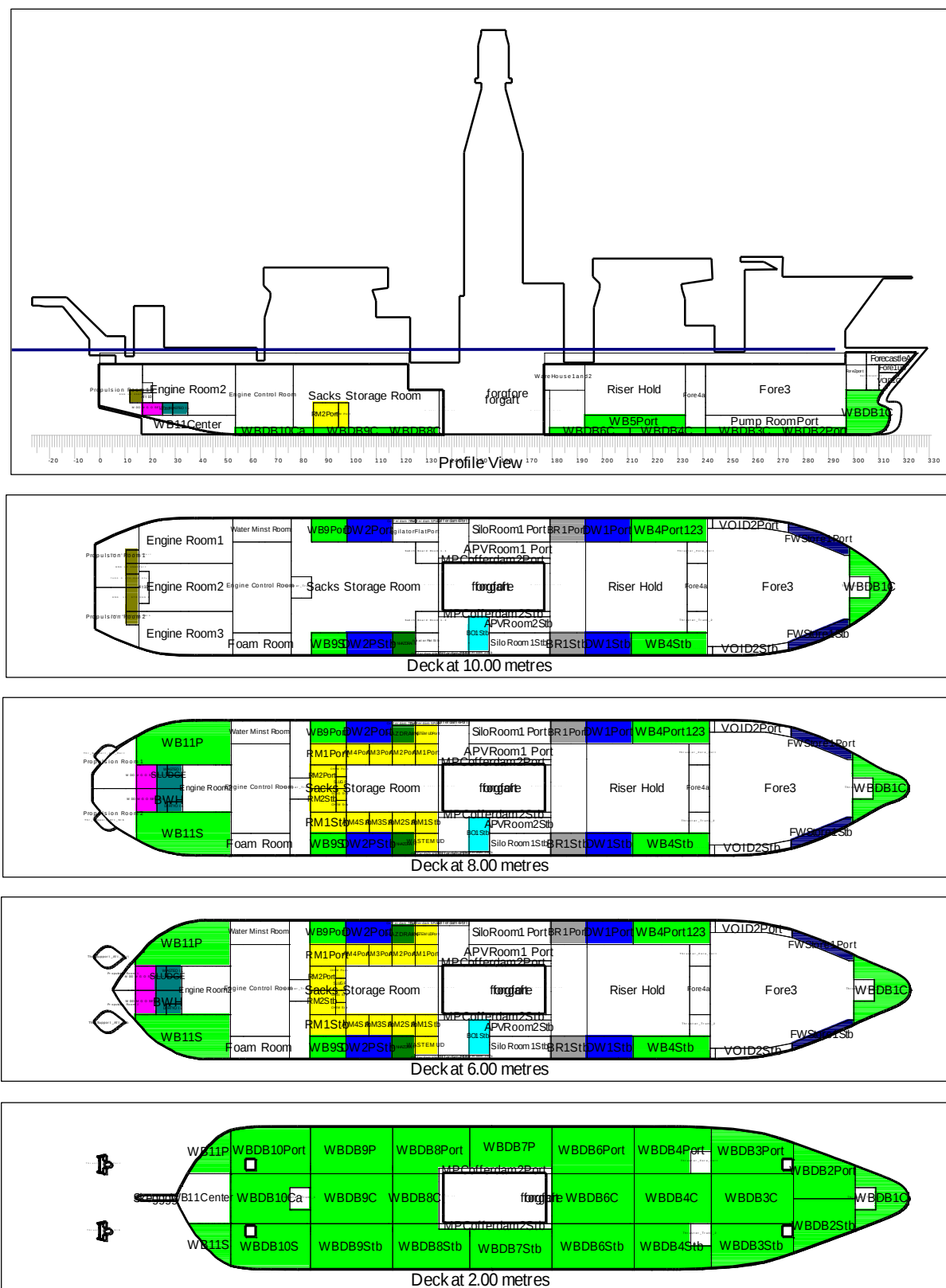


Figure 8.3 Configuration for Water Ballast, Fresh Water, and Marine oil Tanks

The above figures illustrated perceptibly the tanks arrangement with the colors used in Figure 8.2 is taken as a basis.

9. LIGHTSHIP WEIGHT ESTIMATION

The lightship weight is the actual weight of a ship when ready for service but empty of cargo. This includes the weight of scantling, painting, furniture, all machinery, propulsion, mooring, piping, and electrical systems. In order to calculate LCG, TCG, and VCG, the 3D modeling software AVEVA Hydrostatics, and Hydrodynamic module were used.

9.1. Definition of Weight Composition

The weight parameters used by the company throughout the lightship calculation are shown in Table 9.1 below.

Table 9.1 Lightship components followed by the company

Deadweight (DWT)	Loads		Weight & Load Budget
	Future Growth Margin		
	Design Margin		
Lightship Weight (LSW)	Contingency		Gross Weight
	Allowance	Constant Fluids	Net Weight
	Steel Weight	Item Weight	Dry Weight

The below section 9.2 is explained that equipment and component weights with summary tables which break down weights by 13 main weight groups. The ship hull (below main deck), X-Mas Tree, Substructure, Burner Booms, Accommodation, Derrick, Mud Module, Reel Module, Well Test Area, Superstructures aft and fore, Helideck, Funnel was listed along with each item's weight and LCG, TCG, and VCG from the origin of the reference system (Frame 0, centerline, baseline).

9.2. Lightweight Groups

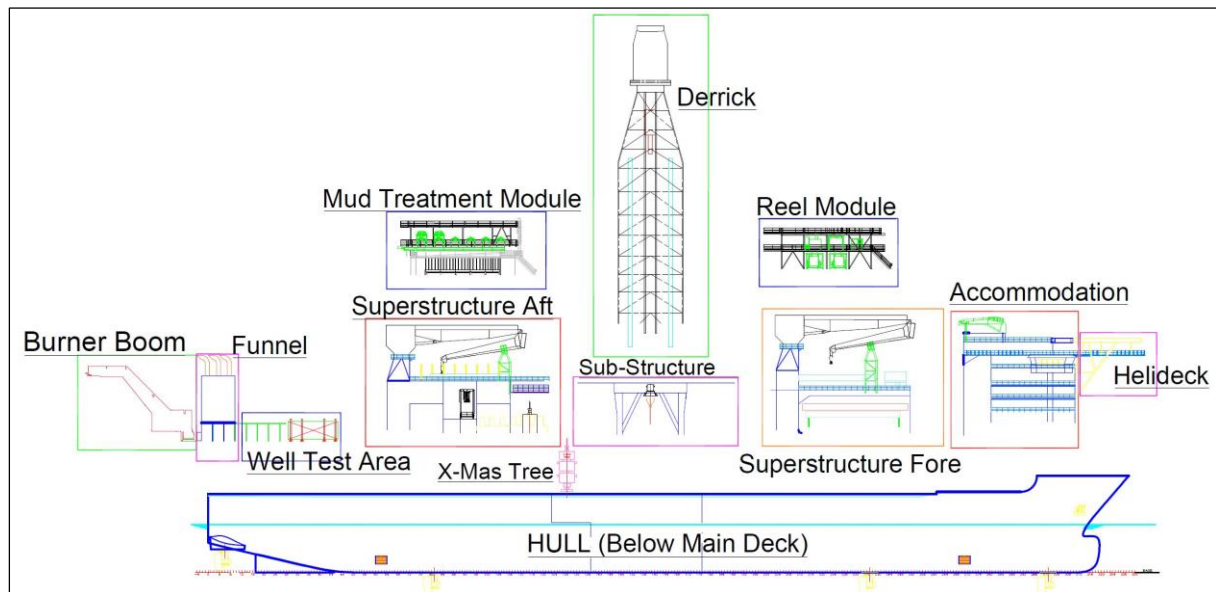


Figure 9.1 Estimated Lightweight Groups

Table 9.2 Estimated Lightweight Summary

Description	Net Weight [t]	Contingency %	Gross Weight [t]	LCG [m] from Fr0	TCG [m] from CL +PS	VCG [m] from BL	Frame Aft-Fore	xA [m]	xF [m]
Hull (Below Main D.)	16000	20%	19200	108,6	0,1	12,0	0-329	0,0	230,3
X-Mas Tree	80	30%	104	91,0	0,6	22,0	128-132	89,6	92,4
Substructure (Incl.Drillfloor)	2000	30%	2600	112,7	0,7	33,0	135-187	94,5	130,9
Burner Booms	90	30%	117	13,3	0,0	28,0	1-137	0,7	25,9
Accomodaion	1600	20%	1920	197,4	0,1	32,0	267-297	186,9	207,9
Derrick	800	30%	1040	108,5	-1,1	67,0	144-166	100,8	116,2
Mud Module (Starboard)	200	30%	260	77,0	-12,8	25,0	68-152	47,6	106,4
Reel Module (Portside)	150	30%	195	142,0	11,4	26,0	14-220	130,0	154,0
Well Test Area	130	30%	169	24,5	-0,1	24,0	26-44	18,2	30,8
Superstructure (Fore)	1100	20%	1320	157,5	0,2	31,0	197-253	137,9	177,1
Superstructure (Aft)	1800	20%	2160	61,6	0,6	31,0	66-110	46,2	77,0
Helideck	500	20%	600	217,0	0,0	44,0	296-324	207,2	226,8
Funnel	200	20%	240	14,0	0,3	28,0	14-26	9,8	18,2
Total			29925	113,86	-0,11	20,37			

9.3. Lightweight Distribution

AVEVA Hydrostatics and Hydrodynamics module were used to perform the lightship estimation and obtain the lightweight distribution, which is shown below in Figure 9.2.

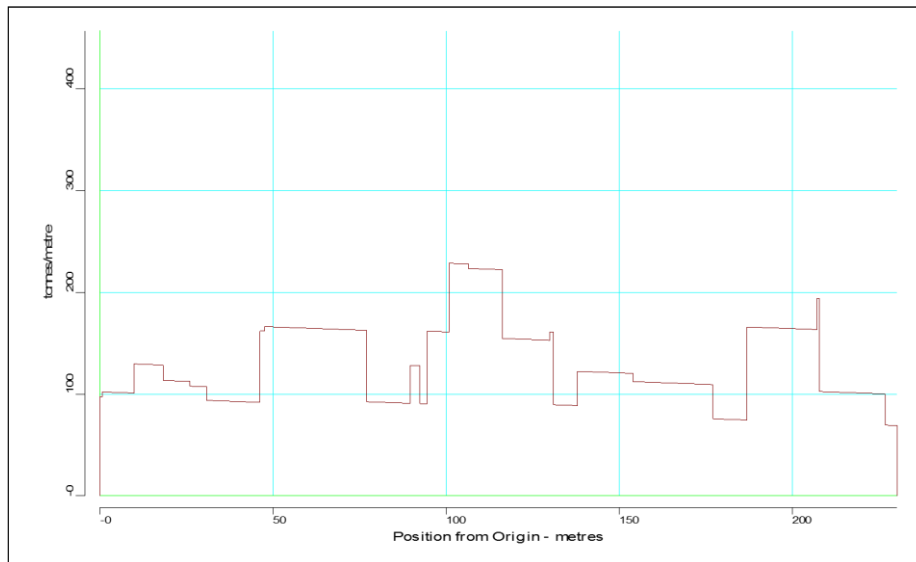


Figure 9.2 Estimated Lightweight Distribution

9.4. Deadweight Estimation

At the summer draught of 12, 0 m the deadweight is 503310, 04 tons by using following items.

Table 9.3 Deadweight Items

1 -	BOP Stack	11 -	Heavy Weight Drill Pipes
2 -	Casing 10"	12 -	Pipe Joints
3 -	Casing 14"	13 -	Provisions
4 -	Casing 20"	14 -	Riser system
5 -	Casing 30"	15 -	Riser yokes
6 -	Crew & Effects	16 -	Sacks
7 -	Drill Collars	17 -	Static Hook Load
8 -	Drill Pipes 5 7/8"	18 -	Third Part Equipment
9 -	Drill Pipes 5"	19 -	Tools and Equipment
10 -	Drill Pipes 7 & 6 5/8"		

10.LOADING CONDITIONS

For the design of the vessel, a limited of loading conditions have been considered representing the boundaries of the design envelope with respect to weight, the center of gravity and wind area. The loading conditions can be divided into following four categories;

- Harbor conditions
- Transit conditions
- Operating conditions
- In-field conditions

The loading conditions will be presented in a qualitative way which describes with schematic illustration. Only the loading conditions required by the classification have been considered for this study. For each individual loading conditions, the deadweight items loaded can be seen as an overview end of the chapter.

10.1.1. Harbour Conditions

10.1.1.1. Lightship

This loading condition represents the empty outfitted hull of the vessel (lightship weight) and including margins. This is a non-sailing condition.

10.1.1.2. Docking

This loading condition represents the vessel during dry docking as shown in Figure 10.1, the BOP is on board, riser joints and third part equipment have been offloaded. No casing and cuttings are stored on board. All drill pipe and drill collars are stored in tubular racks on superstructure deck aft.

- Drilling consumable levels (Cement, sacks, drill water, brine, and base oil) and all vessel consumable levels (Provisions, potable water, marine diesel oil and lubrication oil) are defined as %5.
- Water ballast was taken in to compensate for even heel and trim.

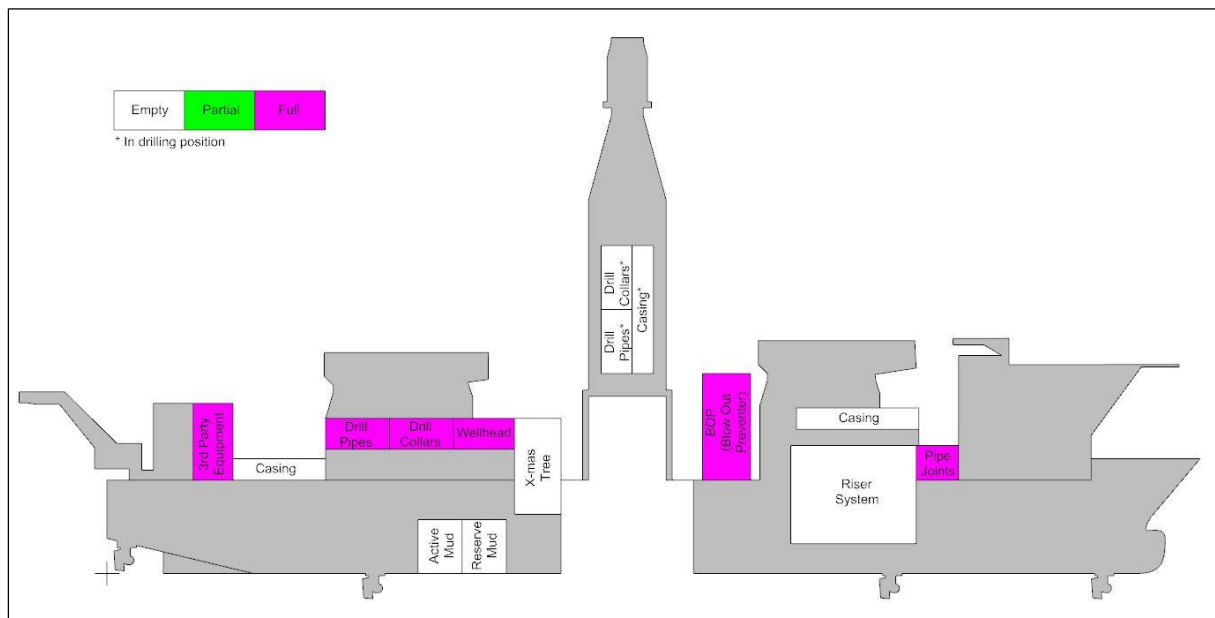


Figure 10.1 Docking Conditions

10.1.2. Transit Conditions

10.1.2.1. Transit with Casing – %100

This loading condition represents the vessel in departure condition as shown in Figure 10.2 with 100% consumables. There are no drilling loads (Static hook load, no riser tensioning load and setback load). All stored riser joints and the BOP are located forward of the moonpool. All drilling tubular (drill pipe, drill collars, and casing) are stored in tubular racks on superstructure deck forward, superstructure deck aft and main deck.

- Drilling consumable and vessel consumable levels are maximum. Active and reserve mud tanks are empty also, no cuttings and no waste are stored on board.
- Water ballast was taken in to compensate for even heel and trim.

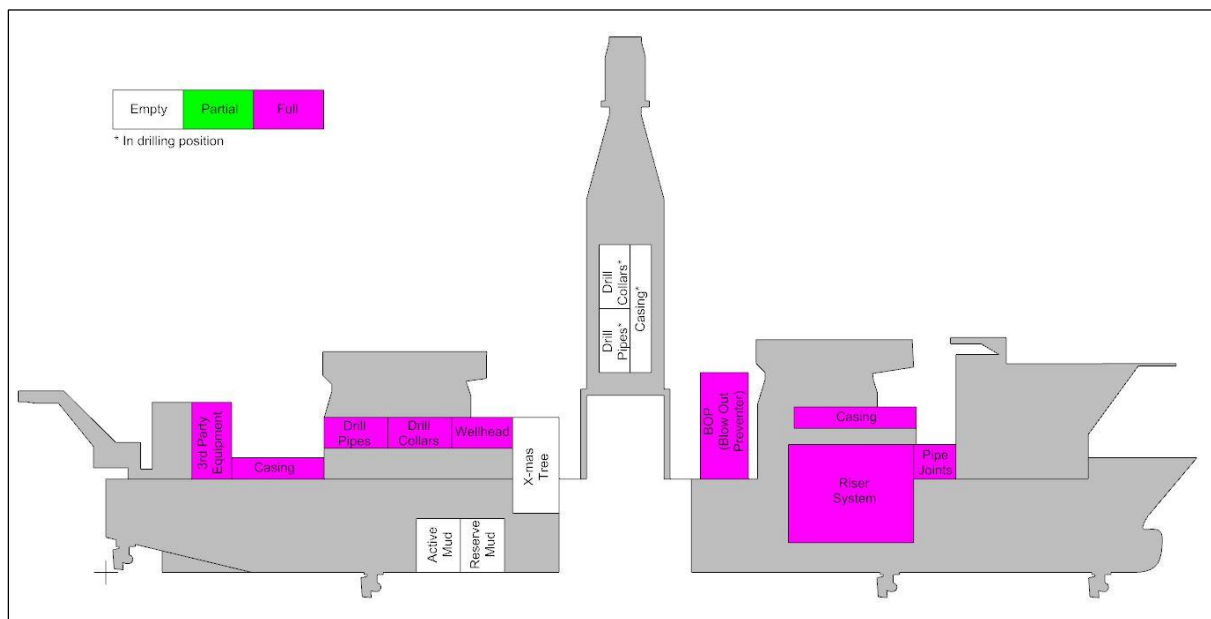


Figure 10.2 Transit with Casing Conditions

10.1.2.2. Transit with Casing – %10

This loading condition represents the vessel in arrival condition with 10% consumables. This condition is identical to transit with casing 100% except for following;

- 10% drilling consumables are on board.
- 10% vessel consumables are on board.
- 90% dirty oil and waste are produced.

10.1.2.3. Transit without Casing – %10

This loading condition represents the vessel in arrival condition with 10% consumables. This condition is identical to transit with casing 10% except for following,

- No casing is stored on board.

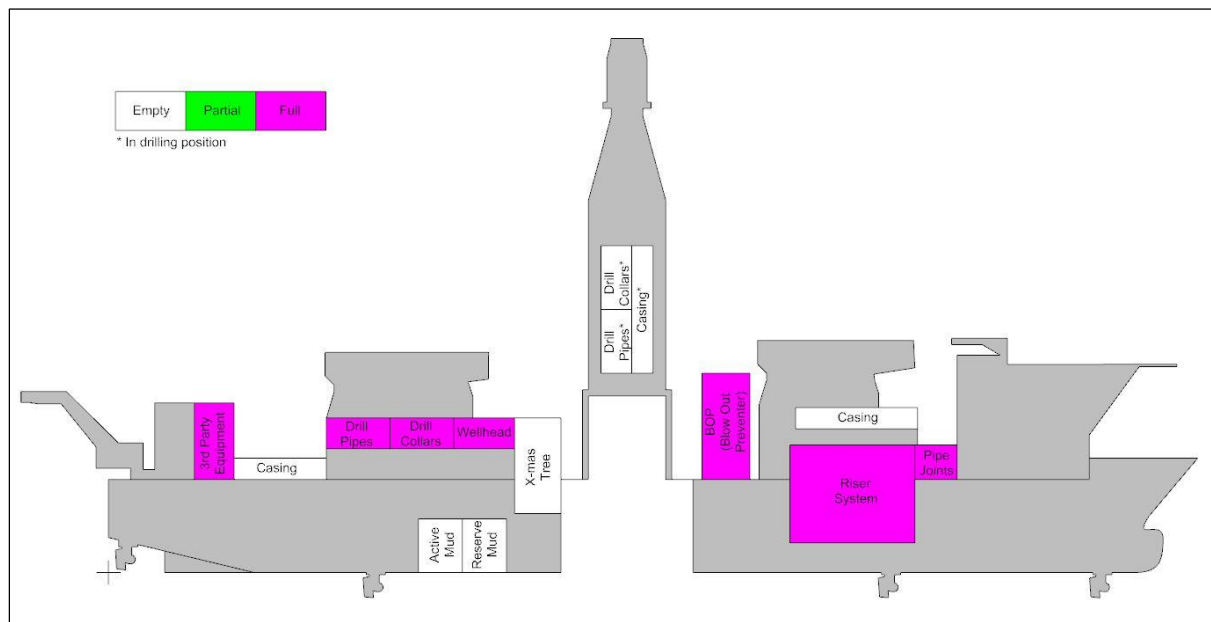


Figure 10.3 Transit without Casing Conditions

10.1.3. Operating Conditions

10.1.3.1. Drilling – %100

This loading condition represents the vessel during drilling as shown in Figure 10.4 with %100 consumables. The drill string has been built up and drilling operations have been started after connecting to the well. It is assumed that a hole section of the well is being drilled.

- The most drill pipe and drill collars are being used for the drilling operation. The large part of the casing string is transferred to the setback and is prepared for running. In addition, the third part of the equipment is on board as required.
- All riser joints are removed from the riser hold. The remainder of the casing is stored in tubular racks on superstructure deck forward and main deck.
- Drilling consumable levels are at maximum. Mud is circulated that means, minimum volume residing in the active mud tanks is equal to the volume of the cased hole plus the volume of the open hole, including excess. Active and reserve mud tanks are filled up to 30% and 50%, respectively. In addition, cuttings are stored on board.
- Drilling consumable and vessel consumable levels are maximum. Dirty water and dirty oil tanks are empty also no waste on board.
- Water ballast was taken in to compensate for even heel and trim.

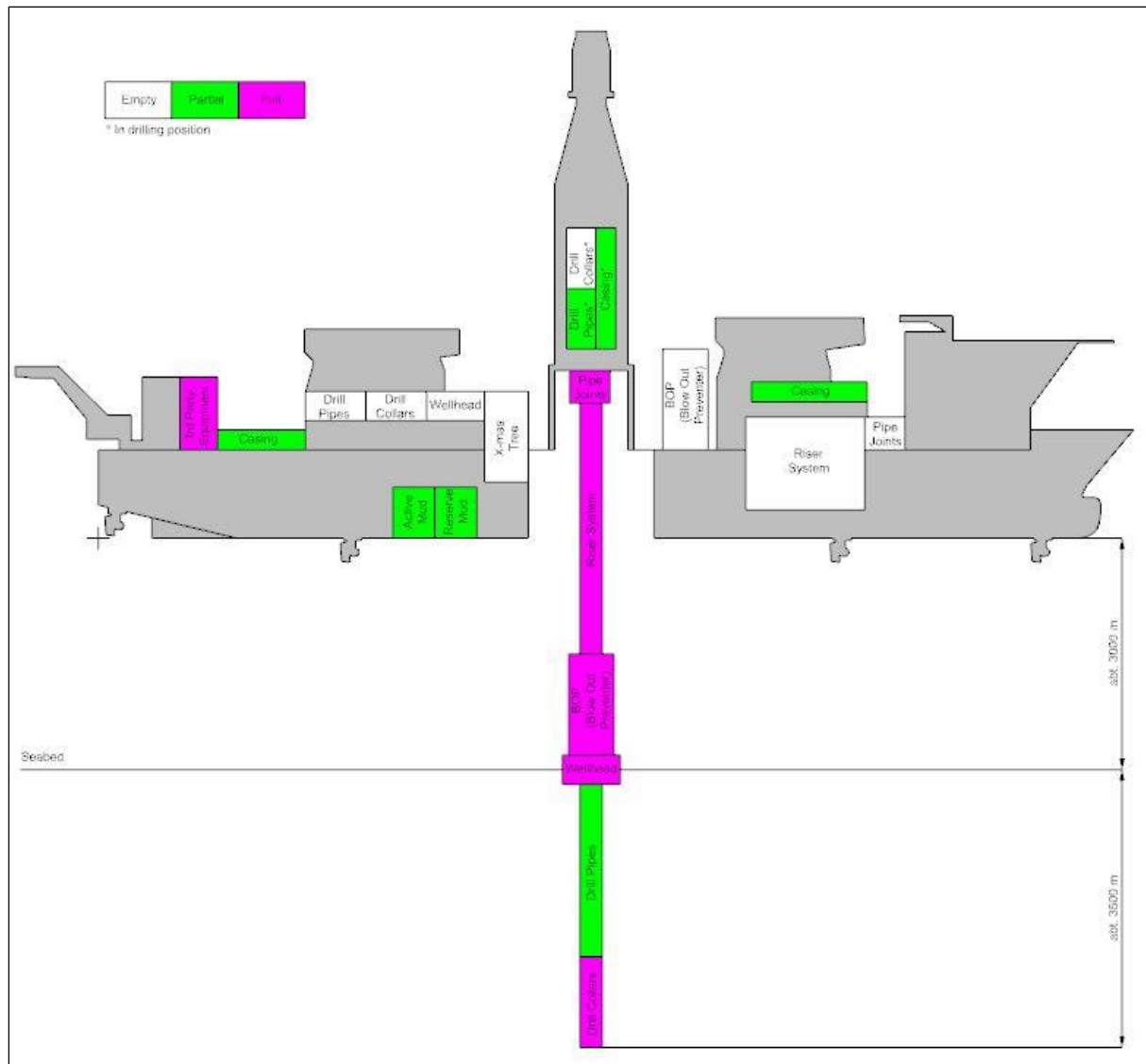


Figure 10.4 Drilling Conditions

10.1.3.2. Drilling – %10

This loading condition represents the vessel during the drilling operation with 10% consumables. This condition is identical to drilling 100% except for following;

- 10% drilling consumables are on board.
- 10% vessel consumables are on board.
- 90% dirty oil and waste are produced.

10.1.3.3. Start Running Casing – %100

This loading condition represents the vessel ready for running casing as shown in Figure 10.5 with 100% consumables. The drill string is tripped out and racked back in the setback when having drilled a hole section of the well. The larger part of the casing string is stored in the setback to being ready for running.

- All riser joints are removed from the riser hold. The remainder of the casing is stored in tubular racks on superstructure deck forward and main deck. Third part equipment and cuttings are stored on board as required. Also, there is no hook load.
- Drilling consumable levels are at maximum. Active and reserve mud tanks are filled up to 30% and 50%, respectively.
- Drilling consumable and vessel consumable levels are maximum. Dirty water and dirty oil tanks are empty also no waste on board.
- Water ballast was taken in to compensate for even heel and trim.

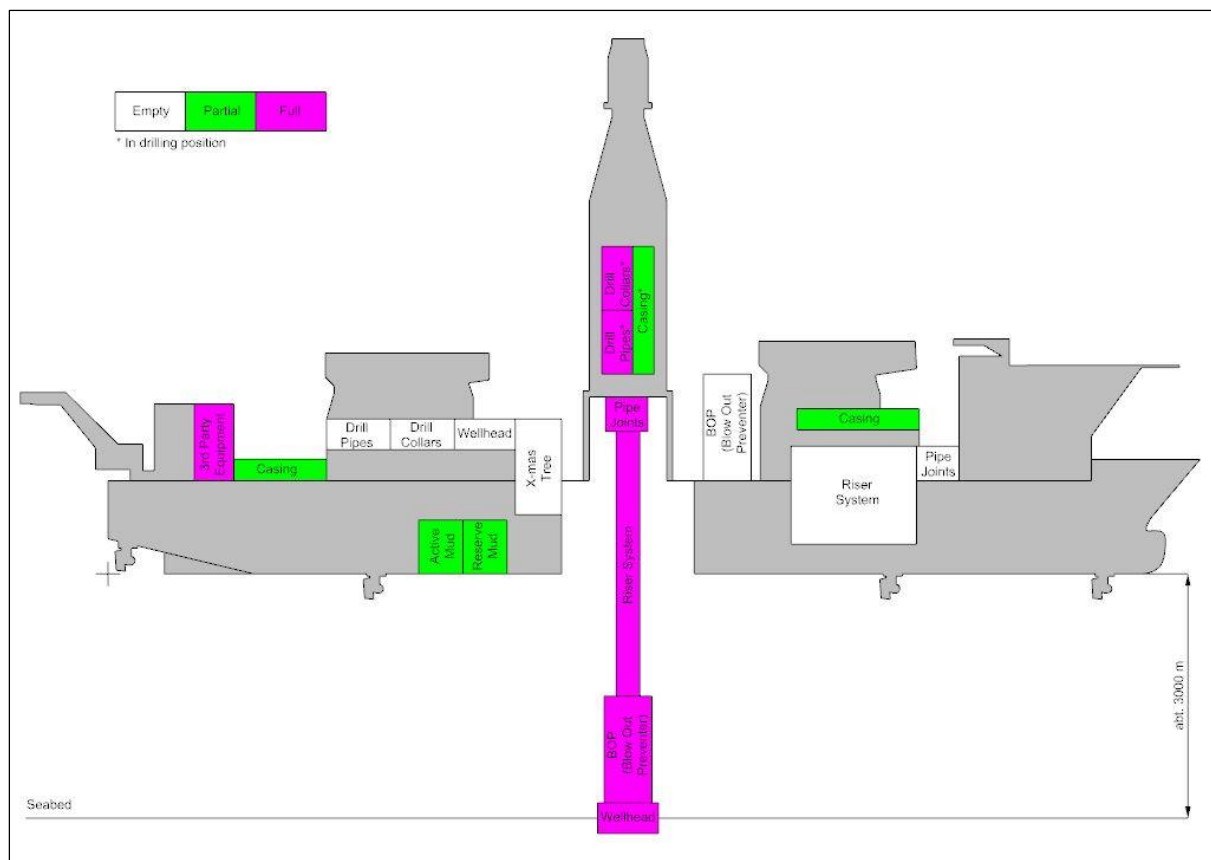


Figure 10.5 Starting Running Casing Conditions

10.1.3.4. Start Running Casing – %10

This loading condition represents the vessel ready for running casing with 10% consumables. This condition is identical to start running casing 100% except for following;

- 10% drilling consumables are on board.
- 10% vessel consumables are on board.
- 90% dirty oil and waste are produced.

10.1.3.5. Running Casing – %100

This loading condition represents the vessel during running casing as shown in Figure 10.6 with 100% consumables. The heaviest casing string is being lowered into the well using the landing string. Maximum riser tensioning load assumed. The setback is filled with drill pipe and drill collars.

- All riser joints are removed from the riser hold. The remainder of the casing is stored in tubular racks on superstructure deck forward and main deck. Third part equipment and cuttings are stored on board as required. Also, there is no hook load.
- Drilling consumable levels are at maximum. Active and reserve mud tanks are filled up to 30% and 50%, respectively.
- Drilling consumable and vessel consumable levels are maximum. Dirty water and dirty oil tanks are empty also no waste on board.
- Water ballast was taken in to compensate for even heel and trim.

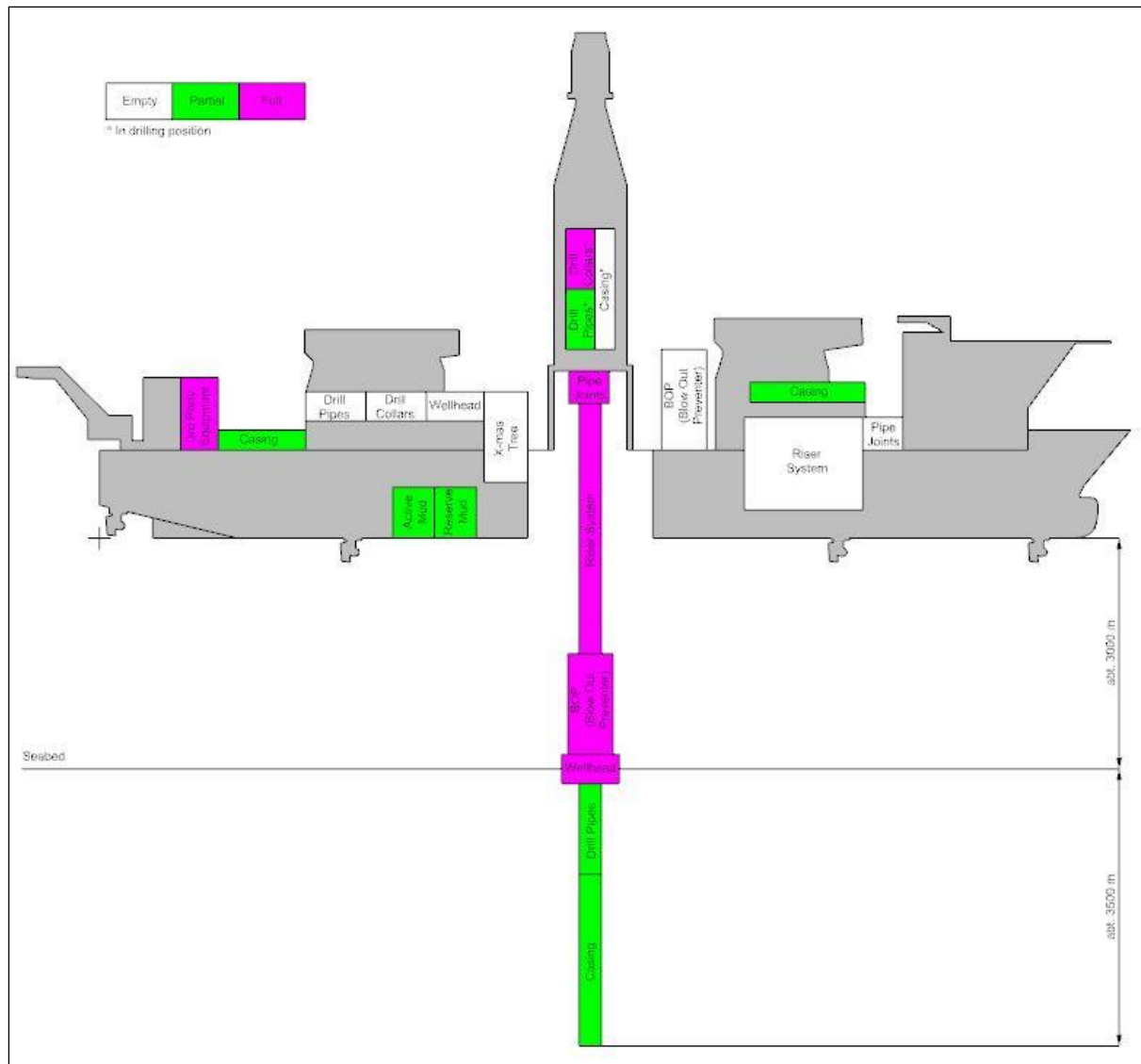


Figure 10.6 Running Casing Conditions

10.1.3.6. Running Casing – %10

This loading condition represents the vessel during running casing with 10% consumables. This condition is identical to running casing 100% except for following;

- 10% drilling consumables are on board.
- 10% vessel consumables are on board.
- 90% dirty oil and waste are produced.

10.1.4. In-Field Conditions

10.1.4.1. Stand by – %100

This loading condition represents the vessel in stand-by condition as shown in Figure 10.7 with 100% consumables. Drilling operations are suspended and preparations are made for a disconnect. The vessel is connected to the well, however, the drill string has already been tripped out. Maximum riser tensioning load assumed. The setback is filled with drill pipe and drill collars.

- All riser joints are removed from the riser hold. The remainder of the casing is stored in tubular racks on superstructure deck forward and main deck. Third part equipment and cuttings are stored on board as required. In addition, there is no hook load.
- Drilling consumable and vessel consumable levels are maximum. Mud circulation is stopped also, active and reserve mud tanks are filled up 50%. Dirty water and dirty oil tanks are empty also, no waste on board.
- Water ballast was taken in to compensate for even heel and trim.

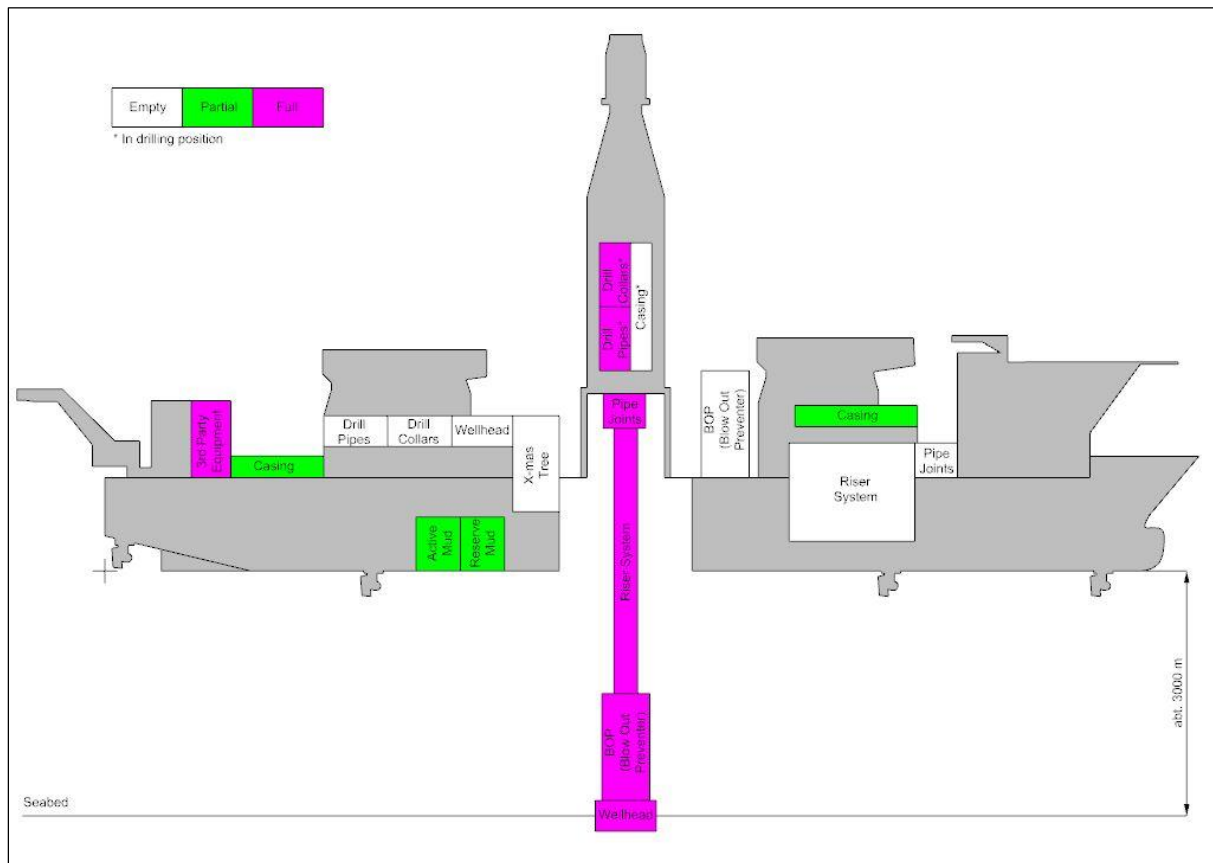


Figure 10.7 Stand-by Conditions

10.1.4.2. Stand by – %10

This loading condition represents the vessel in stand-by condition with 10% consumables. This condition is identical to stand-by 100% except for following;

- 10% drilling consumables are on board.
- 10% vessel consumables are on board.
- 90% dirty oil and waste are produced.

10.1.4.3. In-Field Survival – %100

The loading condition represents the vessel during in-field survival as shown in Figure 10.8 with 100% consumables. The vessel stays dynamically positioned in the field to ride out a storm, after planned disconnected. The setback is partially filled with drill pipe. There is no static hook load and no riser tensioning load. The BOP is sea fastened in the transporter forward in the moonpool. Third part equipment and cuttings are stored on board as required.

- All riser joints are stored in the riser hold. Drill tubular (drill pipe, drill collars, and casing) are stored in tubular racks on superstructure deck forward, superstructure deck aft and main deck.
- Drilling consumable and vessel consumable levels are maximum. The active mud tanks are filled up to their maximum capacity and the reserve mud tanks are filled up to 50%. Dirty water and dirty oil tanks are empty also, no waste on board.
- Water ballast was taken in to compensate for even heel and trim.

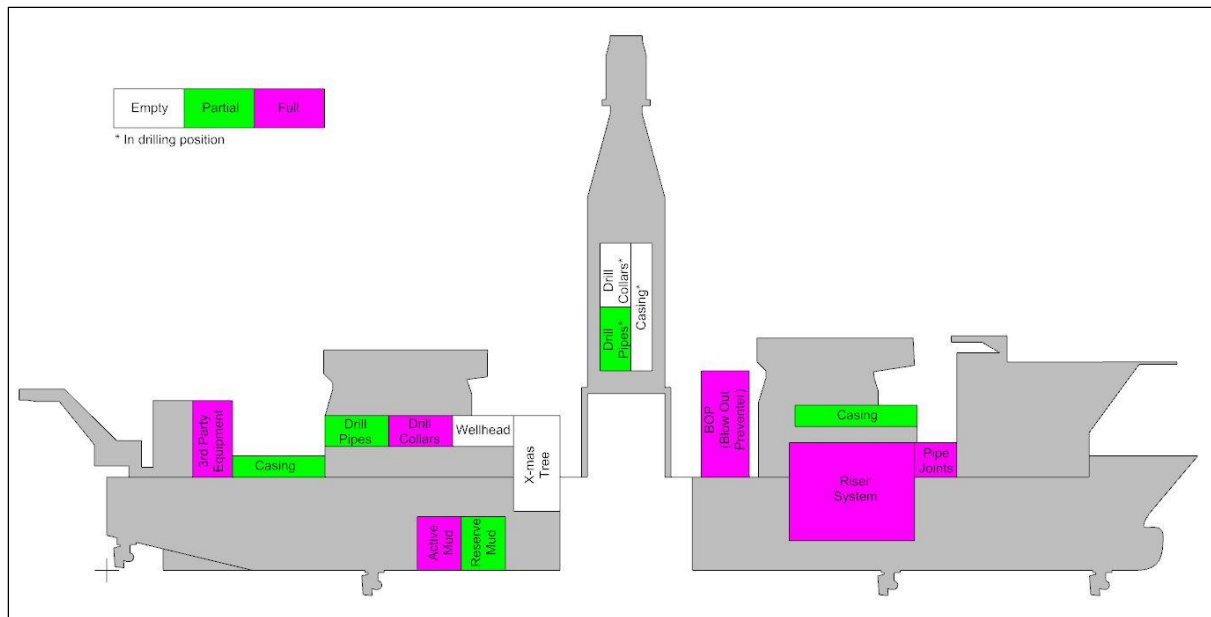


Figure 10.8 In-field Survival Conditions

10.1.4.4. In-Field Survival – %10

This loading condition represents the vessel during in-field survival with 10% consumables.

This condition is identical to in-field survival 100% except for following;

- 10% drilling consumables are on board.
- 10% vessel consumables are on board.
- 90% dirty oil and waste are produced.

11.INTACT STABILITY & SFBM PRELIMINARY EVALUATION

11.1.General

The loading conditions defined in Chapter can be analyzed with regard to Intact stability criteria as required by the 2008 Intact Stability Code.

In general commercial ships which have a conventional propulsion arrangement in the aft part (engine & propeller) the maximum acceptable trim is 1,5% of aft perpendicular. This is to ensure propeller immersion. The trim can also be limited by equipment (e.g. Cranes) functionality profile. As for this design, propulsion is by thrusters and at this stage, we have no information of any of the equipment maximum operational profiles, we will limit the trim by 1,5% of aft perpendicular. This means a trim of max 3,3 m or 0,8 degrees.

11.2.Intact Stability Calculation

11.2.1. Criteria

The following criteria were considered during the intact stability calculation. The criteria can be further explained and defined in Appendix 8.

- International Code on Intact Stability 2008.
- Severe wind and rolling criterion (Weather criteria).
- MARPOL 73/78 Annex 1 (2011 Edition) Reg.
- ABS Rules For Building And Classing Steel Vessels 2012 / Part 3 Hull Construction And Equipment.

11.2.2. Wind Profile

The wind profile used for stability calculations was made based on the General Arrangement plan and is presented in Appendix 3.

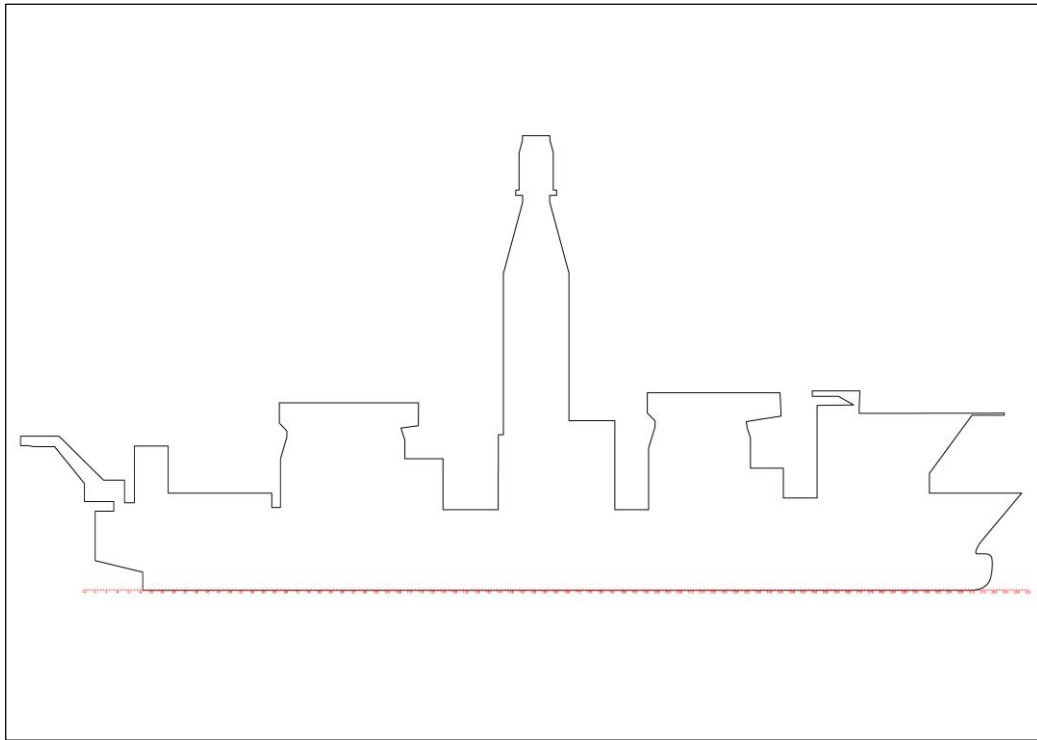


Figure 11.1 Drillship Wind Profile

11.2.3. Watertight and Weathertight Integrity

For stability calculating purposes, the number of deck openings and a deck margin line were specified. These openings can be categorized in the following categories,

- **Unprotected** – Gives no protection to the ingress of water into the connected compartment(s).
- **Weathertight** – Protects connected compartment(s) from unrestricted ingress of water during a brief submergence of the openings. However weathertight openings have to remain above the final waterline in both damaged and intact conditions.
- **Watertight** – Protects any compartment from flooding above or below the final waterline in damaged or intact condition.

All openings considered in the stability calculations are in the most vulnerable locations, such as weathertight vent pipes on the side forward and aft. Special attention will be made to non-watertight doors, ventilation inlet, and outlets. List of unprotected openings used in intact stability calculations is shown in following Table 11.1. In addition, deck margin line is defined as a table in Appendix 9.

Table 11.1 Un-Protected Opening List

Number of Un-Protected Openings	x [m]	y [m]	z [m]	Compartment
0	9,84	-3,07	32,52	EngineRoom3
1	9,84	3,07	32,52	EngineRoom1
2	9,81	-6,11	34,25	EngineRoom2
3	9,81	6,11	34,25	EngineRoom2
4	10,02	-6,38	28,43	EngineRoom1
5	12,85	-6,38	28,43	EngineRoom1
6	10,02	-6,38	30,24	EngineRoom1
7	12,85	-6,38	30,24	EngineRoom1
8	10,02	-6,38	32,4	EngineRoom2
9	12,85	-6,38	32,4	EngineRoom2
10	10,02	6,38	28,43	EngineRoom3
11	12,85	6,38	28,43	EngineRoom3
12	10,02	6,38	30,24	EngineRoom3
13	12,85	6,38	30,24	EngineRoom3
14	10,02	6,38	32,52	EngineRoom2
15	12,85	6,38	32,52	EngineRoom2
16	69,34	9,91	24,34	SacksStorageRoom
17	69,34	-9,91	24,34	SacksStorageRoom
18	119,42	8,3	27,69	APVRoom2Stb
19	120,79	10,4	27,69	APVRoom2Stb
20	119,06	9,53	27,14	WareHouse1and2
21	137,18	-12,8	24,97	APVRoom1 Port
22	143,5	11,2	21,35	RiserHold
23	143,5	-11,2	21,35	RiserHold
24	171,5	11,2	21,35	RiserHold
25	171,5	-11,2	21,35	RiserHold

11.2.4. Free Surface Corrections

The free surface effects were considered according to their filling level for all cargo, ballast and process tanks.

For bunker tanks, the free surface moments were considered such that for each type of liquid at least one transverse pair or a single centerline tank has a free surface and the tank or combination of tanks taken into account should be those where the effect of free surfaces is the greatest.

Provided a tank is completely filled with liquid no movement of the liquid is possible and the effect on the ship's stability is precisely the same as if the tank contained solid material.

As soon as a quantity of liquid is withdrawn, from a tank, the situation changes completely and the stability of the ship is adversely affected by what is called the "free surface effect". This adverse effect on the stability is referred as a "loss in GM" or as a "virtual rise in VCG" and is calculated as follows:

$$\text{Loss in GM (Rise of KG) due to Free Surface Effect} = \frac{\sum \text{Free Surface Moments [m}^4\text{]} \times \text{Density of Liquid in Tanks } [\frac{t}{m^3}]}{\text{Displacement of Vessel [t]}}$$

11.2.5. Cross Curves of Stability

The Cross Curves of Stability are used to determine the length of the righting arm at any angle of inclination for a given displacement. Using the ship's displacement (from the Draft Diagram and Functions of form) a statical stability curve for the ship can be constructed. The process can be summarized as following steps. Each righting arm is plotted at the corresponding angle of inclination on graph paper.

- Determine Ships Displacement and Vertical Centre of Gravity (KG).
- Enter the Cross Curves Table corresponding to the Draught and Net Volume of the Buoyant Hull (Displacement/Water Density).
- Extract the corresponding form stability levers (KN) values for each angle of inclination.
- Determine the righting arm (GZ) for each KN value: $GZ = KN - KG * \sin\theta$

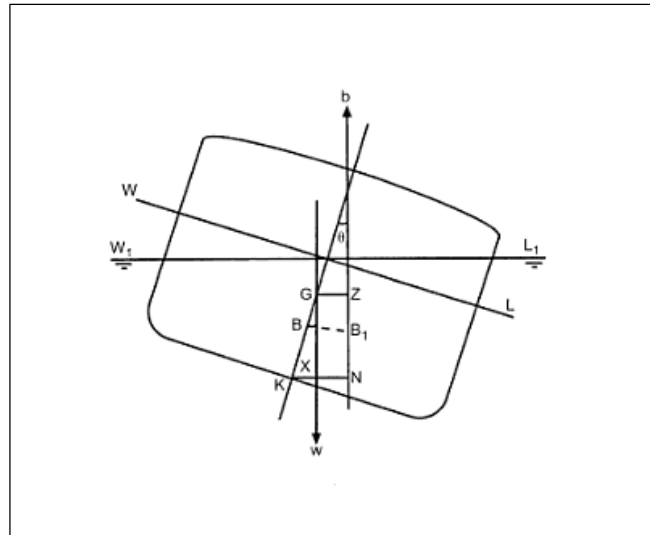


Figure 11.2 GZ-Curve definition for Intact Stability Calculations Available from

<http://www.boatdesign.net/forums/stability/kn-curves-problem-46049.html>

$$GZ = KN - KG * \sin\theta$$

Where,

M= Metacenter

G= Centre of Gravity

K=Keel

θ= Angle of Inclination

GZ= Righting Moment Lever

KN= Stability from Lever

11.2.6. Intact Stability Check Results

The simulated loading conditions are generated for the purpose of classifications. Appendix 12A and Appendix 12B are included all intact stability results. GZ curve, one of the most important parameters of stability calculations, has been added into study as an Appendix 13.

11.3. Shear Force and Bending Moment Envelope

The Shear Force and Bending Moment assessment, based on the same loading conditions, at the concept design stage determines the effective strength limits of the ship based on defined loading conditions, which will serve as input for the Midship section design and the effective Shear Force and Bending Moment envelope will serve as input for the generation of the longitudinal strength limits of the ship (scantling evaluation). The maximum shearing force and bending moment falling on the ship in relation to loading conditions can be easily seen from the list below. Furthermore, the distribution of loading conditions according to each frame is shown in Appendix 14.

Table 11.2 Maximum Shear Force and Bending Moments for all conditions

	Max. Shearing Force [kN]	Max Bending Moment [kNm]	Max. Shearing Force Position [m]	Max Bending Moment Position [m]
Docking	20013.7	920576.7	26.16	86.1
Transit with casing %10	23808.6	1056878.1	38	81.9
Transit with casing %100	21699.33	652750.5	168.7	86.1
Transit without casing %10	-18822.2	660406.4	189.13	163.77
Transit without casing %100	-23036.1	695413	191.1	168.7
Drilling 10%	-33180	1413052.4	188.9	152.35
Drilling 100%	22347.4	485741.8	163.1	179.56
Start Running Casing 10%	-25677.7	919209.6	197.69	172.3
Start Running Casing 100%	-30408.3	972504.9	193.05	175
Running Casing 10%	-27838.6	1032525.1	195.1	134.47
Running Casing 100%	-45325.6	1627980.9	191.1	161.53
Stand by 10%	-3330.2	1322191.5	188.9	157
Stand by 100%	-32990.4	1027598.2	191.73	174.75
In-field Survival 10%	-23229	756026.7	201.55	179.56
In-field Survival 100%	27957.8	756160.4	163.1	180.89

12.CONCLUSIONS

The initial design stage of a ship is of utmost importance as it will define ship characteristics which are very hard (with great costs) or almost impossible to modify after the design process goes into the basic and detail design stages. All calculations and drawing done should be revised and optimized at least by one iteration. Further iterations on the above design spiral are necessary for all the areas of the vessel before arriving at the optimum results.

Using numerical approaches to calculating the resistance of special modern ships can really be a bit far away. It was seen in the CFD runs that the effect of the Moonpool on the total resistance of the ship is at least twenty percent. Moonpool covers a very large part of the total ship's resistance. One of the most important parameters for drilling vessels, estimation of thrust power is in relation to dynamic position. For this reason, calculating the power of the main engine power and thrusts calculated in the preliminary calculations can provide convenience for future design phases.

For drilling vessels, it is very important that the stability calculations are carried out in accordance with all kinds of operation scenarios. The stability issue of the drillship is not only important during the journey, but also has to be, safety, profitable and rigorous throughout the operation.

This thesis covers some major aspects of the initial design process of a drillship but can and for a complete analysis should be followed, at a minimum, by the following calculations:

- Damage stability calculations
- Scantling evaluation
- Midship section design
- Seakeeping evaluation
- Cost evaluation

13. REFERENCES

- [1]@2015 Drillships and Semisubs for Dummies. Retrieved July 12, 2016, from <https://rigzonenews.wordpress.com/2014/10/01/drillships-and-semisubs-for-dummies/>
- [2] Veer, R. V., & Tholen, H. J. (2008). Added Resistance of Moonpools in Calm Water. Volume 6: Nick Newman Symposium on Marine Hydrodynamics; Yoshida and Maeda Special Symposium on Ocean Space Utilization; Special Symposium on Offshore Renewable Energy. doi:10.1115/omae2008-57246
- [3] Hammargren, E., & Tornblom, J. (n.d.). Effect of the Moonpool on the Total Resistance of a Drillship. Retrieved February 2012, from Chalmers University of Technology, Division of Marine Design. Master of Science Thesis
- [4] Watson, D. G. (1998). Practical ship design. Amsterdam: Elsevier.
- [5] Barrass, C. B., Derrett, D. R., & Barrass, B. (2012). Ship stability for masters and mates (7th ed.). Oxford: Butterworth-Heinemann.
- [6] Barrass, C. (2004). Ship Trials. Ship Design and Performance for Masters and Mates, 116-119. doi:10.1016/b978-075066000-6/50014-3
- [7] Holtrop, J., and Mennen, G.G.J., An Approximate Power Prediction Method, International Shipbuilding Progress, Volume 89, 1982.
- [8] J. Holtrop. A statistical re-analysis of resistance and propulsion data. International Shipbuilding Progress, 31(363):272-276, November 1984.
- [9] J. Holtrop, "A Statistical Resistance Prediction Method with a Speed Dependent Form Factor. " 1988.
- [10] Toshifumi Fujiwara, Michio Ueno, and Yoshiho Ikeda. Cruising performance of a large passenger ship in heavy sea. In Proceedings of the Sixteenth (2006) International Oshore and Polar Engineering Conference, pages 304-311. The International Society of Oshore and Polar Engineers, 2006.
- [11] Steddum, Riddle; Herrmann, Robert P. "Thrust Required and Holding Capacity". Houston, USA: Dynamic Positioning Conference, 21 – 22 October, 1997.
- [12] The International Marine Contractors Association, Specification for DP Capability Plots, Report no: IMCA M 140 Rev. 1, June 2000
- [13] Eyres, D. J., & Bruce, G. J. (2012). Ship construction (6th ed.). Oxford: Butterworth-Heinemann.

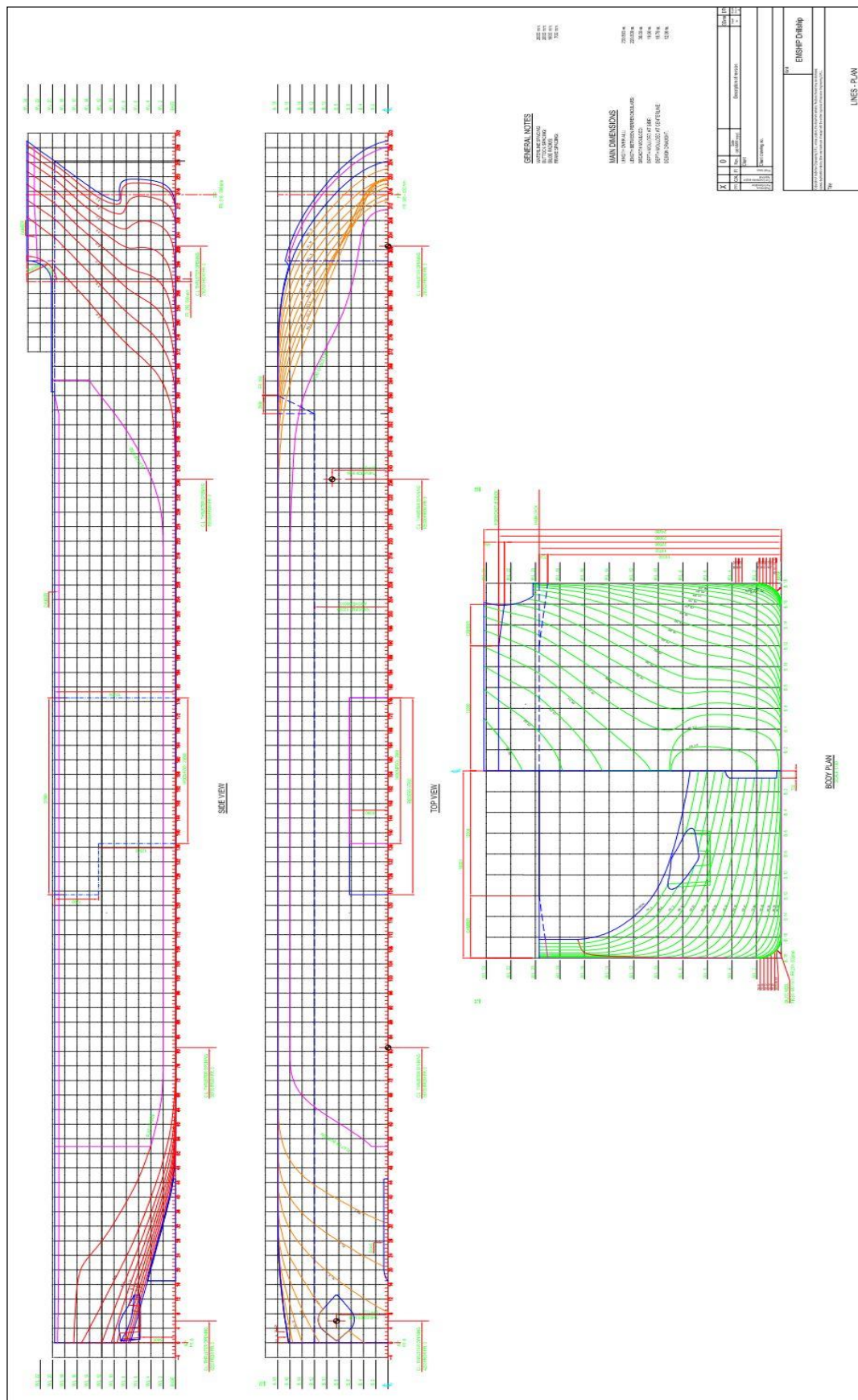
- [14] Papanikolaou A (2009a) Ship design—methodologies of preliminary ship design (in Greek: Μελέτη Πλοίου - Μεθοδολογίες Προμελέτης Πλοίου), SYMEON Publisher, Athens, Vol 1, ISBN 978-960-9600-09-01 & Vol. 2, ISBN 978-969-9400-11-4, October 2009
- [15] Trees Full of Money. Retrieved November 19, 2016, from <http://www.treesfullofmoney.com/an-introduction-to-drilling-offshore-oil-wells/>
- [16] NA 470. Course Notes. Parsons, M.G. University of Michigan. Ann Arbor, MI. Fall, 2002
- [17] Christiaan de Wit. Wave Optimal Thrust Allocation Methods for Dynamic Positioning of Ships. Master's thesis, Delft University of Technology, July 2009.
- [18] Weather and ocean forecast maps. (n.d.). Retrieved December 3, 2016, from <http://globalsailingweather.com/showMap.php?dom=cari1&sn=01>
- [19] Current Marine Data Oceanweather Inc. (n.d.). Retrieved December 3, 2016, from <http://www.oceanweather.com/data/>
- [20] Molland, A. F. (2008). The maritime engineering reference book: a guide to ship design, construction and operation. Amsterdam: Butterworth-Heinemann.
- [21] Lamb, T. (2003). Ship design and construction. Jersey City, NJ: Society of Naval Architects and Marine Engineers.

14.APPENDIX

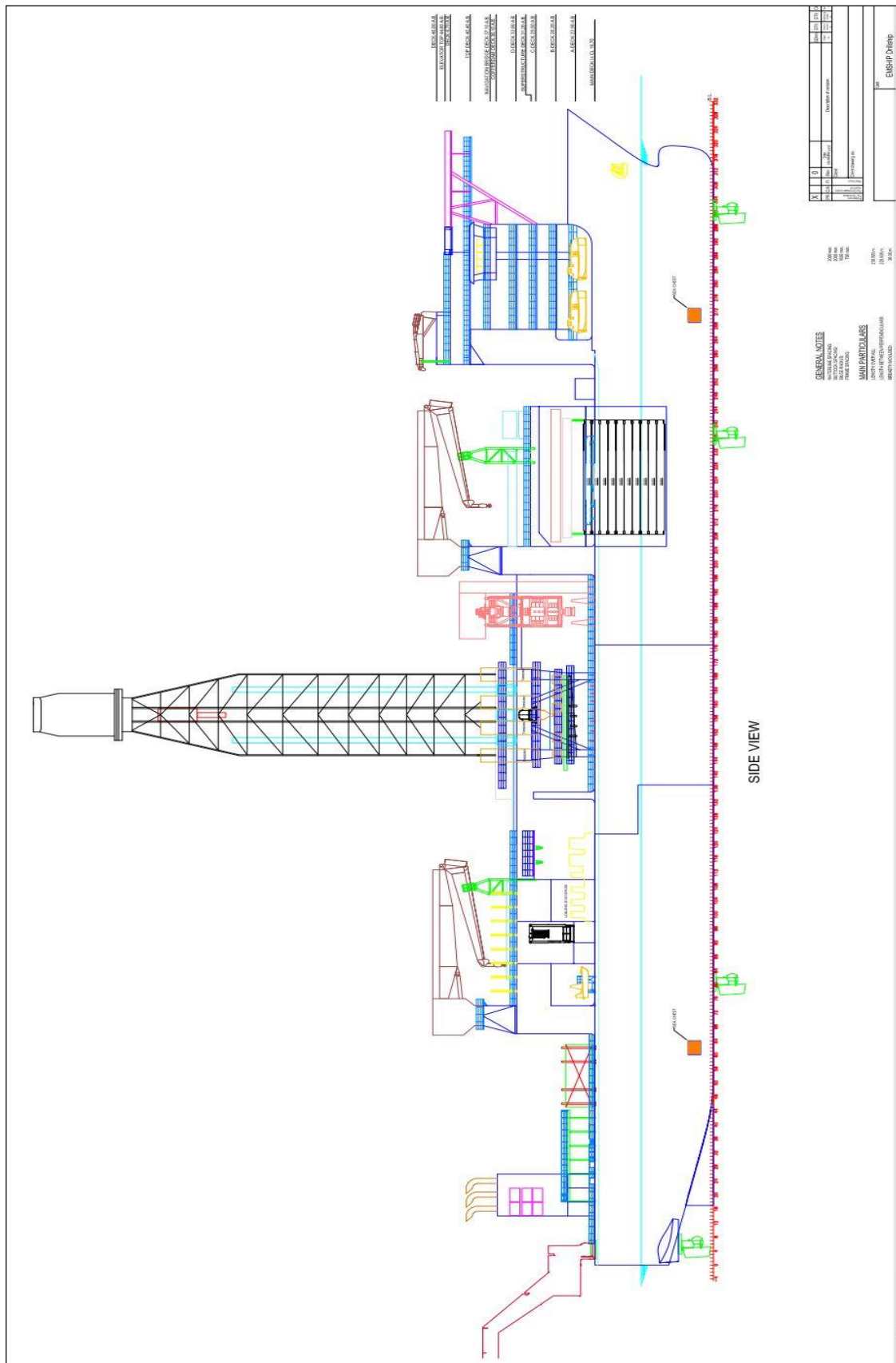
Appendix 1 – Database

Number of Ship	Name of Ships	Year	Main Dimensions					Operation Parameters (Characteristics)			
			Length [m]	Beam [m]	Depth [m]	Drilling Draft [m]	Displacement [m ton]	Water Depth [m]	Drilling Depth [m]	Moonpool Dim. [m * m]	Transit speed [knots]
1	Ensco DS-1	1999	207.3	30	19	9.6	41351	1828	96000	11.9 * 9.4	12
2	Ensco DS-2	1999	207.3	30	19.1	9.6	43260	1828	9600	12 * 9.45	11
3	Ensco DS-3	2010	227.8	42	19	12	96273	3048	12192	12.5 * 25.6	12
4	Ensco DS-4	2010	227.8	42	19	12	96507	3048	12192	12.5 * 25.6	12
5	Ensco DS-5	2011	229	42	19	12	96000	3048	12192	12.4 * 25.6	12
6	Ensco DS-6	2012	228.6	42	19.9	12	96506	3048	12192	12.49 * 25.6	12
7	Ensco DS-7	2013	228.6	42	18.9	12	78463	3048	9143	12.4 * 25.6	12
8	Ensco DS-8	2015	229.97	37.97	18.5	11	78463	3660	12192	10.68 * 48.4	13.7
9	Ensco DS-9	2015	229.97	37.97	18.5	11	78463	3658	12192	10.66 * 24.68	13
10	Ensco DS-10	2015	230	38	18.5	11	77000	3658	12192	12.8 * 25.6	10
11	West Auriga	2013	228	42	18.9	12	96140	3658	11433	18.3 * 24.4	11.5
12	West Capella	2008	228	42	18.9	12	96140	3658	11433	18.3 * 24.4	11.5
13	West Carina	2014	228	42	18.89	12	96140	3658	11433	18.3 * 24.4	11.5
14	West Dorado	2014	228	42		12	96140	3658	11433		12
15	West Draco	2014	228	42				3600	11400		12
16	West Gemini	2010	228	42		12	96400	3048	11433		
17	West Jupiter	2014	228	42		14		3600	11400		12
19	West Neptune	2014	228	42		12	96189	3660	11433		11.5
20	West Vela	2013	228	42		12	96189	3660	11433		11.5
21	West Saturn	2014	228	42		14		3600	11400		12
22	Maersk Valiant	2014	228	42	19	12	96000	3600	12000	12.5 * 25.6	
23	Maersk Viking	2014	228	42	19	12	96000	3600	12000	12.5 * 25.7	
24	Maersk Venturer	2014	228	42	19	12	96000	3600	12000	12.5 * 25.8	
25	Maersk Voyager	2014	228	42	19	12	96000	3600	12000	12.5 * 25.9	
26	Deepwater Asgard	2014	238.05	42.06	18.89	12	104000	3657	12192	10.21 * 24.99	12.5
27	Deepwater Discover	2000	227.38	41.76	19.5	12	98470	3048	9144	12.46 * 15.99	10
28	Deepwater Frontier	1999	221.29	42.06	20.11	12.8	103000	3048	9144	12.49 * 12.19	10
29	Deepwater Invictus	2014	238.05	42.06	18.89	12	104000	3657	12192	10.21 * 24.99	12.5
30	Deepwater Pathfinder	1998	238.05	42.06	20.11	13.11	93043	3048	9144	25.6 * 42.98	10
31	Deepwater Proteus	2016	238.05	42.06	18.89	11.88	103970	3657	12192	24.99 * 9.45	12.5
32	Dhirubhai Deepwater KG1	2009	228	41.76	18.89	11.88	96000	3657	10668	25.6 * 12.5	12
33	Dhirubhai Deepwater KG2	2010	228	41.76	18.89	11.88	96000	3657	10668	25.6 * 12.6	12
34	Discoverer India	2009	254.5	38.1	18.89	12.98	89286	3657	12192	22.25 * 9.14	12
35	Discoverer Americas	2010	254.5	38.1	18.89	12.19	90178	3657	12192	24.38 * 9.14	12
36	Discoverer Clear Leader	2008	254.5	38.1	18.89	12.8	89286	3657	12192	21.95 * 9.14	12
37	Discoverer Deep Seas	2001	254.5	38.1	19.51	12.8	101450	3048	10668	21.95 * 9.14	12
38	Discoverer Enterprise	1999	254.5	38.1	19.51	12.8	101450	3048	10668	22.25 * 9.12	12
39	Discoverer Inspiration	2008	254.5	38.1	18.89	12.8	89286	3657	12192	21.95 * 9.14	12
40	Discoverer Luanda	2010	254.5	38.1	18.89	12.8	89286	3657	9144	21.95 * 9.14	12
41	Discoverer Spirit	2000	254.5	38.1	18.89	12.8	82911	3048	10668	24.99 * 11.89	8
42	Deepwater Millennium	1999	221.29	42.06	20.12	12.8	104205	3048	9144	12.8 * 12.49	8
43	GSF C.R. Luigs	2000	231.34	35.97	18.29	10.06	62638	3048	10668	12.8 * 12.8	12
44	GSF Jack Ryan	2000	231.34	35.97	18.29	10.06		3048	10668	12.8 * 12.8	12
45	Petobras 10000	2009	227.99	41.76	18.89	11.88	97997	3657	11277	25.6 * 12.49	12
46	HuisDrill		189	32.2	18.9	12	54000	3048	12000	27.1 * 11	12
47	HuisDrill 1200		208	36.4	23.8	12	70000	4000	15240	38.4 * 9.8	12
48	Noble Sam Croft	2014	229.21	35.97	18.59	10.97		3048	12192	35.1 * 12.49	
49	Noble Bob Douglas	2013	229.21	35.97	18.59	10.97		3048	12192	35.1 * 12.49	
50	Noble Bully 1	2011	187.5	39	14.94	10.06		2499	12192	19.51 * 12.49	
51	Noble Bully 2	2011	187.5	39	14.94	10.06		2514	12192	19.51 * 12.49	
52	Noble Don Taylor	2013	229.21	35.97	18.59	10.97		3048	12192	35.05 * 12.49	
53	Noble Globetrotter 1	2011	189	32	18.89	11.89		3048	12192	27.13 * 11.28	
54	Noble Globetrotter 2	2013	189	32	18.89	11.89		3048	12192	27.13 * 11.29	
55	Noble Tom Madden	2014	229.21	35.97	18.59	10.97		3048	12192	27.13 * 11.30	
56	Atwood Advantage	2013	238	42	19	12	104000	3657	12192		
57	Atwood Archer	2018	238	42	19	12	104000	3657	12192		
58	Atwood Achiever	2014	238	42	19	12	104000	3657	12192		
59	Atwood Admiral	2017	238	42	19	12	104000	3657	12192		
60	Ocean Rig Olympia	2011	228	42	18.9	12		3048	10668		12
61	Ocean Rig Poseidon	2011	228	42	18.9	11.5		3048			12
62	Ocean Rig Mykonos	2011	228	42	18.9	11.5		3048			12
63	Ocean Rig Corcovado	2011	228	42	18.9	12		3048			12
64	Ocean Rig Mylos	2013	228	42	18.9	12		3650			12
65	Ocean Rig Skyros	2013	228	42	18.9	12		3650			12
66	Ocean Rig Athena	2013	228	42	18.9	12		3650			12
67	Ocean Rig Apollo	2015	228	42	18.9	12		3650			12
68	Ocean Rig Santorini	2017	228	42	18.9	12		3650			12
69	Ocean Rig Crete	2018	230	42		9.5		3657	12192		12
70	Ocean Rig Amorgos	2019	230	42		9.5		3657	12192		12
71	Platinum Explorer	2010	222.8	42.06	18.9	11.98	102826	3048	12192	24.99 * 12.5	
72	Tungsten Explorer	2013	238	42.06	18.9	11.98	102682	3657	12192	24.99 * 12.5	
73	Titanium Explorer	2012	238	42.06	18.9	11.98	102808	3657	12192	24.99 * 12.5	

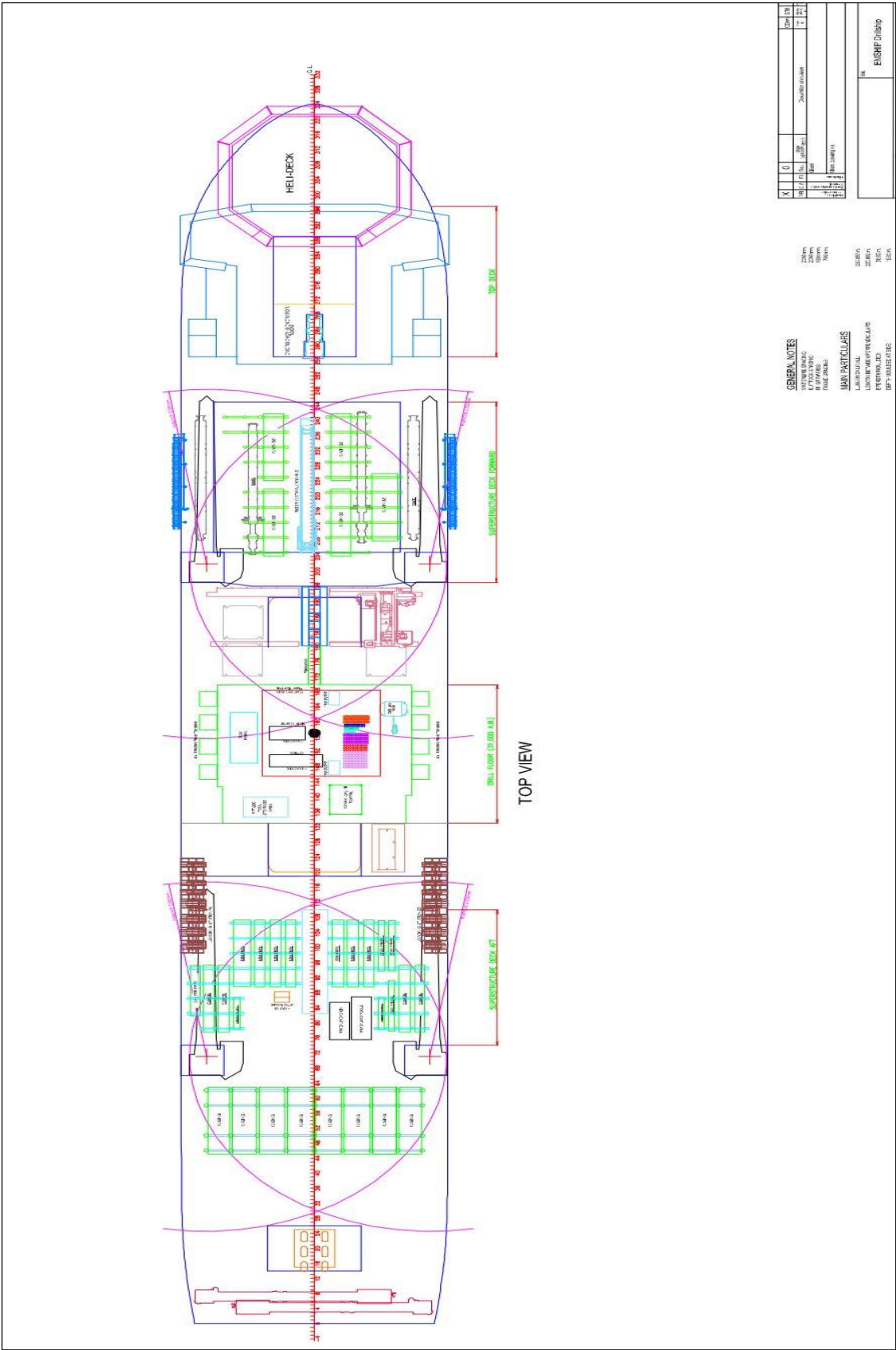
Appendix 2 – Lines Plan AutoCAD



Appendix 3 – General Arrangement Plan Profile View



Appendix 3 – General Arrangement Plan – Main Deck



Appendix 4 – Hydrostatics Table

Draft (m)	Displ (t)	LCB (m)	VCB (m)	WPA (m ²)	LCF (m)	KML (m)	KMT (m)	WSA (m ²)	TPC (t/cm)	MTC (t-m/cm)	CB	CM	CP	CW	BML (m)	BMT (m)	BML (m)	Vol(mld) (m ³)	IL (m ⁴)	IT (m ⁴)
1	5638.33	119.742	0.478	5621.75	118.536	2724.49	100.934	6672.71	57.62	696.21	0.678	0.608	1.115	0.708	2724.02	100.457	2724.02	5387.38	14984308.2	552593
1.5	8574.5	119.213	0.746	5807.81	117.877	1909.69	70.878	7057.23	59.53	741.96	0.692	0.619	1.117	0.731	1908.94	70.132	1908.94	8245.04	15969000.9	586680
2	11598.8	118.768	1.01	5932.57	117.082	1488.77	54.447	7457.04	60.81	782.21	0.704	0.627	1.124	0.747	1487.76	53.437	1487.76	11189.15	16835405.9	604690
2.5	14674.4	118.334	1.271	6032.04	116.382	1228.75	44.03	7895.38	61.83	816.49	0.714	0.631	1.131	0.76	1227.48	42.759	1227.48	14182.26	17573216.7	612154
3	17798.3	117.932	1.532	6118.37	115.619	1051.96	37.388	8254.71	62.71	847.47	0.723	0.635	1.139	0.77	1050.43	35.856	1050.43	17223.84	18239850.4	622607
3.5	20958	117.526	1.793	6195.11	114.909	923.038	32.7	8611.39	63.5	875.19	0.73	0.637	1.147	0.78	921.245	30.907	921.245	20300.46	18836570.4	631959
4	24158.1	117.124	2.053	6277.38	114.088	829.861	29.299	8966.44	64.34	906.5	0.737	0.638	1.155	0.79	827.808	27.246	827.808	23416.42	19510472.3	642161
4.5	27418.9	116.723	2.315	6384.74	113.347	761.791	26.93	9347.33	65.44	943.93	0.744	0.64	1.163	0.804	759.476	24.614	759.476	26591.19	20316043.2	658438
5	30717.4	116.303	2.578	6459.97	112.48	702.562	24.82	9701.65	66.21	974.65	0.751	0.641	1.171	0.813	699.984	22.242	699.984	29803.22	20977235.5	666553
5.5	34055	115.871	2.841	6554.86	111.273	661.637	23.183	10048.6	67.19	1016.97	0.757	0.642	1.18	0.825	658.796	20.342	658.796	33053.57	21888114.7	675850
6	37445.9	115.394	3.105	6660.38	109.878	630.47	21.876	10417.6	68.27	1064.88	0.763	0.642	1.188	0.839	627.365	18.772	627.365	36355.44	22919243.2	685773
6.5	40883.8	114.887	3.37	6732.68	108.977	594.804	20.751	10754.3	69.01	1096.06	0.769	0.643	1.196	0.848	591.434	17.381	591.434	39703.81	23590316.4	693278
7	44358.8	114.381	3.636	6801.6	108.087	563.71	19.827	11089.1	69.72	1126.16	0.775	0.643	1.205	0.856	560.074	16.192	560.074	43088.33	24238251.8	700731
7.5	47863.5	113.888	3.901	6856.42	107.367	534.008	19.049	11407.7	70.28	1150.13	0.781	0.644	1.213	0.863	530.107	15.148	530.107	46502.18	24753949.4	707360
8	51410.2	113.438	4.167	6903.34	106.826	506.086	18.417	11717.7	70.76	1169.66	0.786	0.644	1.221	0.869	501.92	14.25	501.92	49957.11	25174451.3	714734
8.5	54961.4	112.993	4.431	6935.56	106.413	478.964	17.881	12019	71.09	1182.23	0.791	0.645	1.228	0.873	474.533	13.45	474.533	53416.59	25444887.4	721202
9	58525.1	112.579	4.695	6955.13	106.024	452.574	17.43	12318.7	71.29	1188.17	0.796	0.645	1.234	0.876	447.879	12.735	447.879	56888.28	25572856.4	727162
9.5	62106.2	112.192	4.958	6983.33	105.714	430.93	17.065	12616.4	71.58	1199.2	0.8	0.645	1.24	0.879	425.972	12.107	425.972	60376.97	25810283.1	733576
10	65698	111.833	5.22	7020.81	105.499	413.531	16.77	12907.7	71.96	1215.96	0.804	0.645	1.246	0.884	408.31	11.55	408.31	63876.17	26170904.1	740295
10.5	69310.7	111.51	5.483	7053.04	105.605	397.206	16.514	13195.9	72.29	1230.71	0.808	0.646	1.252	0.892	391.723	11.031	391.723	67395.9	26488404.8	745904
11	72936.1	111.217	5.745	7084.35	105.635	382.394	16.305	13484.5	72.61	1245.25	0.812	0.646	1.257	0.892	376.649	10.559	376.649	70927.95	26801276.3	751378
11.5	76580	110.948	6.008	7113.36	105.701	368.642	16.133	13772.6	72.91	1258.81	0.815	0.646	1.262	0.896	362.635	10.126	362.635	74478.08	27092334.2	756502
12	80235	110.711	6.27	7139.02	105.821	355.736	15.99	14059.7	73.17	1271	0.819	0.646	1.267	0.899	349.466	9.72	349.466	78039.07	27355560.6	760862
12.5	83904.8	110.5	6.531	7164.73	105.945	343.949	15.88	14471.2	73.44	1283.31	0.822	0.646	1.272	0.902	337.418	9.349	337.418	81612.32	27620454.5	765272
13	87523	110.325	6.789	7067.29	106.341	333.086	15.785	14770.1	72.44	1294.53	0.825	0.646	1.276	0.89	326.297	8.996	326.297	85137.21	27861926.2	768151
13.5	91166.5	110.175	7.047	7092.41	106.549	323.244	15.73	15069.3	72.7	1306.68	0.827	0.647	1.279	0.893	316.197	8.683	316.197	88686.74	28123482.8	772285
14	94810.9	110.039	7.305	7116.84	106.771	314.122	15.697	15369	72.95	1318.6	0.83	0.647	1.283	0.896	306.816	8.392	306.816	92237.14	28380035.9	776228
14.5	98470.1	109.924	7.564	7138.95	106.988	305.468	15.677	15668.9	73.17	1329.71	0.832	0.647	1.286	0.899	297.905	8.114	297.905	95801.97	28619200	779467
15	102137	109.823	7.822	7159.93	107.222	297.34	15.673	15969.4	73.39	1340.4	0.834	0.647	1.289	0.902	289.518	7.851	289.518	99374.62	28849295.1	782344
15.5	105815	109.736	8.081	7180.35	107.475	289.719	15.685	16270.4	73.6	1350.88	0.836	0.647	1.293	0.904	281.639	7.604	281.639	102957.8	29074799.7	785025
16	109504	109.664	8.339	7200.5	107.743	282.585	15.711	16572.3	73.81	1361.27	0.839	0.647	1.296	0.907	274.246	7.372	274.246	106551	29298484.2	787600
16.5	110612	109.645	8.417	7206.59	107.836	280.543	15.722	16663	73.87	1364.42	0.839	0.647	1.297	0.907	272.136	7.306	272.136	107630.9	29366250	788373
16.5	113202	109.605	8.598	7220.88	108.022	275.939	15.753	16874.8	74.01	1371.82	0.841	0.647	1.299	0.909	267.341	7.155	267.341	110154.5	29525550.5	790178
17	116917	109.564	8.857	7240.81	108.315	269.667	15.806	17177.8	74.22	1382.22	0.843	0.647	1.302	0.912	260.809	6.948	260.809	113773.3	29749296.5	792579
17.5	120635	109.53	9.116	7259.98	108.6	263.749	15.869	17481.2	74.41	1392.41	0.845	0.647	1.305	0.914	254.633	6.752	254.633	117396	29968586.2	794713
18	124364	109.506	9.375	7279.33	108.889	258.205	15.943	17785.2	74.61	1402.73	0.847	0.647	1.308	0.917	248.829	6.568	248.829	121028.3	30190638	796878
18.5	128107	109.496	9.635	7296.93	109.153	252.813	16.027	18088.7	74.79	1412.13	0.849	0.647	1.311	0.919	243.178	6.392	243.178	124674.8	30392971.6	798863

Appendix 5 – Main Engine Spec, Thruster Spec and Endurance Range

Wärtsilä 32			IMO Tier II or III					
Cylinder bore	320 mm	Fuel specification: Fuel oil						
Piston stroke	400 mm	700 cSt/50°C	7200 sR1/100°F					
Cylinder output	580 kW/cyl	ISO 8217, category ISO-F-RMK 700						
Speed	750 rpm	SFOC 174 g/kWh at ISO condition						
Mean effective pressure	28.9 bar							
Piston speed	10.0 m/s							
Rated power								
Engine type	580 kW/cyl							
6L32	3 480							
8L32	4 640							
9L32	5 220							
12V32	6 960							
16V32	9 280							
Dimensions (mm) and weights (tonnes)								
Engine type	A*	A	B*	B	C	D	F	Weight
6L32	4 980	5 260	2 560	2 490	2 305	2 345	1 155	33.3
8L32	5 960	6 245	2 360	2 295	2 305	2 345	1 155	43.4
9L32	6 450	6 730	2 360	2 295	2 305	2 345	1 155	46.8
12V32	6 935	6 615	2 715	2 665	3 020	2 120	1 475	58.7
16V32	8 060	7 735	2 480	2 430	3 020	2 120	1 475	74.1
* Turbocharger at flywheel end.								

Wärtsilä Genset 32			IMO Tier II or III			
Cylinder bore	320 mm	Generator voltage 0.4–13.8 kV				
Piston stroke	400 mm	Generator efficiency 0.95–0.97				
Cylinder output	560, 580 kW/cyl	Fuel specification: Fuel oil				
Speed	720, 750 rpm	700 cSt/50°C	7200 sR1/100°F			
Mean effective pressure	28.9 bar					
Piston speed	9.6, 10.0 m/s	SFOC 174 g/kWh at ISO condition				
Rated power						
Engine type	60 Hz/720 rpm		50 Hz/750 rpm			
	560 kW/cyl		580 kW/cyl			
	Engine kW	Gen. kW	Engine kW	Gen. kW		
6L32	3 360	3 230	3 480	3 340		
8L32	4 480	4 300	4 640	4 450		
9L32	5 040	4 840	5 220	5 010		
12V32	6 720	6 450	6 960	6 680		
16V32	8 960	8 600	9 280	8 910		
Dimensions (mm) and weights (tonnes)						
Engine type	A*	E*	I*	K	L*	Weight*
6L32	8 505	2 490	1 450	2 345	3 745	57
8L32	10 410	2 690	1 630	2 345	4 010	76
9L32	10 505	2 890	1 630	2 345	4 010	86
12V32	10 700	3 060	1 700	2 120	4 130	100
16V32	11 465	3 360	1 850	2 120	4 445	127
* Dependent on generator type and size. Generator output based on a generator efficiency of 96%. Final measurements might differ depending on selected turbocharger execution.						

TECHNICAL SPECIFICATIONS

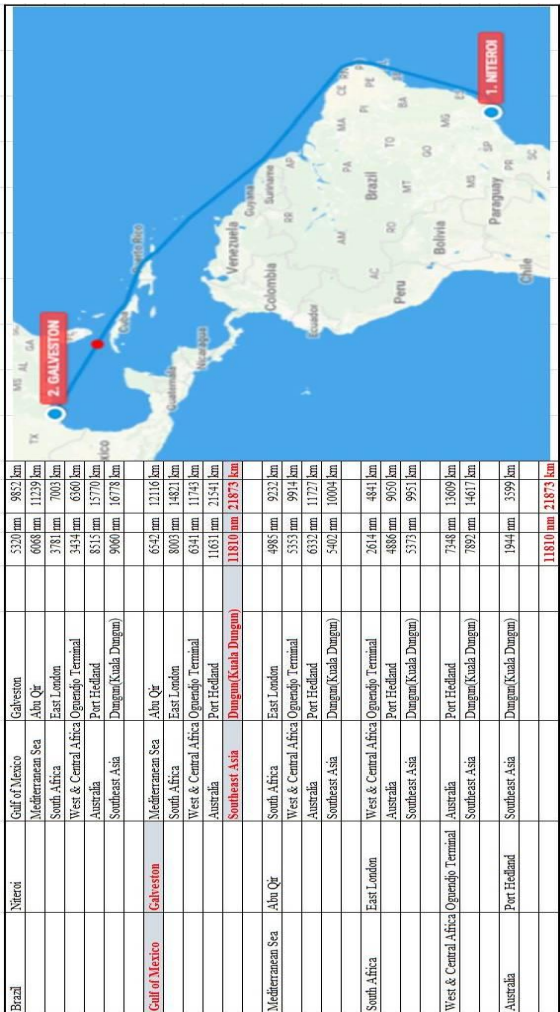
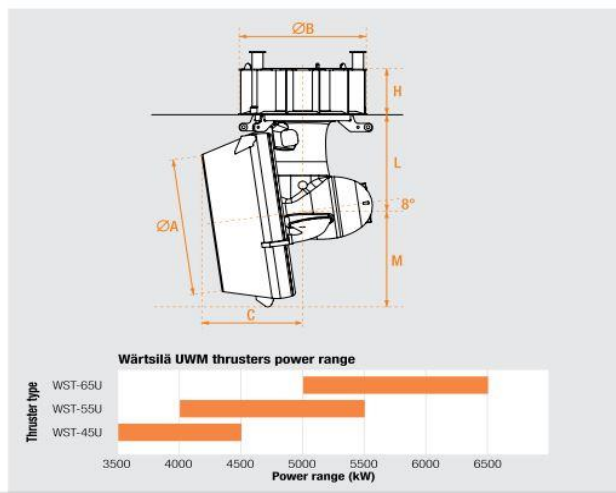
The UWM thruster series includes the Wärtsilä WST-45U and WST-55U steerable thrusters, as well as the largest model, the Wärtsilä WST-65U. The maximum power levels are 4500, 5500, and 6500 kW respectively. By providing two propeller diameter options for the most common power levels of 4500 and 5500 kW, the UWM series provides options for high bollard pull and specific propeller diameter requirements.

Main technical data				
Type	Input speed rpm	Power * kW	Propeller diameter mm	Tilt
WST-45U	720	4000	3600	8°
	600	4500		
WST-55U	720	4500	3900	8°
	600	5500		
WST-65U	720	5500	4200	8°
	600	6500		

* The maximum power level depends on input speed and is given for dynamic positioning (DP) application.

Dimensions and weights									
Type	A mm	B mm	C mm	M mm	L mm	H mm	Outboard part ¹ dry kg	in water kg	Well ² kg
WST-45U	3600	3300	2518	2324	2570	1242	51000	34500	9500
WST-55U	3900	3625	2648	2516	2763	1390	63500	31800	11900
WST-65U	4200	3820	3006	2719	3025	1390	77500	49500	13200

1) With oil
2) Including top cover, bottom cover, pipe covers, valves and position sensor



Appendix 6 –Tanks Contents and Capacities A

Category BW - BW (SG = 1.025 tonnes/cu.m 0 % full)

Compartment	Frames	Volume (m ³)	Weight (t)	LCG (m)	TCG (m)	VCG (m)	IT (m ⁴)
WB11Port	17-54	1533.261	0.000	27.791	-11.393	5.967	3061
WB11Stb	17-54	1533.263	0.000	27.791	11.393	5.967	3061
WB4Port	211-241	1317.759	0.000	158.197	-14.949	7.351	389
WB4Stb	211-241	1317.759	0.000	158.197	14.949	7.351	389
WB5Port	193-233	1142.876	0.000	149.100	-5.950	3.850	3853
WB5Stb	193-233	1142.876	0.000	149.100	5.950	3.850	3853
WB9Port	85-99	615.136	0.000	64.400	-14.950	7.350	181
WB9Stb	85-99	615.136	0.000	64.400	14.950	7.350	181
WBDB10C	54-85	497.974	0.000	47.611	-0.000	1.050	3464
WBDB10Port	54-85	458.566	0.000	49.332	-11.853	1.105	2655
WBDB10Stb	54-85	458.566	0.000	49.332	11.853	1.105	2655
WBDB1C	297-319	1633.845	0.000	214.165	-0.000	6.459	5003
WBDB2Port	273-297	346.369	0.000	198.732	-5.183	1.107	2179
WBDB2Stb	273-297	346.369	0.000	198.732	5.183	1.107	2179
WBDB3C	241-273	580.850	0.000	179.900	0.000	1.050	3659
WBDB3Port	241-273	418.370	0.000	178.644	-11.227	1.091	1992
WBDB3Stb	241-273	418.370	0.000	178.644	11.227	1.091	1992
WBDB4C	211-241	544.547	0.000	158.200	0.000	1.050	3430
WBDB4Port	211-241	420.056	0.000	156.954	-12.349	1.076	2318
WBDB4Stb	211-241	420.056	0.000	156.954	12.349	1.076	2318
WBDB6C	179-211	580.850	0.000	136.500	0.000	1.050	3659
WBDB6Port	179-211	521.398	0.000	136.500	-11.963	1.071	2929
WBDB6Stb	179-211	521.398	0.000	136.500	11.963	1.071	2929
WBDB7Port	147-179	456.858	0.000	114.100	-12.664	1.074	1998
WBDB7Stb	147-179	456.859	0.000	114.100	12.664	1.074	1998
WBDB8C	117-135	326.728	0.000	88.200	0.000	1.050	2058
WBDB8Port	117-147	464.609	0.000	92.072	-12.222	1.072	2423
WBDB8Stb	117-147	464.609	0.000	92.072	12.222	1.072	2423
WBDB9C	85-117	580.850	0.000	70.700	0.000	1.050	3659
WBDB9Port	85-117	521.399	0.000	70.700	-11.963	1.071	2929
WBDB9Stb	85-117	521.399	0.000	70.700	11.963	1.071	2929
Total		21179.000	0.000	118.313	0.000	3.642	

Category FW - FW (SG = 1.000 tonnes/cu.m 0 % full)

Compartment	Frames	Volume (m ³)	Weight (t)	LCG (m)	TCG (m)	VCG (m)	IT (m ⁴)
FWStore1Port	273-297	984.539	0.000	199.988	-13.013	12.439	357
FWStore1Stb	273-297	984.539	0.000	199.988	13.013	12.439	357
Total		1969.080	0.000	199.988	0.000	12.439	

Appendix 6 –Tanks Contents and Capacities B

Category MDO - MDO (SG = 0.890 tonnes/cu.m 0 % full)

Compartment	Frames	Volume (m ³)	Weight (t)	LCG (m)	TCG (m)	VCG (m)	IT (m ⁴)
MDO/MGO1Port	179-193	1200.020	0.000	130.200	-5.950	7.350	0
MDO/MGO1Stb	179-193	1200.020	0.000	130.200	5.950	7.350	0
MDO/MGO2Port	135-147	786.568	0.000	98.700	-12.250	7.350	0
MDO/MGO2Stb	135-147	786.568	0.000	98.700	12.250	7.350	0
MDO/MGOServiceER2	17-21	48.404	0.000	13.300	0.000	13.650	0
MDO/MGOServicePort	9-17	26.891	0.000	9.100	-12.600	14.350	0
MDO/MGOServiceStb	9-17	26.891	0.000	9.100	12.600	14.350	0
MDOMGOSETTPort	17-25	121.010	0.000	14.700	-3.150	7.350	114
MDOMGOSETTStb	17-25	121.010	0.000	14.700	3.150	7.350	114
Total		4317.380	0.000	109.428	0.000	7.508	

Category LO - LO (SG = 0.900 tonnes/cu.m 0 % full)

Compartment	Frames	Volume (m ³)	Weight (t)	LCG (m)	TCG (m)	VCG (m)	IT (m ⁴)
ENGLOSTORAGE	12-17	75.631	0.000	10.150	3.150	10.850	71
ENGLOSTORAGEPort	12-17	42.017	0.000	10.150	-8.050	10.850	12
ENGLOSTORAGEStb	12-17	42.018	0.000	10.150	8.050	10.850	12
ENGLOUSEDPort	12-17	33.614	0.000	10.150	-4.900	10.850	6
THRGOSTORAGEPort	12-17	42.018	0.000	10.150	-1.750	10.850	12
Total		235.300	0.000	10.150	0.000	10.850	

Category BO - BO (SG = 0.800 tonnes/cu.m 0 % full)

Compartment	Frames	Volume (m ³)	Weight (t)	LCG (m)	TCG (m)	VCG (m)	IT (m ⁴)
BO1Stb	147-155	524.378	0.000	105.700	12.250	7.350	344
Total		524.380	0.000	105.700	12.250	7.350	

Category DW - DW (SG = 1.000 tonnes/cu.m 0 % full)

Compartment	Frames	Volume (m ³)	Weight (t)	LCG (m)	TCG (m)	VCG (m)	IT (m ⁴)
DW1Port	193-211	790.889	0.000	141.400	-14.950	7.350	233
DW1Stb	193-211	790.889	0.000	141.400	14.950	7.350	233
DW2Port	99-117	790.889	0.000	75.600	-14.950	7.350	233
DW2Stb	99-117	790.889	0.000	75.600	14.950	7.350	233
Total		3163.560	0.000	108.500	0.000	7.350	

Appendix 6 –Tanks Contents and Capacities C

Category BR - BR (SG = 1.920 tonnes/cu.m 0 % full)

Compartment	Frames	Volume (m ³)	Weight (t)	LCG (m)	TCG (m)	VCG (m)	IT (m ⁴)
BR1Port	179-193	615.136	0.000	130.200	-14.950	7.350	181
BR1Stb	179-193	615.136	0.000	130.200	14.950	7.350	181
Total		1230.280	0.000	130.200	0.000	7.350	

Category MUD - MUD (SG = 2.160 tonnes/cu.m 0 % full)

Compartment	Frames	Volume (m ³)	Weight (t)	LCG (m)	TCG (m)	VCG (m)	IT (m ⁴)
AM1Port	126-135	242.021	0.000	91.350	-9.100	5.600	90
AM1Stb	126-135	242.021	0.000	91.350	9.100	5.600	90
AM2Port	117-126	242.021	0.000	85.050	-9.100	5.600	90
AM2Stb	117-126	242.021	0.000	85.050	9.100	5.600	90
AM3Port	108-117	242.021	0.000	78.750	-9.100	5.600	90
AM3Stb	108-117	242.021	0.000	78.750	9.100	5.600	90
AM4Port	99-108	242.021	0.000	72.450	-9.100	5.600	90
AM4Stb	99-108	242.021	0.000	72.450	9.100	5.600	90
CHEMPort	95-99	60.505	0.000	67.900	-4.725	5.600	7
CHEMStb	95-99	60.505	0.000	67.900	4.725	5.600	7
RM1Port	85-99	376.477	0.000	64.400	-9.100	5.600	140
RM1Stb	85-99	376.477	0.000	64.400	9.100	5.600	140
RM2Port	85-95	302.526	0.000	63.000	-3.150	5.600	142
RM2Stb	85-95	302.526	0.000	63.000	3.150	5.600	142
SLUGPort	95-99	60.505	0.000	67.900	-1.575	5.600	7
SLUGStb	95-99	60.505	0.000	67.900	1.575	5.600	7
WASTEMUD	126-135	211.768	0.000	91.350	14.350	5.600	60
WASTEMUDPort	126-135	211.768	0.000	91.350	-14.350	5.600	60
Total		3959.760	0.000	75.839	0.000	5.600	

Category MIS - MIS (SG = 2.000 tonnes/cu.m 0 % full)

Compartment	Frames	Volume (m ³)	Weight (t)	LCG (m)	TCG (m)	VCG (m)	IT (m ⁴)
HAZDRAIN	117-126	211.768	0.000	85.050	-14.350	5.600	60
NONHAZDRAIN	117-126	395.445	0.000	85.050	14.950	7.350	116
Total		607.210	0.000	85.050	4.731	6.740	

Category DO - DO (SG = 1.000 tonnes/cu.m 0 % full)

Compartment	Frames	Volume (m ³)	Weight (t)	LCG (m)	TCG (m)	VCG (m)	IT (m ⁴)
BWH	25-35	110.926	0.000	20.491	2.514	7.350	87
DIRTYOIL	29-35	40.337	0.000	22.400	4.900	7.350	7
SLUDGE	25-35	110.926	0.000	20.491	-2.514	7.350	87
WASTEOIL	29-35	40.337	0.000	22.400	-4.900	7.350	7
Total		302.540	0.000	21.000	0.000	7.350	

Appendix 7 – Dynamic Position Calculation A

WVEL	40.00	knots	WDIR	0.00	deg
WVREL	40.00	knots	WDREL	0.00	deg
CVEL	1.00	knots	CDIR	0.00	deg
CVREL	1.00	knots	CDREL	0.00	deg
SIGW	5.80	metres	WVDIR	0.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	-28.55	-0.84	-31.43	0.00	-60.82	60.82
TRANS (tonnes)	0.00	0.00	0.00	0.00	0.00	0.00
YAW M (tonnes-metres)	0.00	0.00	0.00	0.00	0.00	-0.00

Thruster	Thrust Req (tonnes)	Thrust Req (tonnes)	Direction (deg)	Power (kW)
1	10.14	(10.14)	0.00	629.92
2	10.14	(10.14)	0.00	629.92
3	10.14	(10.14)	360.00	629.92
4	10.14	(10.14)	0.00	629.92
5	10.14	(10.14)	360.00	629.92
6	10.14	(10.14)	360.00	629.92

WVEL	40.00	knots	WDIR	15.00	deg
WVREL	40.00	knots	WDREL	15.00	deg
CVEL	1.00	knots	CDIR	15.00	deg
CVREL	1.00	knots	CDREL	15.00	deg
SIGW	5.80	metres	WVDIR	15.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	-25.99	-0.81	-39.97	0.00	-66.77	66.77
TRANS (tonnes)	-34.61	-5.77	-19.46	0.00	-59.84	59.84
YAW M (tonnes-metres)	-909.37	-256.71	-289.99	0.00	-1456.06	1456.06

Thruster	Thrust Req (tonnes)	Thrust Req (tonnes)	Direction (deg)	Power (kW)
1	12.69	(12.69)	24.68	788.66
2	11.96	(11.96)	26.30	743.27
3	13.60	(13.60)	35.12	845.43
4	17.46	(17.46)	48.53	1085.31
5	16.90	(16.90)	50.76	1050.02
6	18.87	(18.87)	53.87	1172.83

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WVREL	40.00	knots	WDREL	60.00	deg																																																																																																																																																																																																																																																																																																	
CVEL	1.00	knots	CDIR	60.00	deg																																																																																																																																																																																																																																																																																																	
CVREL	1.00	knots	CDREL	60.00	deg																																																																																																																																																																																																																																																																																																	
SIGW	5.80	metres	WVDIR	60.00	deg																																																																																																																																																																																																																																																																																																	
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	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED		-----------------------	---------	---------	----------	----------	----------	----------		LONG (tonnes)	-8.37	-0.42	-87.36	0.00	-96.15	96.15		TRANS (tonnes)	-164.20	-18.69	-196.56	0.00	-379.45	379.45		YAW M (tonnes-metres)	-909.37	-687.25	-1182.76	0.00	-2779.38	2779.38		Thruster	Thrust Req (tonnes)	Thrust Req (tonnes)	Direction (deg)	Power (kW)		----------	---------------------	---------------------	-----------------	------------		1	51.68	(51.68)	70.46	3211.58		2	50.89	(50.89)	73.13	3162.69		3	58.79	(58.79)	74.18	3653.57		4	74.96	(74.96)	76.59	4658.50		5	74.38	(74.38)	78.63	4622.12		6	81.23	(81.23)	78.62	5047.58									-------	-------	--------	-------	-------	-----		WVEL	40.00	knots	WDIR	75.00	deg		WVREL	40.00	knots	WDREL	75.00	deg		CVEL	1.00	knots	CDIR	75.00	deg		CVREL	1.00	knots	CDREL	75.00	deg		SIGW	5.80	metres	WVDIR	75.00	deg		TPER	8.40	sec						WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED		-----------------------	---------	---------	---------	----------	---------	----------		LONG (tonnes)	-2.08	-0.22	-43.68	0.00	-45.97	45.97		TRANS (tonnes)	-167.12	-20.80	-273.38	0.00	-461.29	461.29		YAW M (tonnes-metres)	170.51	-495.23	-663.53	0.00	-988.26	988.26		Thruster	Thrust Req (tonnes)	Thrust Req (tonnes)	Direction (deg)	Power (kW)		----------	---------------------	---------------------	-----------------	------------		1	65.39	(65.39)	82.35	4063.45		2	65.14	(65.14)	84.17	4048.17		3	71.75	(71.75)	83.87	4458.62		4	85.38	(85.38)	84.09	5305.54		5	85.17	(85.17)	85.60	5292.88		6	90.82	(90.82)	85.16	5643.79			WVEL	40.00	knots	WDIR	90.00	deg																																																								
WVREL	40.00	knots	WDREL	90.00	deg																																																																																																																																																																																																																																																																																																	
CVEL	1.00	knots	CDIR	90.00	deg																																																																																																																																																																																																																																																																																																	
CVREL	1.00	knots	CDREL	90.00	deg																																																																																																																																																																																																																																																																																																	
SIGW	5.80	metres	WVDIR	90.00	deg																																																																																																																																																																																																																																																																																																	
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	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED		-----------------------	---------	---------	---------	----------	---------	----------		LONG (tonnes)	3.60	0.00	0.00	0.00	3.60	-3.60		TRANS (tonnes)	-166.26	-21.99	-321.93	0.00	-510.18	510.18		YAW M (tonnes-metres)	1288.27	-141.49	-177.51	0.00	969.27	-969.27		Thruster	Thrust Req (tonnes)	Thrust Req (tonnes)	Direction (deg)	Power (kW)		----------	---------------------	---------------------	-----------------	------------		1	76.50	(76.50)	89.90	4754.02		2	76.51	(76.51)	91.00	4754.72		3	81.12	(81.12)	90.42	5040.78		4	90.71	(90.71)	89.87	5636.83		5	90.72	(90.72)	90.88	5637.48		6	94.65	(94.65)	90.36	5881.53									-------	-------	--------	-------	--------	-----		WVEL	40.00	knots	WDIR	105.00	deg		WVREL	40.00	knots	WDREL	105.00	deg		CVEL	1.00	knots	CDIR	105.00	deg		CVREL	1.00	knots	CDREL	105.00	deg		SIGW	5.80	metres	WVDIR	105.00	deg		TPER	8.40	sec						WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED		-----------------------	---------	---------	---------	----------	---------	----------		LONG (tonnes)	6.09	0.22	43.68	0.00	49.99	-49.99		TRANS (tonnes)	-168.75	-21.07	-273.38	0.00	-463.20	463.20		YAW M (tonnes-metres)	3012.28	454.80	456.73	0.00	3923.80	-3923.80		Thruster	Thrust Req (tonnes)	Thrust Req (tonnes)	Direction (deg)	Power (kW)		----------	---------------------	---------------------	-----------------	------------		1	77.03	(77.03)	96.17	4786.73		2	77.04	(77.04)	96.25	4787.45		3	77.37	(77.37)	96.18	4807.73		4	78.05	(78.05)	96.08	4850.28		5	78.06	(78.06)	96.17	4851.05		6	78.34	(78.34)	96.10	4868.28																																																																

Appendix 7 – Dynamic Position Calculation B

WVEL	40.00	knots	WDIR	120.00	deg
WVREL	40.00	knots	WDREL	120.00	deg
CVEL	1.00	knots	CDIR	120.00	deg
CVREL	1.00	knots	CDREL	120.00	deg
SIGW	5.80	metres	WVDIR	120.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	13.11	0.42	87.36	0.00	100.88	-100.88
TRANS (tonnes)	-158.87	-18.87	-196.56	0.00	-374.30	374.30
YAW M (tonnes-metres)	4054.26	687.25	1182.76	0.00	5924.28	-5924.28

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)
1	70.49	(70.49)	104.23	4380.64
2	70.25	(70.25)	103.42	4365.38
3	67.25	(67.25)	104.48	4179.10
4	60.95	(60.95)	106.56	3787.76
5	60.64	(60.64)	105.55	3768.61
6	58.16	(58.16)	106.80	3614.45

WVEL	40.00	knots	WDIR	135.00	deg
WVREL	40.00	knots	WDREL	135.00	deg
CVEL	1.00	knots	CDIR	135.00	deg
CVREL	1.00	knots	CDREL	135.00	deg
SIGW	5.80	metres	WVDIR	135.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	14.12	0.59	81.24	0.00	95.95	-95.95
TRANS (tonnes)	-139.64	-15.21	-125.46	0.00	-280.30	280.30
YAW M (tonnes-metres)	5001.51	707.47	1074.43	0.00	6783.41	-6783.41

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)
1	59.12	(59.12)	106.56	3673.65
2	58.65	(58.65)	104.95	3644.62
3	53.72	(53.72)	107.32	3338.25
4	43.52	(43.52)	112.88	2704.41
5	42.83	(42.83)	110.59	2661.50
6	38.94	(38.94)	114.25	2419.73

WVEL	40.00	knots	WDIR	150.00	deg
WVREL	40.00	knots	WDREL	150.00	deg
CVEL	1.00	knots	CDIR	150.00	deg
CVREL	1.00	knots	CDREL	150.00	deg
SIGW	5.80	metres	WVDIR	150.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	16.02	0.73	50.89	0.00	67.64	-67.64
TRANS (tonnes)	-101.68	-11.00	-58.55	0.00	-171.23	171.23
YAW M (tonnes-metres)	4546.83	545.76	538.57	0.00	5631.16	-5631.16

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)
1	39.97	(39.97)	107.62	2483.90
2	39.50	(39.50)	105.33	2454.76
3	34.80	(34.80)	108.90	2162.71
4	25.29	(25.29)	118.76	1571.80
5	24.48	(24.48)	115.08	1521.37
6	21.04	(21.04)	122.40	1307.33

WVEL	40.00	knots	WDIR	165.00	deg
WVREL	40.00	knots	WDREL	165.00	deg
CVEL	1.00	knots	CDIR	165.00	deg
CVREL	1.00	knots	CDREL	165.00	deg
SIGW	5.80	metres	WVDIR	165.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	18.24	0.81	39.97	0.00	59.01	-59.01
TRANS (tonnes)	-49.98	-5.96	-19.46	0.00	-75.40	75.40
YAW M (tonnes-metres)	2803.88	262.77	295.60	0.00	3362.25	-3362.25

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)
1	21.51	(21.51)	118.85	1336.69
2	21.01	(21.01)	116.25	1305.48
3	18.31	(18.31)	122.49	1137.97
4	13.38	(13.38)	141.18	831.42
5	12.48	(12.48)	137.79	775.83
6	11.26	(11.26)	150.82	700.01

WVEL	40.00	knots	WDIR	180.00	deg
WVREL	40.00	knots	WDREL	180.00	deg
CVEL	1.00	knots	CDIR	180.00	deg
CVREL	1.00	knots	CDREL	180.00	deg
SIGW	5.80	metres	WVDIR	180.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	19.59	0.84	31.43	0.00	51.87	-51.87
TRANS (tonnes)	0.00	0.00	0.00	0.00	0.00	0.00
YAW M (tonnes-metres)	0.00	0.00	0.00	0.00	0.00	0.00

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)
1	8.64	(8.64)	180.00	537.17
2	8.64	(8.64)	180.00	537.17
3	8.64	(8.64)	180.00	537.17
4	8.64	(8.64)	180.00	537.17
5	8.64	(8.64)	180.00	537.17
6	8.64	(8.64)	180.00	537.17

WVEL	40.00	knots	WDIR	195.00	deg
WVREL	40.00	knots	WDREL	195.00	deg
CVEL	1.00	knots	CDIR	195.00	deg
CVREL	1.00	knots	CDREL	195.00	deg
SIGW	5.80	metres	WVDIR	195.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	18.24	0.81	39.97	0.00	59.01	-59.01
TRANS (tonnes)	-49.98	-5.96	-19.46	0.00	-75.40	75.40
YAW M (tonnes-metres)	-2803.88	-262.77	-295.60	0.00	-3362.25	3362.25

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)
1	21.01	(21.01)	243.75	1305.46
2	21.51	(21.51)	241.15	1336.70
3	18.31	(18.31)	237.51	1137.97
4	12.48	(12.48)	222.21	775.82
5	13.38	(13.38)	218.82	831.43
6	11.26	(11.26)	209.18	700.01

WVEL	40.00	knots	WDIR	210.00	deg
WVREL	40.00	knots	WDREL	210.00	deg
CVEL	1.00	knots	CDIR	210.00	deg
CVREL	1.00	knots	CDREL	210.00	deg
SIGW	5.80	metres	WVDIR	210.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	16.02	0.73	50.89	0.00	67.64	-67.64
TRANS (tonnes)	101.68	11.00	58.55	0.00	171.23	-171.23
YAW M (tonnes-metres)	-4546.83	-545.76	-538.57	0.00	-5631.16	5631.16

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)
1	39.50	(39.50)	254.67	2454.73
2	39.97	(39.97)	252.38	2483.92
3	34.80	(34.80)	251.10	2162.71
4	24.48	(24.48)	244.92	1521.35
5	25.29	(25.29)	241.24	1571.81
6	21.04	(21.04)	237.60	1307.33

WVEL	40.00	knots	WDIR	225.00	deg
WVREL	40.00	knots	WDREL	225.00	deg
CVEL	1.00	knots	CDIR	225.00	deg
CVREL	1.00	knots	CDREL	225.00	deg
SIGW	5.80	metres	WVDIR	225.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	14.12	0.59	81.24	0.00	95.95	-95.95
TRANS (tonnes)	139.64	15.21	125.46	0.00	280.30	-280.30
YAW M (tonnes-metres)	-5001.51	-707.47	-1074.43	0.00	-6783.41	6783.41

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)
1	58.65	(58.65)	255.05	3644.59
2	59.12	(59.12)	253.44	3673.67
3	53.72	(53.72)	252.68	3338.25
4	42.83	(42.83)	249.41	2661.48
5	43.52	(43.52)	247.12	2704.43
6	38.94	(38.94)	245.75	2419.73

WVEL	40.00	knots	WDIR	180.00	deg
WVREL	40.00	knots	WDREL	180.00	deg
CVEL	1.00	knots	CDIR	180.00	deg
CVREL	1.00	knots	CDREL	180.00	deg
SIGW	5.80	metres	WVDIR	180.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	19.59	0.84	31.43	0.00	51.87	-51.87
TRANS (tonnes)	0.00	0.00	0.00	0.00	0.00	0.00
YAW M (tonnes-metres)	0.00	0.00	0.00	0.00	0.00	0.00

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(deg)	(kW)	
1	8.64	(8.64)	180.00	537.17
2	8.64	(8.64)	180.00	537.17
3	8.64	(8.64)	180.00	537.17
4	8.64	(8.64)	180.00	537.17
5	8.64	(8.64)	180.00	537.17
6	8.64	(8.64)	180.00	537.17

WVEL	40.00	knots	WDIR	195.00	deg
WVREL	40.00	knots	WDREL	195.00	deg
CVEL	1.00	knots	CDIR	195.00	deg
CVREL	1.00	knots	CDREL	195.00	deg
SIGW	5.80	metres	WVDIR	195.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	18.24	0.81	39.97	0.00	59.01	-59.01
TRANS (tonnes)	49.98	5.96	19.46	0.00	75.40	-75.40
YAW M (tonnes-metres)	-2803.88	-262.77	-295.60	0.00	-3362.25	3362.25

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(deg)	(kW)	
1	21.01	(21.01)	243.75	1305.46
2	21.51	(21.51)	241.15	1336.70
3	18.31	(18.31)	237.51	1137.97
4	12.48	(12.48)	222.21	775.82
5	13.38	(13.38)	218.82	831.43
6	11.26	(11.26)	209.18	700.01

WVEL	40.00	knots	WDIR	210.00	deg
WVREL	40.00	knots	WDREL	210.00	deg
CVEL	1.00	knots	CDIR	210.00	deg
CVREL	1.00	knots	CDREL	210.00	deg
SIGW	5.80	metres	WVDIR	210.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	16.02	0.73	50.89	0.00	67.64	-67.64
TRANS (tonnes)	101.68	11.00	58.55	0.00	171.23	-171.23
YAW M (tonnes-metres)	-4546.83	-545.76	-538.57	0.00	-5631.16	5631.16

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(deg)	(kW)	
1	39.50	(39.50)	254.67	2454.73
2	39.97	(39.97)	252.38	2483.92
3	34.80	(34.80)	251.10	2162.71
4	24.48	(24.48)	244.92	1521.35
5	25.29	(25.29)	241.24	1571.81
6	21.04	(21.04)	237.60	1307.33

WVEL	40.00	knots	WDIR	225.00	deg
WVREL	40.00	knots	WDREL	225.00	deg
CVEL	1.00	knots	CDIR	225.00	deg
CVREL	1.00	knots	CDREL	225.00	deg
SIGW	5.80	metres	WVDIR	225.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	14.12	0.59	81.24	0.00	95.95	-95.95
TRANS (tonnes)	139.64	15.21	125.46	0.00	280.30	-280.30
YAW M (tonnes-metres)	-5001.51	-707.47	-1074.43	0.00	-6783.41	6783.41

Thrust	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(deg)	(kW)	
1	58.65	(58.65)	255.05	3644.59
2	59.12	(59.12)	253.44	3673.67
3	53.72	(53.72)	252.68	3338.25
4	42.83	(42.83)	249.41	2661.48
5	43.52	(43.52)	247.12	2704.43
6	38.94	(38.94)	245.75	2419.73

Appendix 7 – Dynamic Position Calculation C

WVEL	40.00	knots	WDIR	240.00	deg
WVREL	40.00	knots	WDREL	240.00	deg
CVEL	1.00	knots	CDIR	240.00	deg
CVREL	1.00	knots	CDREL	240.00	deg
SIGW	5.80	metres	WVDIR	240.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	13.11	0.42	87.36	0.00	100.88	-100.88
TRANS (tonnes)	158.87	18.87	196.56	0.00	374.30	-374.30
YAW M (tonnes-metres)	-4054.26	-687.25	-1182.76	0.00	-5924.28	5924.28

Thruster	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)
1	70.25	(70.25)	256.58	4365.36
2	70.49	(70.49)	255.77	4380.66
3	67.25	(67.25)	255.52	4179.10
4	60.64	(60.64)	254.45	3768.60
5	60.95	(60.95)	253.44	3787.77
6	58.16	(58.16)	253.20	3614.45

WVEL	40.00	knots	WDIR	255.00	deg
WVREL	40.00	knots	WDREL	255.00	deg
CVEL	1.00	knots	CDIR	255.00	deg
CVREL	1.00	knots	CDREL	255.00	deg
SIGW	5.80	metres	WVDIR	255.00	deg
TPER	8.40	sec			

	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED
LONG (tonnes)	6.09	0.22	43.68	0.00	49.99	-49.99
TRANS (tonnes)	168.75	21.07	273.38	0.00	463.20	-463.20
YAW M (tonnes-metres)	-3012.28	-454.80	-456.73	0.00	-3923.80	3923.81

Thruster	Thrust Req	Thrust Req	Direction	Power
(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)
1	77.04	(77.04)	263.75	4787.45
2	77.03	(77.03)	263.83	4786.73
3	77.37	(77.37)	263.82	4807.73
4	78.06	(78.06)	263.83	4851.05
5	78.05	(78.05)	263.92	4850.28
6	78.34	(78.34)	263.90	4868.28

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WVREL	40.00	knots	WDREL	300.00	deg																																																																																																																																																																																																																																																																																																													
CVEL	1.00	knots	CDIR	300.00	deg																																																																																																																																																																																																																																																																																																													
CVREL	1.00	knots	CDREL	300.00	deg																																																																																																																																																																																																																																																																																																													
SIGW	5.80	metres	WVDIR	300.00	deg																																																																																																																																																																																																																																																																																																													
TPER	8.40	sec																																																																																																																																																																																																																																																																																																																
	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED		-----------------------	--------	---------	---------	----------	---------	----------		LONG (tonnes)	-8.37	-0.42	-87.36	0.00	-96.15	96.15		TRANS (tonnes)	164.20	18.69	196.56	0.00	379.45	-379.45		YAW M (tonnes-metres)	909.37	687.25	1182.76	0.00	2779.38	-2779.39		Thruster	Thrust Req	Thrust Req	Direction	Power		----------	------------	------------	-----------	---------		(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)		1	50.89	(50.89)	286.87	3162.73		2	51.68	(51.68)	289.54	3211.54		3	58.79	(58.79)	285.82	3653.57		4	74.38	(74.38)	281.37	4622.15		5	74.96	(74.96)	283.41	4658.47		6	81.23	(81.23)	281.38	5047.58									-------	-------	--------	-------	--------	-----		WVEL	40.00	knots	WDIR	315.00	deg		WVREL	40.00	knots	WDREL	315.00	deg		CVEL	1.00	knots	CDIR	315.00	deg		CVREL	1.00	knots	CDREL	315.00	deg		SIGW	5.80	metres	WVDIR	315.00	deg		TPER	8.40	sec						WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED		-----------------------	---------	---------	---------	----------	---------	----------		LONG (tonnes)	-18.48	-0.59	-81.24	0.00	-100.30	100.30		TRANS (tonnes)	139.12	15.30	125.46	0.00	279.88	-279.88		YAW M (tonnes-metres)	1136.71	636.72	1051.56	0.00	2824.99	-2824.99		Thruster	Thrust Req	Thrust Req	Direction	Power		----------	------------	------------	-----------	---------		(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)		1	37.52	(37.52)	294.62	2331.75		2	38.48	(38.48)	297.56	2391.04		3	44.18	(44.18)	292.24	2745.24		4	57.15	(57.15)	285.78	3551.31		5	57.83	(57.83)	288.02	3593.76		6	63.04	(63.04)	285.38	3917.26			WVEL	40.00	knots	WDIR	330.00	deg																																																								
WVREL	40.00	knots	WDREL	330.00	deg																																																																																																																																																																																																																																																																																																													
CVEL	1.00	knots	CDIR	330.00	deg																																																																																																																																																																																																																																																																																																													
CVREL	1.00	knots	CDREL	330.00	deg																																																																																																																																																																																																																																																																																																													
SIGW	5.80	metres	WVDIR	330.00	deg																																																																																																																																																																																																																																																																																																													
TPER	8.40	sec																																																																																																																																																																																																																																																																																																																
	WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED		-----------------------	---------	---------	--------	----------	---------	----------		LONG (tonnes)	-25.63	-0.73	-50.89	0.00	-77.25	77.25		TRANS (tonnes)	104.77	10.81	58.55	0.00	174.13	-174.13		YAW M (tonnes-metres)	1250.38	505.33	538.57	0.00	2294.28	-2294.28		Thruster	Thrust Req	Thrust Req	Direction	Power		----------	------------	------------	-----------	---------		(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)		1	23.34	(23.34)	301.20	1450.56		2	24.19	(24.19)	304.37	1503.30		3	28.00	(28.00)	297.37	1740.00		4	37.06	(37.06)	288.94	2302.86		5	37.64	(37.64)	291.38	2339.22		6	41.29	(41.29)	288.17	2565.89									-------	-------	--------	-------	--------	-----		WVEL	40.00	knots	WDIR	345.00	deg		WVREL	40.00	knots	WDREL	345.00	deg		CVEL	1.00	knots	CDIR	345.00	deg		CVREL	1.00	knots	CDREL	345.00	deg		SIGW	5.80	metres	WVDIR	345.00	deg		TPER	8.40	sec						WIND	CURRENT	WAVE	EXTERNAL	DEMAND	ACHIEVED		-----------------------	--------	---------	--------	----------	---------	----------		LONG (tonnes)	-25.99	-0.81	-39.97	0.00	-66.77	66.76		TRANS (tonnes)	34.61	5.77	19.46	0.00	59.84	-59.84		YAW M (tonnes-metres)	909.37	256.71	289.99	0.00	1456.06	-1456.06		Thruster	Thrust Req	Thrust Req	Direction	Power		----------	------------	------------	-----------	---------		(tonnes)	(tonnes)	(tonnes)	(deg)	(kW)		1	11.96	(11.96)	333.70	743.27		2	12.69	(12.69)	335.32	788.66		3	13.60	(13.60)	324.88	845.43		4	16.90	(16.90)	309.24	1050.03		5	17.46	(17.46)	311.47	1085.30		6	18.87	(18.87)	306.13	1172.83																																																																

Appendix 8 – Intact Stability Criteria

ISCode2008 / MARPOL / ABS

International Code on Intact Stability 2008

Criteria regarding lever curve properties intact assumptions:

- The area under the righting lever curve from equilibrium to 30 deg. (or down flooding) $> 0.055 \text{ m.rad}$.
- The area under the righting lever curve from equilibrium to 40 deg. (or down flooding) $> 0.090 \text{ m.rad}$.
- The area under the righting lever curve from 30 to 40 deg. (or down flooding) $> 0.030 \text{ m.rad}$.
- The righting lever $GZ > 0.20\text{m}$, for at least 30 deg. angle of heel.
- The maximum GZ occurs to a min angle of heel of 25.00 deg.
- Initial $GM > 0.15 \text{ m}$.

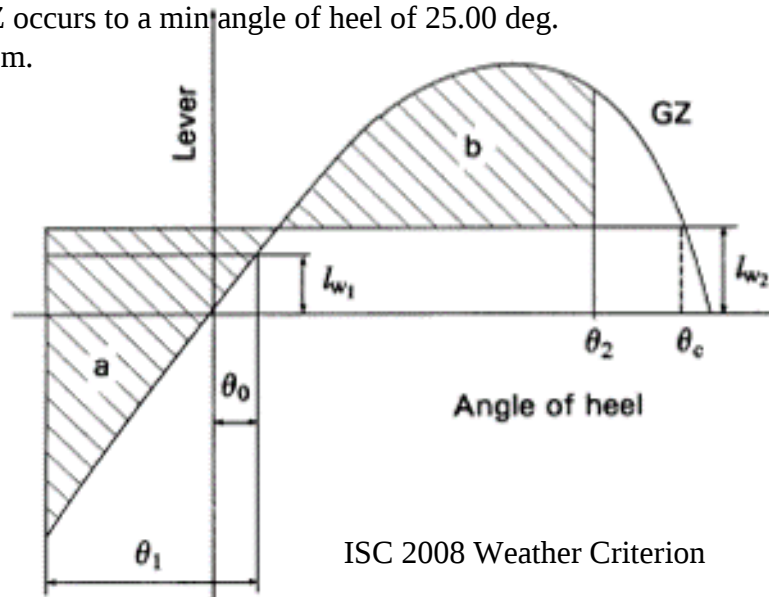
Severe wind and rolling criterion (Weather criterion):

- Angle of heel due to steady wind (pressure of 504 [Pa]) preferably less than 16.00 deg. or 80% of the angle of deck edge immersion, whichever is less.
- Angle of windward roll as per International Code on Intact Stability 2008 2.3.4.
- Stability area (b) should be equal to or greater than the “gust” area (a) (gust wind pressure = 1.5*steady wind pressure).
- Stability area (b) assumed to end at the least of either 50.00 deg or down flooding angle or second intercept with wind.

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Criteria regarding lever curve properties intact assumptions:

- The area under the righting lever curve from equilibrium to 30 deg. (or down flooding) $> 0.055 \text{ m.rad}$.
- The area under the righting lever curve from equilibrium to 40 deg. (or down flooding) $> 0.090 \text{ m.rad}$.
- The area under the righting lever curve from 30 to 40 deg. (or down flooding) $> 0.030 \text{ m.rad}$.
- The righting lever $GZ > 0.20\text{m}$, for at least 30 deg. angle of heel.
- The maximum GZ occurs to a min angle of heel of 25.00 deg.
- Initial $GM > 0.15 \text{ m}$.



Appendix 9 – Deck Margin Line Table

#	x (m)	y (m)	z (m)	#	x (m)	y (m)	z (m)	#	x (m)	y (m)	z (m)
0	0	0	19,7	73	193,9	-17,86	19,7	146	191,8	17,93	19,7
1	0	-16,2	19,21	74	195,3	-17,78	19,7	147	188,3	17,99	19,7
2	0,7	-16,29	19,2	75	196	-17,74	19,7	148	186,9	18	19,7
3	2,1	-16,48	19,18	76	196,7	-17,69	19,7	149	186,2	18	19,7
4	2,8	-16,57	19,17	77	197,4	-17,63	19,7	150	184,8	18	19,7
5	3,5	-16,66	19,16	78	198,8	-17,5	19,7	151	184,1	18	19,7
6	4,2	-16,75	19,15	79	199,5	-17,42	19,7	152	183,4	18	19,7
7	5,6	-16,92	19,13	80	200,2	-17,33	19,7	153	182,7	18	19,7
8	6,3	-17	19,12	81	201,6	-17,13	19,7	154	182	18	19,7
9	7	-17,08	19,11	82	203,7	-16,76	19,7	155	180,6	18	19,42
10	8,4	-17,22	19,09	83	205,1	-16,46	19,7	156	179,9	18	19,28
11	9,8	-17,36	19,07	84	205,8	-16,3	19,7	157	179,2	18	19,14
12	10,5	-17,42	19,07	85	207,9	-15,71	19,7	158	178,64	18	19,03
13	12,6	-17,58	19,05	86	209,3	-15,24	19,7	159	178,5	18	19
14	14	-17,66	19,04	87	210	-14,98	19,7	160	177,8	18	19
15	14,7	-17,7	19,04	88	211,4	-14,41	19,7	161	177,1	18	19
16	16,8	-17,8	19,02	89	212,1	-14,1	19,7	162	176,4	18	19
17	18,9	-17,87	19,02	90	212,8	-13,76	19,7	163	175	18	19
18	21	-17,91	19,01	91	213,5	-13,4	19,7	164	174,3	18	19
19	23,1	-17,95	19,01	92	214,2	-13,02	19,7	165	172,9	18	19
20	25,2	-17,97	19	93	214,9	-12,62	19,7	166	170,8	18	19
21	27,3	-17,98	19	94	215,6	-12,2	19,7	167	168,7	18	19
22	28,7	-17,99	19	95	216,3	-11,77	19,7	168	168	18	19
23	30,8	-17,99	19	96	217	-11,32	19,7	169	166,6	18	19
24	32,9	-18	19	97	217,7	-10,85	19,7	170	164,5	18	19
25	35	-18	19	98	218,4	-10,37	19,7	171	162,4	18	19
26	39,2	-18	19	99	219,1	-9,87	19,7	172	161	18	19
27	41,3	-18	19	100	219,8	-9,35	19,7	173	160,3	18	19
28	42	-18	19	101	220,5	-8,81	19,7	174	158,2	18	19
29	43,4	-18	19	102	221,9	-7,65	19,7	175	156,1	18	19
30	45,5	-18	19	103	222,6	-7,01	19,7	176	154	18	19
31	47,6	-18	19	104	223,3	-6,32	19,7	177	151,9	18	19
32	49	-18	19	105	224	-5,56	19,7	178	149,8	18	19
33	49,7	-18	19	106	225,4	-3,72	19,7	179	147,7	18	19
34	51,8	-18	19	107	226,8	-0,85	19,7	180	55,3	18	19
35	53,9	-18	19	108	227,03	0	19,7	181	53,9	18	19
36	55,3	-18	19	109	226,8	0,85	19,7	182	51,8	18	19
37	147,7	-18	19	110	225,4	3,72	19,7	183	49,7	18	19
38	149,8	-18	19	111	224	5,56	19,7	184	49	18	19
39	151,9	-18	19	112	223,3	6,32	19,7	185	47,6	18	19
40	154	-18	19	113	222,6	7,01	19,7	186	45,5	18	19
41	156,1	-18	19	114	221,9	7,65	19,7	187	43,4	18	19
42	158,2	-18	19	115	220,5	8,81	19,7	188	42	18	19
43	160,3	-18	19	116	219,8	9,35	19,7	189	41,3	18	19
44	161	-18	19	117	219,1	9,87	19,7	190	39,2	18	19
45	162,4	-18	19	118	218,4	10,37	19,7	191	35	18	19
46	164,5	-18	19	119	217,7	10,85	19,7	192	32,9	18	19
47	166,6	-18	19	120	217	11,32	19,7	193	30,8	17,99	19
48	168	-18	19	121	216,3	11,77	19,7	194	28,7	17,99	19
49	168,7	-18	19	122	215,6	12,2	19,7	195	27,3	17,98	19
50	170,8	-18	19	123	214,9	12,62	19,7	196	25,2	17,97	19
51	172,9	-18	19	124	214,2	13,02	19,7	197	23,1	17,95	19,01
52	174,3	-18	19	125	213,5	13,4	19,7	198	21	17,91	19,01
53	175	-18	19	126	212,8	13,76	19,7	199	18,9	17,87	19,02
54	176,4	-18	19	127	212,1	14,1	19,7	200	16,8	17,8	19,02
55	177,1	-18	19	128	211,4	14,41	19,7	201	14,7	17,7	19,04
56	177,8	-18	19	129	210	14,98	19,7	202	14	17,66	19,04
57	178,5	-18	19	130	209,3	15,24	19,7	203	12,6	17,58	19,05
58	178,64	-18	19,03	131	207,9	15,71	19,7	204	10,5	17,42	19,07
59	179,2	-18	19,14	132	205,8	16,3	19,7	205	9,8	17,36	19,07
60	179,9	-18	19,28	133	205,1	16,46	19,7	206	8,4	17,22	19,09
61	180,6	-18	19,42	134	203,7	16,76	19,7	207	7	17,08	19,11
62	182	-18	19,7	135	201,6	17,13	19,7	208	6,3	17	19,12
63	182,7	-18	19,7	136	200,2	17,33	19,7	209	5,6	16,92	19,13
64	183,4	-18	19,7	137	199,5	17,42	19,7	210	4,2	16,75	19,15
65	184,1	-18	19,7	138	198,8	17,5	19,7	211	3,5	16,66	19,16
66	184,8	-18	19,7	139	197,4	17,63	19,7	212	2,8	16,57	19,17
67	186,2	-18	19,7	140	196,7	17,69	19,7	213	2,1	16,48	19,18
68	186,9	-18	19,7	141	196	17,74	19,7	214	0,7	16,29	19,2
69	188,3	-17,99	19,7	142	195,3	17,78	19,7	215	0	16,2	19,21
70	191,8	-17,93	19,7	143	193,9	17,86	19,7	216	0	0	19,7
71	192,5	-17,91	19,7	144	193,2	17,89	19,7				
72	193,2	-17,89	19,7	145	192,5	17,91	19,7				

Appendix 10 – Cross Curve Stability

Draft (m)	Heel (deg)	Trim (m)	Displ (t)	KN (m)	GM (m)	Deck (m)
2	0	0	11602,85	0	54,443	17
	10	0,239	11599,58	8,372	28,289	13,751
	20	0,979	11600,88	11,244	9,995	10,883
	30	1,78	11599,67	12,326	3,804	8,178
	40	2,46	11599,66	12,647	0,507	5,558
	50	2,939	11599,64	12,52	-1,691	3,064
	60	3,089	11599,64	12,13	-2,538	0,825
4	0	0	24168,41	0	29,293	15
	10	0,147	24164,04	5,185	29,188	11,656
	20	0,705	24164,38	9,245	15,864	8,211
	30	1,483	24164,85	11,242	8,637	5,13
	40	2,202	24167,38	12,363	4,791	2,336
	50	2,79	24164,62	12,971	2,605	-0,138
	60	3,344	24164,34	13,157	-0,59	-2,352
6	0	0	37456,63	0	21,874	13
	10	0,162	37451,52	3,846	22,089	9,67
	20	0,576	37452,12	7,699	20,415	6,083
	30	1,252	37452,1	10,46	12,363	2,73
	40	1,954	37452,05	12,188	8,246	-0,167
	50	2,67	37452,19	13,169	3,073	-2,768
	60	3,358	37452,4	13,302	-1,114	-5,224
8	0	0	51413,4	0	18,418	11
	10	0,08	51407,48	3,246	18,746	7,711
	20	0,355	51408,41	6,588	19,518	4,198
	30	0,81	51408,22	9,777	15,66	0,721
	40	1,4	51411,84	11,899	8,38	-2,359
	50	1,989	51408,68	12,844	3,071	-5,257
	60	2,434	51408,88	13,033	-0,711	-7,945
10	0	0	65701,72	0	16,771	9
	10	0,039	65695,26	2,948	16,989	5,751
	20	0,146	65699,03	5,983	17,751	2,355
	30	0,351	65696,48	9,121	16,263	-1,073
	40	0,63	65696,36	11,211	8,372	-4,489
	50	0,853	65696,42	12,209	3,392	-7,718
	60	1,024	65694,17	12,469	-0,269	-10,709
12	0	0	80241,94	0	15,99	7
	10	-0,003	80233,64	2,807	16,138	3,787
	20	-0,001	80235,99	5,673	16,73	0,497
	30	-0,045	80236,19	8,347	13,208	-2,973
	40	-0,2	80236,08	10,249	8,228	-6,785
	50	-0,419	80231,36	11,288	3,992	-10,54
	60	-0,626	80235,93	11,711	0,956	-13,991
14	0	0	94814,5	0	15,698	5
	10	-0,028	94807,02	2,75	15,799	1,805
	20	-0,14	94808,16	5,46	13,992	-1,502
	30	-0,444	94808,14	7,576	10,423	-5,321
	40	-0,932	94808,03	9,148	7,743	-9,577
	50	-1,456	94807,99	10,256	4,91	-13,857
	60	-1,922	94805,03	10,869	2,235	-17,768

Appendix 11 – Deadweight Scale

Summer Freeboard=	12	metres	Lightship weight=	29925	tonnes
Freeboard Depth =	19,7	metres	Shell thickness =	17	mm
			Water density =	1,025	tonnes/cu.m

Draft	Displ Salt water	Displ Fresh water	Deadweight Salt water	Deadweight Fresh water	Free- board	TPI	MCT
(m)	(t)	(t)	(t)	(t)	(m)	(t/cm)	(t-m/cm)
5	30717,4	29968,1	792,35	43,14	14,7	66,215	974,65
5,1	31381,3	30615,9	1456,26	690,86	14,6	66,359	980,44
5,2	32046,3	31264,7	2121,27	1339,65	14,5	66,504	986,3
5,3	32713	31915,1	2787,97	1990,1	14,4	66,662	992,85
5,4	33383	32568,8	3457,98	2643,76	14,3	67,038	1010,83
5,5	34055	33224,4	4130	3299,39	14,2	67,187	1016,97
5,6	34729,3	33882,3	4804,31	3957,26	14,1	67,334	1023,04
5,7	35404,7	34541,2	5479,74	4616,21	14	67,481	1029,08
5,8	36081,9	35201,8	6156,88	5276,84	13,9	67,594	1033,45
5,9	36763,2	35866,6	6838,23	5941,56	13,8	68,12	1038,67
6	37445,9	36532,5	7520,85	6607,54	13,7	68,269	1064,88
6,1	38130,3	37200,3	8205,32	7275,31	13,6	68,419	1071,13
6,2	38815,9	37869,2	8890,93	7944,2	13,5	68,568	1077,37
6,3	39503,3	38539,8	9578,27	8614,78	13,4	68,715	1083,58
6,4	40193,7	39213,4	10268,69	9288,35	13,3	68,862	1089,79
6,5	40883,8	39886,6	10958,8	9961,63	13,2	69,01	1096,06
6,6	41575,5	40561,4	11650,48	10636,44	13,1	69,159	1102,39
6,7	42270	41239	12345,02	11314,04	13	69,308	1108,76
6,8	42964,5	41916,6	13039,52	11991,61	12,9	69,453	1114,93
6,9	43660,9	42596	13735,87	12670,98	12,8	69,589	1120,75
7	44358,8	43276,8	14433,76	13351,84	12,7	69,716	1126,16
7,1	45057,4	43958,5	15132,44	14033,48	12,6	69,836	1131,26
7,2	45757	44640,9	15831,96	14715,94	12,5	69,951	1136,18
7,3	46458,9	45325,7	16533,89	15400,74	12,4	70,061	1140,86
7,4	47159,4	46009,1	17234,37	16084,14	12,3	70,17	1145,52
7,5	47863,5	46696,1	17938,51	16771,11	12,2	70,278	1150,13
7,6	48570,1	47385,4	18645,07	17460,44	12,1	70,383	1154,53
7,7	49283,8	48081,8	19358,81	18156,76	12	70,483	1158,67
7,8	49989,8	48770,5	20064,78	18845,52	11,9	70,586	1162,72
7,9	50702,2	49465,5	20777,17	19540,53	11,8	70,675	1166,33

8	51410,22	50156,31	21485,22	20231,31	11,7	70,759	1169,66
8,1	52119,07	50847,87	22194,07	20922,87	11,6	70,836	1172,69
8,2	52828,96	51540,45	22903,96	21615,45	11,5	70,907	1175,44
8,3	53539,4	52233,56	23614,4	22308,56	11,4	70,973	1177,93
8,4	54249,96	52926,79	24324,96	23001,79	11,3	71,034	1180,21
8,5	54961,44	53620,92	25036,44	23695,92	11,2	71,089	1182,23
8,6	55673,5	54315,61	25748,5	24390,61	11,1	71,139	1183,91
8,7	56385,66	55010,4	26460,66	25085,4	11	71,185	1185,4
8,8	57098,36	55705,71	27173,36	25780,71	10,9	71,227	1186,75
8,9	57811,46	56401,42	27886,46	26476,42	10,8	71,26	1187,54
9	58525,14	57097,69	28600,14	27172,69	10,7	71,29	1188,17
9,1	59239,38	57794,51	29314,38	27869,51	10,6	71,285	1186,67
9,2	59953,25	58490,98	30028,25	28565,98	10,5	71,35	1189,39
9,3	60675,08	59195,2	30750,08	29270,2	10,4	71,42	1192,33
9,4	61390,21	59892,88	31465,21	29967,88	10,3	71,5	1195,76
9,5	62106,23	60591,44	32181,23	30666,44	10,2	71,579	1199,2
9,6	62822,97	61290,7	32897,97	31365,7	10,1	71,659	1202,66
9,7	63540,87	61991,09	33615,87	32066,09	10	71,734	1205,97
9,8	64259,11	62691,81	34334,11	32766,81	9,9	71,81	1209,29
9,9	64978,17	63393,33	35053,17	33468,33	9,8	71,886	1212,61
10	65698	64095,6	35773	34170,6	9,7	71,963	1215,96
10,1	66422,18	64802,13	36497,18	34877,13	9,6	72,03	1218,89
10,2	67143,38	65505,74	37218,38	35580,74	9,5	72,096	1221,86
10,3	67865,17	66209,93	37940,17	36284,93	9,4	72,162	1224,83
10,4	68587,63	66914,76	38662,63	36989,76	9,3	72,228	1227,78
10,5	69310,74	67620,24	39385,74	37695,24	9,2	72,294	1230,71
10,6	70034,55	68326,39	40109,55	38401,39	9,1	72,358	1233,63
10,7	70758,96	69033,13	40833,96	39108,13	9	72,423	1236,54
10,8	71484,05	69740,54	41559,05	39815,54	8,9	72,487	1239,46
10,9	72209,78	70448,57	42284,78	40523,57	8,8	72,551	1242,36
11	72936,11	71157,19	43011,11	41232,19	8,7	72,615	1245,25
11,1	73663,08	71866,42	43738,08	41941,42	8,6	72,678	1248,12
11,2	74390,67	72576,26	44465,67	42651,26	8,5	72,74	1250,97
11,3	75119,11	73286,94	45194,11	43361,94	8,4	72,802	1253,78
11,4	75849,64	73999,65	45924,64	44074,65	8,3	72,858	1256,36
11,5	76580,02	74712,22	46655,02	44787,22	8,2	72,912	1258,81
11,6	77310,38	75424,76	47385,38	45499,76	8,1	72,964	1261,23
11,7	78040,78	76137,35	48115,78	46212,35	8	73,017	1263,67
11,8	78771,71	76850,45	48846,71	46925,45	7,9	73,07	1266,11

Draft	Displ Salt water	Displ Fresh water	Deadweight Salt water	Deadweight Fresh water	Free- board	TPI	MCT
(m)	(t)	(t)	(t)	(t)	(m)	(t/cm)	(t-m/cm)
11,9	79503,15	77564,05	49578,15	47639,05	7,8	73,122	1268,55
12	80235,04	78278,09	50310,04	48353,09	7,7	73,175	1271
12,1	80967,55	78992,73	51042,55	49067,73	7,6	73,228	1273,46
12,2	81700,44	79707,75	51775,44	49782,75	7,5	73,28	1275,92
12,3	82434,01	80423,42	52509,01	50498,42	7,4	73,333	1278,39
12,4	83168,01	81139,52	53243,01	51214,52	7,3	73,386	1280,85
12,5	83904,78	81858,33	53979,78	51933,33	7,2	73,438	1283,31
12,6	84627,19	82563,11	54702,19	52638,11	7,1	72,225	1284,42
12,7	85350,24	83268,53	55425,24	53343,53	7	72,278	1286,9
12,8	86073,71	83974,35	56148,71	54049,35	6,9	72,331	1289,39
12,9	86798,44	84681,4	56873,44	54756,4	6,8	72,385	1291,93
13	87523,01	85388,3	57598,01	55463,3	6,7	72,44	1294,53
13,1	88260,43	86107,74	58335,43	56182,74	6,6	72,491	1296,92
13,2	88986,13	86815,74	59061,13	56890,74	6,5	72,542	1299,35
13,3	89712,35	87524,25	59787,35	57599,25	6,4	72,594	1301,79
13,4	90439,24	88233,4	60514,24	58308,4	6,3	72,646	1304,23
13,5	91166,49	88942,92	61241,49	59017,92	6,2	72,697	1306,68
13,6	91894,26	89652,94	61969,26	59727,94	6,1	72,749	1309,13
13,7	92622,55	90363,46	62697,55	60438,46	6	72,8	1311,58
13,8	93351,56	91074,69	63426,56	61149,69	5,9	72,851	1314,01
13,9	94081,09	91786,43	64156,09	61861,43	5,8	72,899	1316,29
14	94810,88	92498,42	64885,88	62573,42	5,7	72,948	1318,6
14,1	95544,47	93214,12	65619,47	63289,12	5,6	72,996	1320,9
14,2	96275,18	93927,01	66350,18	64002,01	5,5	73,04	1323,09
14,3	97006,34	94640,34	67081,34	64715,34	5,4	73,084	1325,28
14,4	97737,95	95354,1	67812,95	65429,1	5,3	73,129	1327,49
14,5	98470,05	96068,34	68545,05	66143,34	5,2	73,174	1329,71
14,6	99202,58	96783,01	69277,58	66858,01	5,1	73,219	1331,91
14,7	99935,88	97498,42	70010,88	67573,42	5	73,262	1334,04
14,8	100669,2	98213,9	70744,24	68288,9	4,9	73,304	1336,16
14,9	101403	98929,79	71478,04	69004,79	4,8	73,347	1338,28
15	102137,3	99646,1	72212,25	69721,1	4,7	73,389	1340,4
15,1	102872,5	100363,4	72947,47	70438,38	4,6	73,432	1342,52
15,2	103607,5	101080,5	73682,54	71155,52	4,5	73,474	1344,62
15,3	104343	101798,1	74418,02	71873,07	4,4	73,515	1346,71
15,4	105078,9	102516	75153,93	72591,03	4,3	73,557	1348,8
15,7	107289,4	104672,6	77364,43	74747,62	4	73,681	1355,03
15,8	108027,2	105392,4	78102,23	75467,42	3,9	73,722	1357,1
15,9	108765,2	106112,4	78840,2	76187,39	3,8	73,764	1359,19
16	109503,6	106832,8	79578,59	76907,77	3,7	73,805	1361,27
16,1	110242,5	107553,6	80317,46	77628,61	3,6	73,847	1363,37
16,2	110981,7	108274,8	81056,67	78349,8	3,5	73,888	1365,48
16,3	111721,3	108996,4	81796,31	79071,4	3,4	73,93	1367,59
16,4	112461,9	109718,9	82536,9	79793,93	3,3	73,972	1369,71
16,5	113202,4	110441,3	83277,38	80516,34	3,2	74,014	1371,82
16,6	113943,4	111164,3	84018,4	81239,29	3,1	74,055	1373,91
16,7	114684,7	111887,5	84759,7	81962,52	3	74,097	1376
16,8	115426,4	112611,1	85501,42	82686,14	2,9	74,138	1378,11
16,9	116172,1	113338,7	86247,14	83413,68	2,8	74,179	1380,2
17	116916,9	114065,3	86991,94	84140,31	2,7	74,218	1382,22
17,1	117659,9	114790,1	87734,87	84865,11	2,6	74,257	1384,25
17,2	118403,2	115515,3	88478,18	85590,3	2,5	74,297	1386,28
17,3	119146,9	116240,9	89221,89	86315,87	2,4	74,336	1388,31
17,4	119891	116966,8	89965,99	87041,82	2,3	74,375	1390,36
17,5	120635,5	117693,2	90710,49	87768,16	2,2	74,415	1392,41
17,6	121380,4	118419,9	91455,38	88494,89	2,1	74,454	1394,47
17,7	122125,7	119147	92200,67	89222	2	74,494	1396,53
17,8	122871,4	119874,5	92946,36	89949,5	1,9	74,534	1398,6
17,9	123617,4	120602,4	93692,44	90677,38	1,8	74,574	1400,67
18	124363,9	121330,7	94438,92	91405,66	1,7	74,613	1402,73
18,1	125110,8	122059,3	95185,8	92134,32	1,6	74,652	1404,76
18,2	125862,3	122792,5	95937,34	92867,53	1,5	74,691	1406,77
18,3	126610,5	123522,4	96685,5	93597,44	1,4	74,726	1408,61
18,4	127358,5	124252,2	97433,48	94327,17	1,3	74,76	1410,36
18,5	128106,8	124982,3	98181,81	95057,26	1,2	74,794	1412,13
18,6	128855,6	125712,7	98930,55	95787,73	1,1	74,827	1413,89
18,7	129604,5	126443,5	99679,53	96518,45	1	74,861	1415,66
18,8	130353,9	127174,5	100428,6	97249,5	0,9	74,895	1417,43
18,9	131103,5	127905,9	101178,53	97980,88	0,8	74,929	1419,2

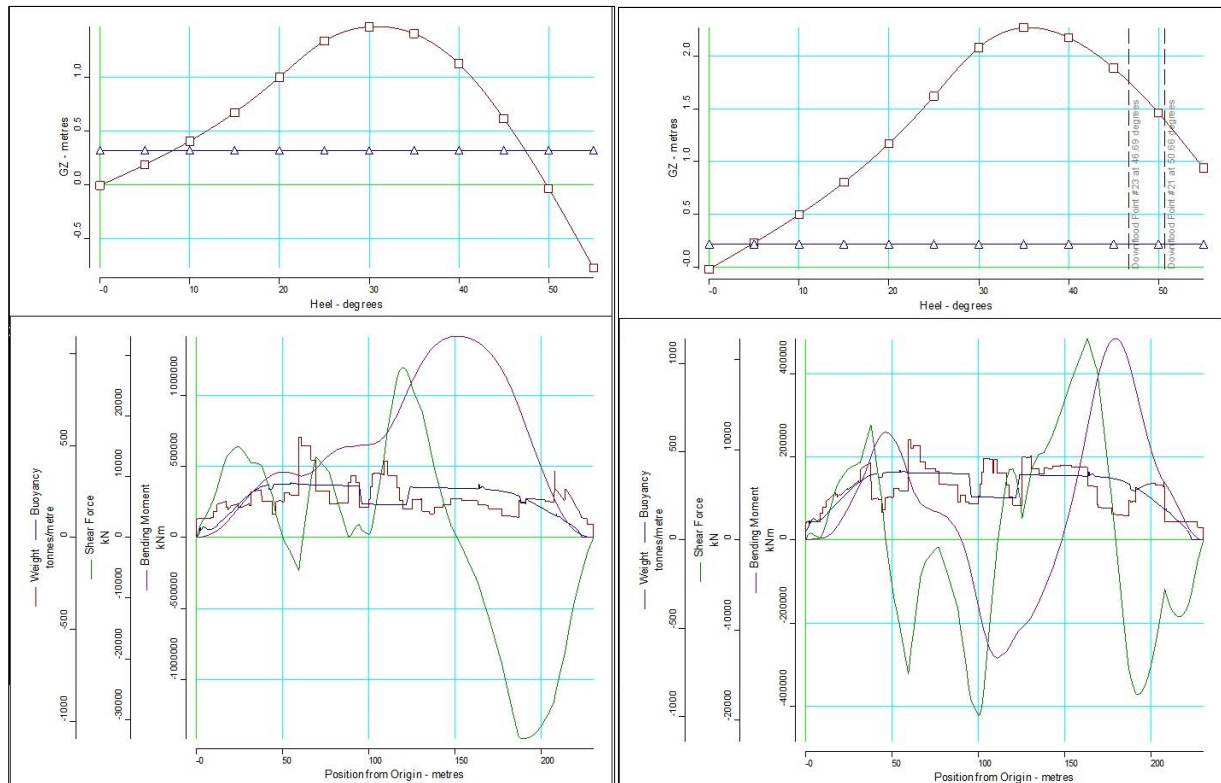
Appendix 12 – Intact Stability Results for all Conditions A

Property	Docking		Transit with casing %10		Transit with casing %100		Transit without casing %10		Transit without casing %100		Drilling 10%		Drilling 100%			
	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme		
Draft at LCF [m]	7.211	7.241	9.412	9.442	9.721	9.751	7.232	7.262	8.005	8.035	7.561	7.591	10.154	10.184		
Draft aft at marks [m]	7.258	7.288	9.013	9.043	9.516	9.546	7.339	7.369	7.874	7.904	8.229	8.259	10.686	10.716		
Draft fwd at marks [m]	7.163	7.193	9.843	9.873	9.943	9.973	7.12	7.15	8.144	8.174	6.847	6.877	9.571	9.601		
Draft at AP [m]	7.258	7.288	9.013	9.043	9.516	9.546	7.339	7.369	7.874	7.904	8.229	8.259	10.686	10.716		
Draft at FP [m]	7.163	7.193	9.843	9.873	9.943	9.973	7.12	7.15	8.144	8.174	6.847	6.877	9.571	9.601		
Mean draft at midships [m]	7.21	7.24	9.428	9.458	9.73	9.76	7.229	7.259	8.009	8.039	7.538	7.568	10.128	10.158		
Criterion	Docking		Transit with casing %10		Transit with casing %100		Transit without casing %10		Transit without casing %100		Drilling 10%		Drilling 100%			
	Actual Value	Critical Value	Actual Value	Critical Value	Actual Value	Critical Value	Actual Value	Critical Value	Actual Value	Critical Value	Actual Value	Critical Value	Actual Value	Critical Value		
Area under GZ curve up to 30 degrees > 0.055	0.498	0.055	0.486	0.055	0.43	0.055	0.514	0.055	0.449	0.055	0.378	0.055	0.466	0.055		
Area under GZ curve from 30 to 40 deg. or downflood > 0.03	0.301	0.03	0.41	0.03	0.37	0.03	0.321	0.03	0.337	0.03	0.239	0.03	0.388	0.03		
Area under GZ curve up to 40 deg. or downflood > 0.09	0.799	0.09	0.896	0.09	0.8	0.09	0.835	0.09	0.785	0.09	0.617	0.09	0.854	0.09		
Initial GM to be at least 0.15 metres	3.124	0.15	3.012	0.15	2.606	0.15	3.367	0.15	2.854	0.15	2.077	0.15	2.798	0.15		
GZ to be at least 0.20m at an angle > 30 degrees	1.827	0.2	2.402	0.2	2.167	0.2	1.921	0.2	1.979	0.2	1.469	0.2	2.273	0.2		
Max GZ to be at an angle > 30 degrees	30.44	30	35.526	30	35.13	30	30.937	30	34.314	30	30.617	30	35.466	30		
IMO Weather Criterion (Maximum Initial Angle Of Heel)	4.331	16	3.846	16	4.259	16	4.455	16	5.145	16	5.665	16	3.345	16		
IMO Weather Criterion (Areas)	35.715	1	75.266	1	60.571	1	36.64	1	35.493	1	19.987	1	92.294	1		
Property	Start Running Casing 10%		Start Running Casing 100%		Running Casing 10%		Running Casing 100%		Stand by 10%		Stand by 100%		In-field Survival 10%		In-field Survival 100%	
	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme
Draft at LCF [m]	9.501	9.531	10.508	10.538	9.209	9.239	8.707	8.737	7.604	7.634	10.338	10.368	10.13	10.16	11.155	11.185
Draft aft at marks [m]	10.137	10.167	10.232	10.262	10.048	10.078	9.134	9.164	8.953	8.983	10.349	10.379	10.68	10.71	10.873	10.903
Draft fwd at marks [m]	8.806	8.836	10.808	10.838	8.294	8.324	8.246	8.276	6.146	6.176	10.327	10.357	9.526	9.556	11.461	11.491
Draft at AP [m]	10.137	10.167	10.232	10.262	10.048	10.078	9.134	9.164	8.953	8.983	10.349	10.379	10.68	10.71	10.873	10.903
Draft at FP [m]	8.806	8.836	10.808	10.838	8.294	8.324	8.246	8.276	6.146	6.176	10.327	10.357	9.526	9.556	11.461	11.491
Mean draft at midships [m]	9.472	9.502	10.52	10.55	9.171	9.201	8.69	8.72	7.55	7.58	10.338	10.368	10.103	10.133	11.167	11.197
Criterion	Start Running Casing 10%		Start Running Casing 100%		Running Casing 10%		Running Casing 100%		Stand by 10%		Stand by 100%		In-field Survival 10%		In-field Survival 100%	
	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme	Moulded	Extreme
Area under GZ curve up to 30 degrees > 0.055	0.409	0.055	0.413	0.055	0.594	0.055	0.472	0.055	0.433	0.055	0.491	0.055	0.459	0.055	0.466	0.055
Area under GZ curve from 30 to 40 deg. or downflood > 0.03	0.351	0.03	0.343	0.03	0.476	0.03	0.379	0.03	0.284	0.03	0.41	0.03	0.386	0.03	0.364	0.03
Area under GZ curve up to 40 deg. or downflood > 0.09	0.76	0.09	0.756	0.09	1.07	0.09	0.851	0.09	0.717	0.09	0.901	0.09	0.845	0.09	0.83	0.09
Initial GM to be at least 0.15 metres	2.363	0.15	2.394	0.15	3.595	0.15	2.783	0.15	2.615	0.15	3.113	0.15	2.802	0.15	2.844	0.15
GZ to be at least 0.20m at an angle > 30 degrees	2.063	0.2	2.007	0.2	2.787	0.2	2.228	0.2	1.703	0.2	2.403	0.2	2.261	0.2	2.128	0.2
Max GZ to be at an angle > 30 degrees	34.712	30	35.146	30	35.984	30	34.831	30	31.844	30	36.081	30	35.45	30	36	30
IMO Weather Criterion (Maximum Initial Angle Of Heel)	4.583	16	3.593	16	2.578	16	3.989	16	5.058	16	3.436	16	3.596	16	2.781	16
IMO Weather Criterion (Areas)	54.356	1	71.269	1	153.952	1	64.219	1	28.289	1	87.866	1	82.612	1	102.345	1

Appendix 12 – Intact Stability Results for all Conditions B

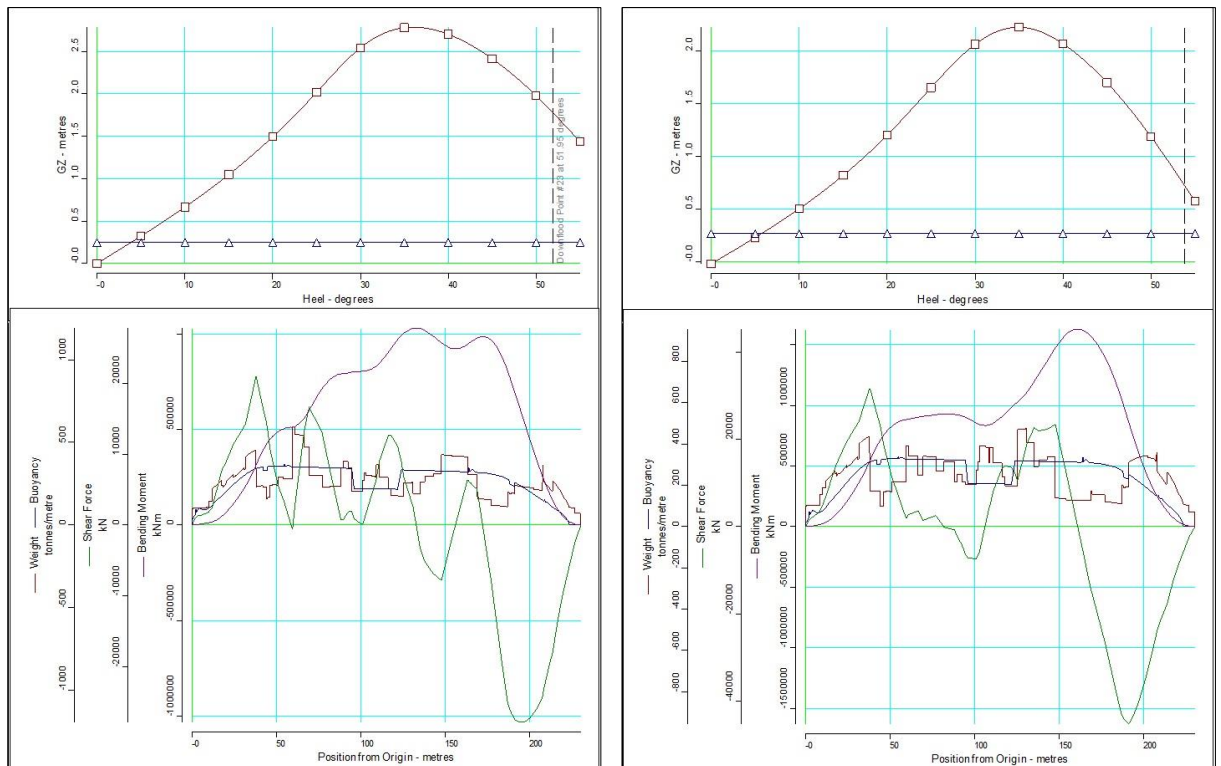
	Docking	Transit with casing %10	Transit with casing %100	Transit without casing %10	Transit without casing %100	Drilling 10%	Drilling 100%	Start Running Casing 10%	Start Running Casing 100%	Running Casing 10%	Running Casing 100%	Stand by 10%	Stand by 100%	In-field Survival 10%	In-field Survival 100%
Hydrostatics at equilibrium angle	Density of water [tonnes/cu m]	1.025	1.025	1.025	1.025	1.025	1.025	1.025	1.025	1.025	1.025	1.025	1.025	1.025	1.025
	Heel to starboard [degrees]	0.37	0.88	0.98	0.75	0.22	0.46	0.94	0.42	No heel	0.46	0.61	0.89	0.7	0.29
	Trim by the stern [m]	0.095	0.83	0.427	0.219	1.381	1.116	1.331	0.576	1.754	0.888	2.807	No trim	1.155	0.587
	KG [m]	16.276	14.05	14.343	16.041	16.832	13.871	14.628	14.047	13.629	14.898	16.413	13.463	13.862	13.362
	FSC [m]	0.123	0.097	0.024	0.129	0.16	0.065	0.124	0.062	0.128	0.077	0.159	0.064	0.098	0.037
	KGF [m]	16.399	14.146	14.367	16.17	16.991	13.936	14.752	14.109	13.757	14.975	16.572	13.526	13.96	13.399
	GMe [m]	3.124	3.012	2.606	3.367	2.854	2.1	2.418	2.418	3.595	2.783	2.615	3.113	2.802	2.859
	BMe [m]	15.735	12.192	11.845	15.703	14.232	15.125	12.153	11.015	12.534	13.181	15.135	11.196	11.43	10.415
	BML [m]	547.875	428.192	417.415	547.886	499.816	536.383	403.13	391.59	446.541	467.243	539.68	396.932	404.024	372.504
	Waterplane area [m2]	6829.91	6970.58	6997.47	6837.88	6907.04	7031.61	7010.11	7054.27	7007.3	6963.86	6936.7	7043.42	7030.53	7095.6
	LCG [m]	113.954	113.844	112.825	113.63	110.581	109.727	109.657	112.51	108.954	110.979	107.15	111.573	109.67	112.107
	LCB [m]	113.949	113.879	112.843	113.617	110.5	109.684	109.599	112.532	108.883	110.938	106.99	111.572	109.624	112.127
	TCB [m]	-0.101	-0.188	-0.203	-0.207	0.058	-0.092	0.2	0.082	0	-0.107	-0.161	-0.175	-0.14	0.054
	LCF [m]	107.709	106.168	105.878	107.592	106.547	105.119	105.352	105.874	105.42	106.022	105.98	105.584	105.105	105.91
IMO Wind Heeling	TCF [m]	-0.057	-0.174	-0.2	-0.119	0.036	-0.096	0.18	0.09	0	-0.085	-0.098	-0.186	-0.145	0.066
	TPC [t/cm]	70.007	71.448	71.724	70.088	70.797	72.074	71.854	72.306	71.825	71.38	71.101	72.195	72.063	72.73
	MTC [tonnes-m/cm]	1138.717	1193.17	1205.168	1142.361	1174.248	1220.812	1213.761	1231.379	1214.81	1195.324	1188	1226.098	1220.261	1250.673
	Shell thickness [mm]	17	17	17	17	17	17	17	17	17	17	17	17	17	17
	Docking	Value	Value	Value	Value	Value	Value	Value	Value	Value	Value	Value	Value	Value	Value
	Transit with casing %10	227.227	227.227	227.227	227.227	227.227	227.227	227.227	227.227	227.227	227.227	227.23	227.227	227.227	227.227
	Transit with casing %100	7547.454	7053.972	6987.448	7543.37	7476.179	6899.079	7044.04	6813.912	7111.641	7217.966	7475.6	6853.676	6904.666	6671.526
	Area to leeward (Area b) [m-radians]	0.70395	0.99823	0.85289	0.76795	0.47487	0.92092	0.83943	0.72113	1.28093	0.92762	0.6192	0.93596	0.91438	0.73445
	Area to windward (Area a) [m-radians]	0.01971	0.01326	0.01408	0.02096	0.02376	0.00998	0.01544	0.01012	0.00832	0.01444	0.0219	0.01065	0.01107	0.00718
	GZc [m]	0.336	0.238	0.228	0.335	0.294	0.216	0.236	0.205	0.246	0.264	0.316	0.21	0.216	0.189
	Gust angle [degrees]	6.27	5.315	5.876	6.271	6.996	8.12	4.771	5.162	3.842	5.726	7.166	4.7	5.033	4.008
	Rollback angle [degrees]	22.369	19.303	18.276	22.782	20.567	20.184	18.381	17.735	20.504	19.67	20.981	18.777	18.381	18.997
	Steady state angle [degrees]	4.331	3.846	4.259	4.455	5.145	5.665	4.583	3.593	2.578	3.989	5.058	3.436	3.596	2.781
	Max. angle to leeward [degrees]	50	48.075	47.114	50	47.395	46.691	50	43.516	50	50	49.839	44.885	46.832	40.921

Appendix 13 – GZ Curves and SFBM Diagrams for all Conditions A



Drilling %10

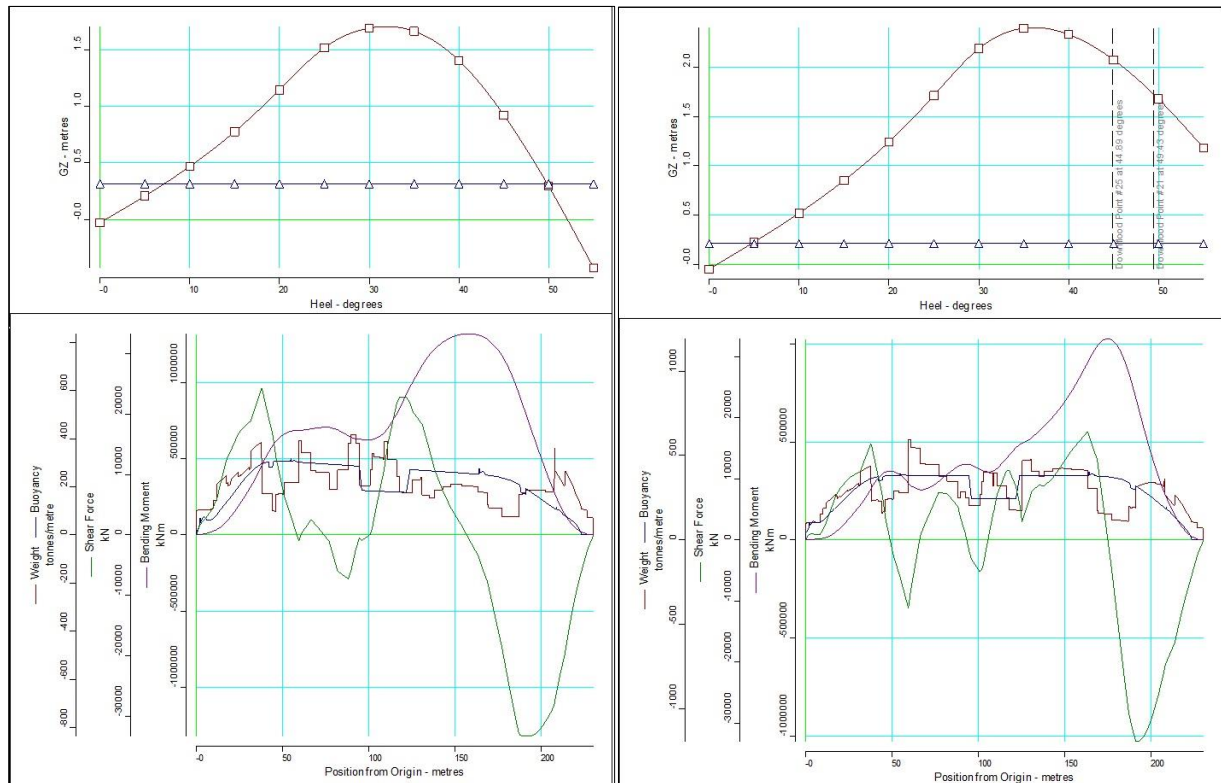
Drilling %100



Running casing %10

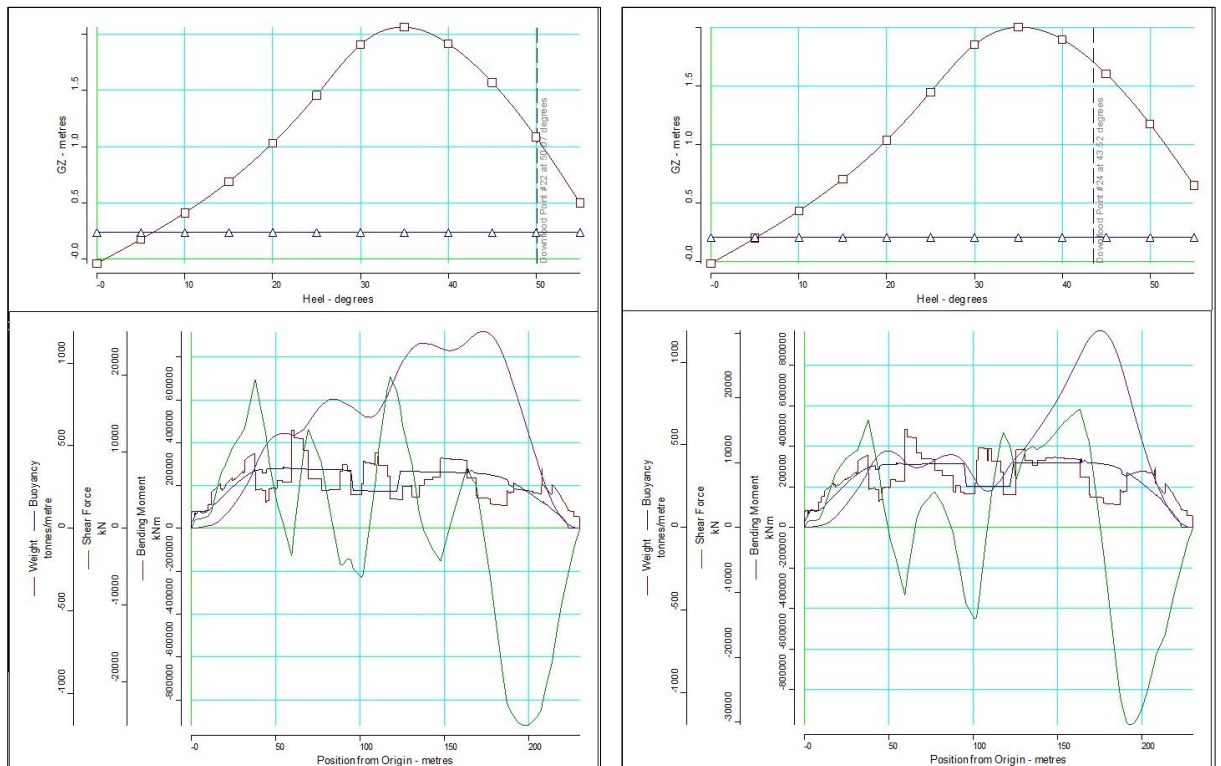
Running casing %100

Appendix 13 – GZ Curves and SFBM Diagrams for all Conditions B



Stand - by %10

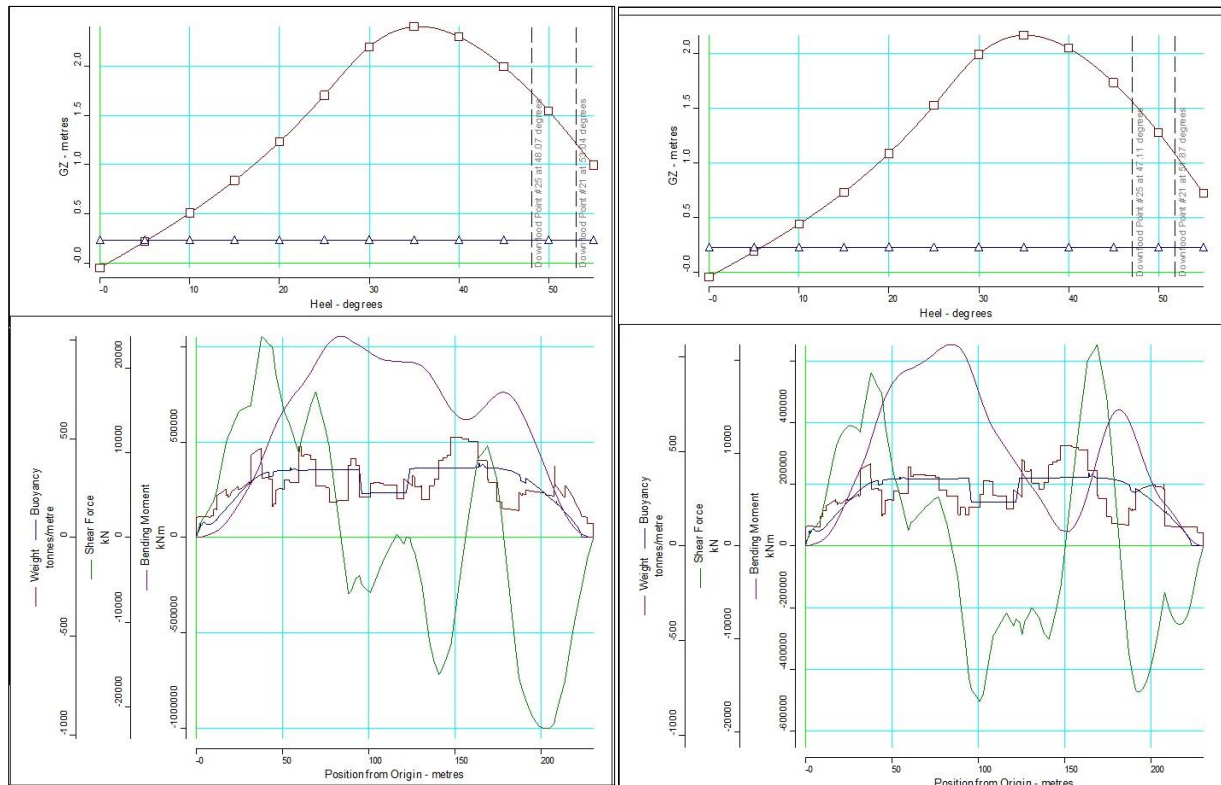
Stand by %100



Start Running casing %10

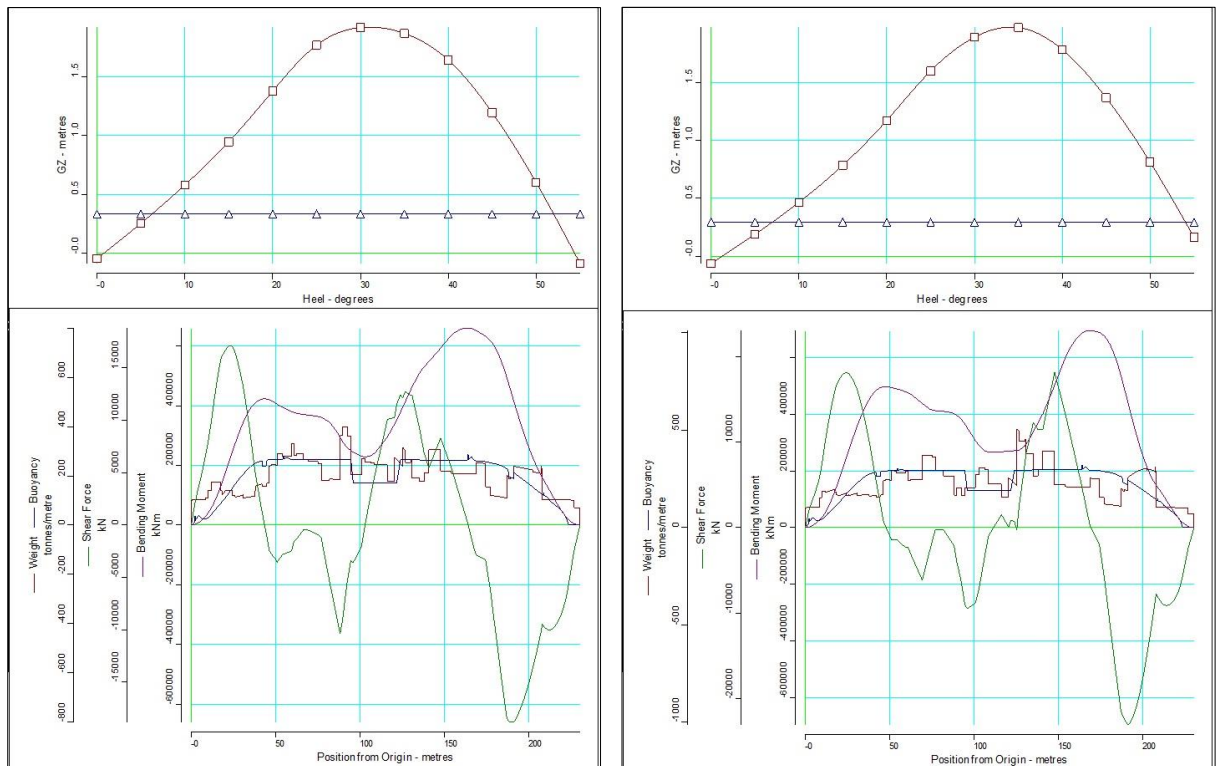
Start Running casing %100

Appendix 13 – GZ Curves and SFBM Diagrams for all Conditions C



Transit with casing %10

Transit with casing %100



Transit without casing %10

Transit without casing %100

Appendix 14 Maximum Shear Force and Bending Moments A

Distance from Origin (m)	Docking	Transit with casing %100				Transit without casing %100				Drilling 10%				Drilling 100%				Start Running Casing 10%				Start Running Casing 100%				Running Casing 10%				Running Casing 100%				Stand by 10%				Stand by 100%				In-field Survival 10%				In-field Survival 100%																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																									
		Shearing Force (kN)		Bending Moment (kNm)		Shearing Force (kN)		Bending Moment (kNm)		Shearing Force (kN)		Bending Moment (kNm)		Shearing Force (kN)		Bending Moment (kNm)		Shearing Force (kN)		Bending Moment (kNm)		Shearing Force (kN)		Bending Moment (kNm)		Shearing Force (kN)		Bending Moment (kNm)		Shearing Force (kN)		Bending Moment (kNm)		Shearing Force (kN)		Bending Moment (kNm)		Shearing Force (kN)		Bending Moment (kNm)																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																															
		(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																																
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Appendix 14 Maximum Shear Force And Bending Moments B

Distance from Origin	Docking		Transit with casing %10		Transit with casing %100		Transit without casing %10		Transit without casing %100		Drilling 10%		Drilling 100%		Start Running Casing 10%		Start Running Casing 100%		Running Casing 10%		Running Casing 100%		Stand by 10%		Stand by 100%		In-field Survival 10%		In-field Survival 100%		
	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)	Shearing Force (kN)	Bending Moment (kNm)			
111	77.7 #111	7408	89224	87357	IE-06	4448	6375872	-1840	359140	-264	412430	101340	383855	-181	61484	7828	5786615	52671	334181	122585	723694	21483	9247052	-13775	704501	7764	303978	83568	107600	41507	-211894
112	78.4 #112	6774	89221	8620	IE-06	4135	648958	-2454	36671	-413	412430	101340	383855	-181	61484	7828	5786615	52671	334181	122585	723694	21483	9247052	-13775	704501	7764	303978	83568	107600	41507	-211894
113	79.1 #113	6066	89118	7504	IE-06	3889	642512	-3029	354280	-539	412430	101340	383855	-181	61484	7828	5786615	52671	334181	122585	723694	21483	9247052	-13775	704501	7764	303978	83568	107600	41507	-211894
114	79.8 #114	5394	89015	6387	IE-06	3567	640974	-3624	341583	-1	412430	101340	383855	-181	61484	7828	5786615	52671	334181	122585	723694	21483	9247052	-13775	704501	7764	303978	83568	107600	41507	-211894
115	80.5 #115	4723	88906	5273	IE-06	3181	644368	-4219	349424	-286	412430	101340	383855	-181	61484	7828	5786615	52671	334181	122585	723694	21483	9247052	-13775	704501	7764	303978	83568	107600	41507	-211894
116	81.2 #116	4052	88798	4157	IE-06	2702	648989	-4814	346995	-234	412430	101340	383855	-181	61484	7828	5786615	52671	334181	122585	723694	21483	9247052	-13775	704501	7764	303978	83568	107600	41507	-211894
117	81.9 #117	3380	88685	3041	IE-06	2273	653261	-5408	344561	-182	412430	101340	383855	-181	61484	7828	5786615	52671	334181	122585	723694	21483	9247052	-13775	704501	7764	303978	83568	107600	41507	-211894
118	82.6 #118	2715	88573	1863	IE-06	1873	657635	-5960	339512	-126	412430	101340	383855	-181	61484	7828	5786615	52671	334181	122585	723694	21483	9247052	-13775	704501	7764	303978	83568	107600	41507	-211894
119	83.3 #119	1767	88468	885	IE-06	1505	652491	-6512	334462	-69	412430	101340	383855	-181	61484	7828	5786615	52671	334181	122585	723694	21483	9247052	-13775	704501	7764	303978	83568	107600	41507	-211894
120	84.0 #120	901	88365	-192	IE-06	347	652556	-7081	329413	-493	412430	101340	383855	-181	61484	7828	5786615	52671	334181	122585	723694	21483	9247052	-13775	704501	7764	303978	83568	107600	41507	-211894
121	84.7 #121	154	88288	-1270	IE-06	-268	656208	-7615	324362	-1097	40704	3318	40834	-641	60218	10652	574917	4537	370403	-429	272788	-6578	73844	6642	54325	3493	15966	6793	21889	18889	
122	85.4 #122	-4521	89127	-2348	IE-06	-903	653585	-8166	319312	-1301	40663	2647	633724	-5874	37260	-1454	6030323	-1141	538716	3704	7846169	-508	927376	-6722	16933	4080	53810	6734	16457	8936	11743
123	86.1 #123	-1658	89075	-3425	IE-06	-1559	6575704	-8718	314260	-1504	40582	2016	61021	-6812	33888	-2567	6018186	634	560716	2954	783295	-5724	927028	-6862	66521	6173	56396	6203	16935	4275	12541
124	86.8 #124	-2287	88988	-4506	IE-06	-2179	658905	-9271	307499	-1713	40470	1346	636021	-6716	2871	-3078	600770	128	539968	8234	786209	-6415	926445	-7002	66073	936	16731	6756	6877	1313	16606
125	87.5 #125	-3076	88965	-5593	IE-06	-2818	648854	-9825	300674	-1919	40306	716	637088	-6716	2871	-3078	600770	128	539968	8234	786209	-6415	926445	-7002	66073	936	16731	6756	6877	1313	16606
126	88.2 #126	-3885	88917	-6677	IE-06	-3458	646904	-10379	293874	-2127	40163	674	638755	-7398	1875	-4071	599237	-861	539693	716	792367	-788	925281	-7262	67509	5615	37396	5699	18180	3495	16104
127	88.9 #127	-4628	88928	-7845	IE-06	-4162	643658	-10912	286967	-2393	39939	635	638668	-8099	12582	-4473	592617	-2025	537665	736	792367	-788	925281	-7262	67509	5615	37396	5699	18180	3495	16104
128	89.6 #128	-5437	88939	-9024	IE-06	-4866	641927	-11467	280555	-2689	39755	305	638516	-8901	718	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
129	90.3 #129	-6246	88943	-10214	IE-06	-5617	641927	-12017	274453	-2977	39376	673	638461	-9607	1369	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
130	91.0 #130	-7055	88948	-11403	IE-06	-6368	641927	-12568	268353	-3266	39016	1045	638311	-10481	2166	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
131	91.7 #131	-7864	88953	-12592	IE-06	-7119	641927	-13119	262263	-3555	38641	1419	638161	-11290	2961	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
132	92.4 #132	-8673	88958	-13781	IE-06	-7870	641927	-13670	256173	-3844	38266	1792	638011	-12099	3756	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
133	93.1 #133	-9482	88963	-14970	IE-06	-8621	641927	-14221	250083	-4711	37121	2908	637561	-14466	6141	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
134	93.8 #134	-10291	88968	-16159	IE-06	-9372	641927	-14772	244000	-4422	37506	3280	637411	-15255	6936	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
135	94.5 #135	-11100	88973	-17348	IE-06	-10123	641927	-15323	237907	-4711	37121	3652	637261	-16044	7731	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
136	95.2 #136	-11909	88978	-18537	IE-06	-10874	641927	-15874	231814	-5000	36736	4024	637111	-16833	8526	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
137	95.9 #137	-12718	88983	-19726	IE-06	-11625	641927	-16425	225721	-5289	36351	4396	636961	-17622	9321	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
138	96.6 #138	-13527	88988	-20915	IE-06	-12376	641927	-16976	219628	-5578	35966	4768	636811	-18411	10116	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
139	97.3 #139	-14336	88993	-22104	IE-06	-13127	641927	-17527	213535	-5867	35581	5140	636661	-19200	10911	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
140	98.0 #140	-15145	88998	-23293	IE-06	-13878	641927	-18078	207442	-6156	35196	5512	636511	-20000	11706	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
141	98.7 #141	-15954	88999	-24482	IE-06	-14629	641927	-18629	201349	-6445	34811	5884	636361	-20799	12501	-4790	589138	-3182	536060	654	793371	-879	925281	-7262	67509	5615	37396	5699	18180	3495	16104
142	99.4 #142	-16763	89000	-25671	IE-06	-15380	641927	-19180	195256	-6734	34426	6256	636211	-21598	13296	-4790	589138	-3182	5360</												

Appendix 14 Maximum Shear Force and Bending Moments C

Distance from Origin	Decking		Transit with casing %10		Transit with casing %100		Transit without casing %10		Transit without casing %100		Drilling 10%		Drilling 100%		Start Running Casing 10%		Start Running Casing 100%		Running Casing 10%		Running Casing 100%		Stand by 10%		Stand by 100%		In-field Survival 10%		In-field Survival 100%			
	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment	Shearing Force	Bending Moment		
	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)	(kN)	(kNm)		
223	155.4	+2222	-157.7	31110	-1008.6	619262	8421.7	63174.5	4745.1	642166.1	12138.7	618823	-2497.8	1408888.2	18203.1	1E+05	30723	831992.4	16015.6	716548	-312.6	923539.7	9985.8	1602371	-719.9	1320983	15164.7	80102	31773	1320983	20890.9	240577
224	156.1	+2233	-107.6	511654	-14.7	618731	9252.2	69185.1	4564.3	645034	11535.7	624056	-2091.1	1408846.4	18857.5	1E+05	2496.9	833629.9	16774.4	727910.5	109.4	923534.7	8718.9	1608323	-349.7	1321646	15400.8	790571	4183.3	504980	21538.7	255158
225	156.8	+2234	552.6	511765	978.3	618735	10627.9	75790.1	3983.5	647858.6	10932.7	624167	-3739.0	1404761.7	18203.1	1E+05	30720	835503.5	16932.5	739357.2	732	923846	7452.1	1614125	-719.9	1321077	15164.7	80102	31773	1320983	20890.9	240577
226	157.5	+2235	906.7	512371	1970	619575	11729.7	83324.4	3602.3	650008	10328.2	639114	-3819.7	1402061.2	19553.6	2E+05	3545.6	837971.7	17089.2	750952.1	1355.9	924752.8	6186.3	1617701	-384.6	1322056	15672.2	811894	6202.4	512347	22831.3	285631
227	158.2	+2236	1260.7	513110	2937.4	620199	12807.2	91238.6	3221.1	652159.9	9723.7	645811	-4259.2	1390195.3	19902.1	2E+05	4070.7	840730.8	17448.8	762646	1880.5	926081.3	4920.4	1621276	-749.2	1323167	16106.7	827253	7211.5	517156	23475.9	301588
228	158.9	+2237	1614.7	513840	3845.1	621312	13825.3	100481.6	2839.8	654510.5	9119.2	652208	-4698.8	1386239.4	20250.7	2E+05	4596.1	843664.9	17399.9	774930.8	2605.4	927578	3654.6	1624851	-1113.8	1323167	16341	833999	8220.7	522302	24120	317760
229	159.6	+2238	1968.1	514891	4751.8	625862	14842.4	110235.5	2458.4	655867.4	8513.6	658203	-5137.8	1381087.1	20599.6	2E+05	5123.8	847006.3	17533.6	786276.9	3231.3	929674.7	2389.6	1626460	-1476.5	1321181	16574.5	845260	9210.3	528383	24762.6	334126
230	160.3	+2239	2320.8	516416	5658.8	629448	15851.1	120730.4	2076.7	657104.6	7907.3	661013	-5575.9	1382942.4	20948.8	2E+05	5640.1	850855.6	17705.6	798305.5	3858.2	932365.5	1125.2	1627011	-1836	1320188	16801.7	856458	10240.1	533421	25403.5	351898
231	161.0	+230	2673.5	519062	7464.7	637847	17807.5	143115.3	1313.4	665998.4	6694.6	673719	-6451.9	1381375.9	21647.6	2E+05	6174.9	853119.3	17871.7	810389	4485.5	935285.7	-139.2	1627501	-2195.3	1319194	17034.9	888037	11270.1	543228	26043.9	369504
232	161.7	+231	3025.9	519106	9062.1	641378	18312.2	148511.1	1000.3	667500.6	5947.7	676530	-6888.7	1376557.7	21997.4	3E+05	6697.4	856119.3	17871.7	810389	4485.5	935285.7	-139.2	1627501	-2195.3	1319194	17034.9	888037	11270.1	543228	26043.9	369504
233	162.4	+232	3377.5	51884	8366.7	643165	18900.4	155862.7	931.8	669881.1	6087.1	677765	-6888.7	1376557.7	21997.4	3E+05	6697.4	856119.3	17871.7	810389	4485.5	935285.7	-139.2	1627501	-2195.3	1319194	17034.9	888037	11270.1	543228	26043.9	369504
234	163.1	+233	3729	524073	9267.4	649023	19912.5	169016.4	550	660357.7	5479.7	681805	-7326.4	1371934.6	22347.4	3E+05	7757.1	869742.4	18304.3	847280.3	6371.9	946770	-3929.2	1626314	-3264.8	1314639	17732	903999	14281.7	569428	27957.8	425081
235	163.8	+234	4112.3	526347	9480.8	65332	20174.6	17666.3	1561	660359.9	4839.3	685152	-7911.8	1365107.7	21948.3	3E+05	7585.3	875152.8	17702.2	859525.4	6253.4	951265.6	-5203.6	1630223	-3766.4	1315218	17212.7	915966	14544	57911	27844.6	444271
236	164.5	+235	4504.1	528544	9664.6	661746	20369.4	186457.9	-304.2	660125.9	4172.4	688024	-8588.3	1361034.8	21484.2	3E+05	7250	880236.5	17023.7	871389	6069.4	956313.2	-4543.7	1615902	-4538.3	1310013	16626.4	92753	14402	58946	27663.9	463242
237	165.2	+236	4904.1	530402	9855.5	668301	20590.4	210402.1	-736.7	659551	3512.7	690437	-9238.6	1354404.2	21046.9	3E+05	6901	885343.5	16389	882465	5913.2	959897	-7853.5	1610669	-4920.9	1307109	16667.7	938709	14963.8	600135	27509.3	482305
238	165.9	+237	5304.1	531984	10010.3	674976	20775.2	223448	-1204.3	658600	2817.7	692376	-9917.9	1348209.1	21078.4	3E+05	6697.4	890158.8	17094.9	893491	6201.8	9604512	-8201.8	1604512	-5516.7	1303790	16671.2	949458	15123.6	610674	27718.4	518939
239	166.6	+238	5698.6	53311	10162.3	681746	20957.3	238604.7	-1673.8	657450	2120.3	693814	-10654	1341234.5	21011.2	3E+05	6402.3	894782.2	15028.8	903896.9	5531.2	9608404	-10550	1597392	-6112.5	1300808	14873.3	959822	15339.8	621327	27124.7	518939
240	167.3	+239	6094.1	534367	10352.6	688647	21177.6	250308.3	-2104.2	655890.5	1461.6	694783	-11250	1333691.2	19967.4E+05	6047.1	899197.3	14382.2	913802.1	5379.6	971929.5	-11858	1589346	-6669.4	1295957	14313.9	969748	15566.6	632141	26968.9	538411	
241	168.0	+240	6498.6	535821	10568.7	694406	21431.4	267559	-2496	640637.7	841.1	695303	-11857	1325701.1	19267.6	4E+05	5926.7	903500.2	13771.1	923283.6	5263.5	975719.8	-13127	1580393	-7185.9	1291739	20805.5	970309	15828.2	64816	27302	559845
242	168.7	+241	6904.1	537399	10814.9	702802	21699.3	282230.3	-2871.2	651890.8	236.8	695413	-12448	1317307.6	18583.4	4E+05	5723.1	9075662	11374	932375.8	5163	979440.6	-14378	1570590	-7684.7	1286077	13277.5	983335	16010.2	654273	26738.3	575302
243	169.4	+242	7316.1	539141	11043.8	710042	21968.4	298255.5	-3928.7	646657.6	-49.1	694784	-13056	1304845.7	17784.2	4E+05	4801.9	911280.1	11860.3	940722.5	4165.6	982773.3	-15312	1560003	-8516.3	1281243	11888	907767	15494.6	66523	27371	593215
244	170.1	+243	7781.3	540858	11300.9	718320	22359.2	310112.4	-2982.2	647334.6	-335	694455	-13719.2	1303726.3	16586.4E+05	4378.3	9191328	10444.1	946651.2	3070.5	9841635	-16345	1547588	-9432.1	13070.5	129533	100291.1	16475	97075	26021.1	639602	
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246	171.5	+245	8801.3	544738	11883.9	734693	23093.2	336988.3	-4068.2	642688.7	-906.8	692807	-15178	1278298.5	14589.7	4E+05	4761.9	9150163	7611.7	958408.5	3777.4	982053.6	-18411	1522757	-11284	1269891	7335.8	1014598	13336	696977	22501.4	641455
247	172.2	+246	9384.2	546731	12183.9	743747	23497.3	350973.2	-4548.7	640368	-1192.7	692268	-15885	1268253.6	12991.2	4E+05	4783.9	91851	6195.5	964287.1	-214.8	983451.1	-19444	1510422	-12180	1248994	7285	1020442	12665.6	704627	23291.5	657502
248	172.9	+247	9981.6	548808	12497.7	753508	23893.4	365723.3	-5001.3	637912	-1477	690054	-16587	1256566.8	11098.4E+05	-2843	918253.5	4780	969628.2	-1303	987463	-20473	1495572	-13095	1240086	4316.6	1023835	11891.8	712740	20182.3	67223	
249	173.6	+248	10581.6	550821	12765.8	763506	24241.4	381231.1	-5454.6	635436	-1761.1	690348	-17284	1247280.2	10766.2	4E+05	-2366	9191281	3364.7	9690175	-2390.1	1082719	-21500	1488400	-14011	1232719	20805.5	1026460	11178	720768	19073.1	681447
250	174.3	+249	11184.6	552844	13044.1	773493	24649.9	395960.9	-5905.6	632960	-2045.2	688094	-17987	1232860.2	9413.8	5E+05	-2307.5	919264.1	3403.9	971106.7	-3477.2	984109.1	-22528	1465228	-13004	1026460	10464.1	728796	17964	697972	22501.4	641455
251	175.0	+250	11792.6	554861	13326.7	783741	25059.2	409814.8	-6408.9	630468	-2382.1	686050	-18686	1220365.8	8222.8	5E+05	-2317.9	916380.3	3343	972594.5	-4602.6	985145.7	-23555	1449555	-15836	1021624	9545	73563	16301.5	706449		
252	175.7	+251	12397.1	556878	13613.2	794031	25473.3	421405.7	-6893.2	6280834	-3914.3	683242	-19378	1207097.1	7039.3	5E+05	-2320.5	92860.2	8763	9718423	-3639.6	978154	-24574	1432390	-17619	1027490	13004	1026460	7741.3	741356	14443.5	719625
253	176.4	+252	12997.1	558903	13903.6																											